

(12) **United States Patent**  
**Arnold et al.**

(10) **Patent No.:** **US 9,468,795 B2**  
(45) **Date of Patent:** **Oct. 18, 2016**

(54) **SELECTABLE STRIDE ELLIPTICAL**

21/0051 (2013.01); A63B 21/0053 (2013.01);  
A63B 21/0088 (2013.01); A63B 21/015  
(2013.01); A63B 21/225 (2013.01); A63B  
2022/0028 (2013.01); A63B 2022/0682  
(2013.01); A63B 2071/068 (2013.01)

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(58) **Field of Classification Search**  
CPC ..... A63B 22/0015; A63B 22/0664; A63B  
22/001; A63B 24/00; A63B 24/0062  
See application file for complete search history.

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(\*) Notice: Subject to any disclaimer, the term of this  
patent is extended or adjusted under 35  
U.S.C. 154(b) by 0 days.

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(21) Appl. No.: **15/013,168**

(22) Filed: **Feb. 2, 2016**

(65) **Prior Publication Data**

US 2016/0151659 A1 Jun. 2, 2016

**Related U.S. Application Data**

(63) Continuation-in-part of application No. 14/686,390,  
filed on Apr. 14, 2015, now Pat. No. 9,272,181.

(60) Provisional application No. 62/080,299, filed on Nov.  
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(51) **Int. Cl.**

<b>A63B 24/00</b>	(2006.01)
<b>A63B 22/00</b>	(2006.01)
<b>A63B 22/06</b>	(2006.01)
<b>A63B 21/005</b>	(2006.01)
<b>A63B 21/008</b>	(2006.01)
<b>A63B 21/015</b>	(2006.01)
<b>A63B 21/22</b>	(2006.01)
<b>A63B 71/06</b>	(2006.01)

(52) **U.S. Cl.**

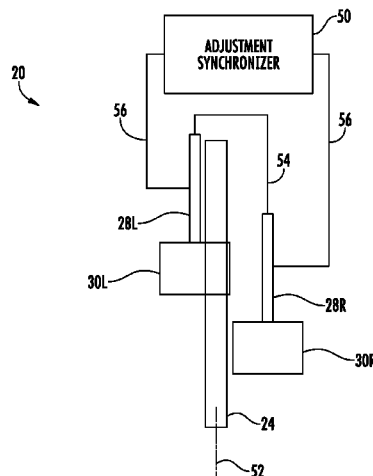
CPC ..... **A63B 22/0015** (2013.01); **A63B 22/001**  
(2013.01); **A63B 22/0664** (2013.01); **A63B**

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**ABSTRACT**

An exercise apparatus comprises a frame, a left foot link supporting a left foot pad, a right foot link supporting a right foot pad in a left foot link supporting a left foot pad. The right foot pad is linked to the left foot pad so as to synchronously move out of phase with the left foot pad through an elliptical path having an adjustable step height and an adjustable stride length. An adjustment synchronizer is operably coupled to the left foot link and the right foot link to synchronously adjust both the step height and the stride length of the elliptical path.

**20 Claims, 39 Drawing Sheets**



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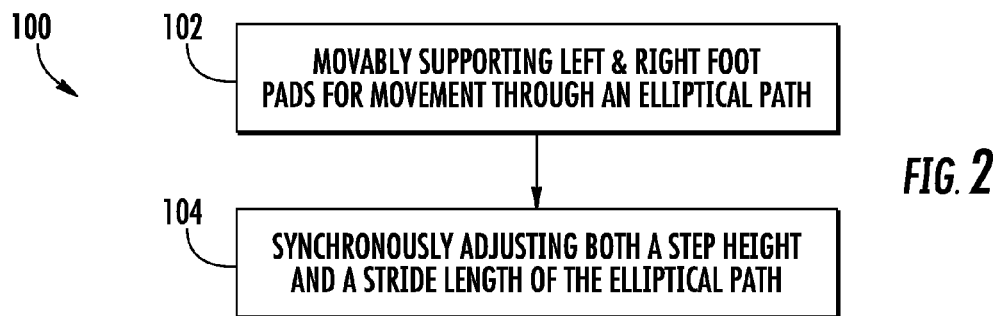
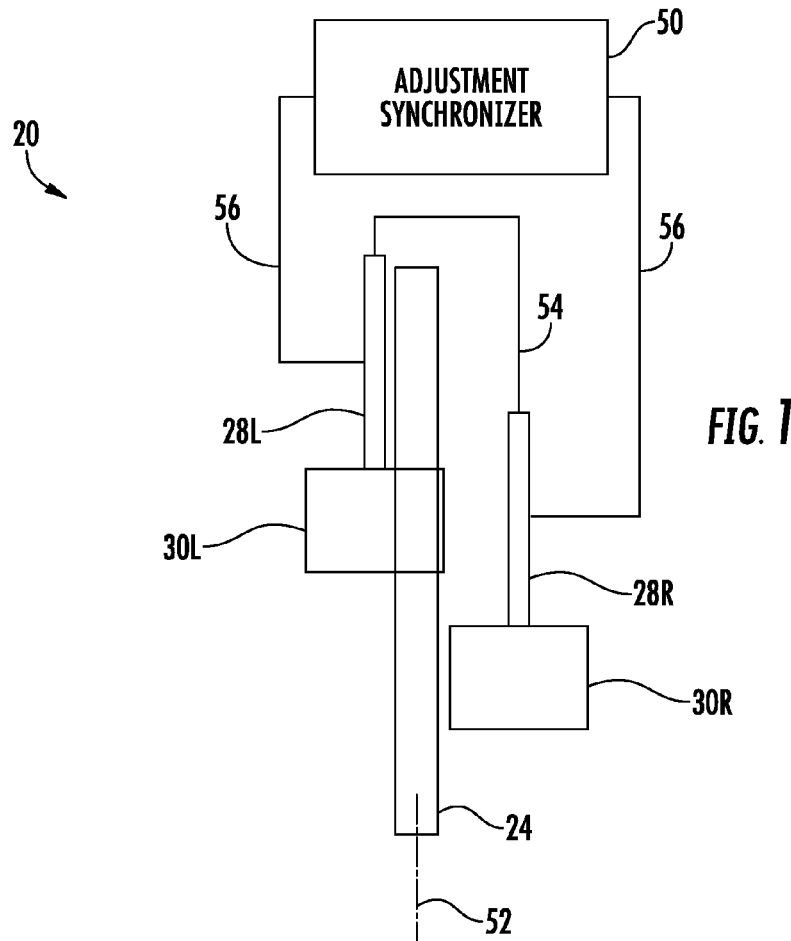
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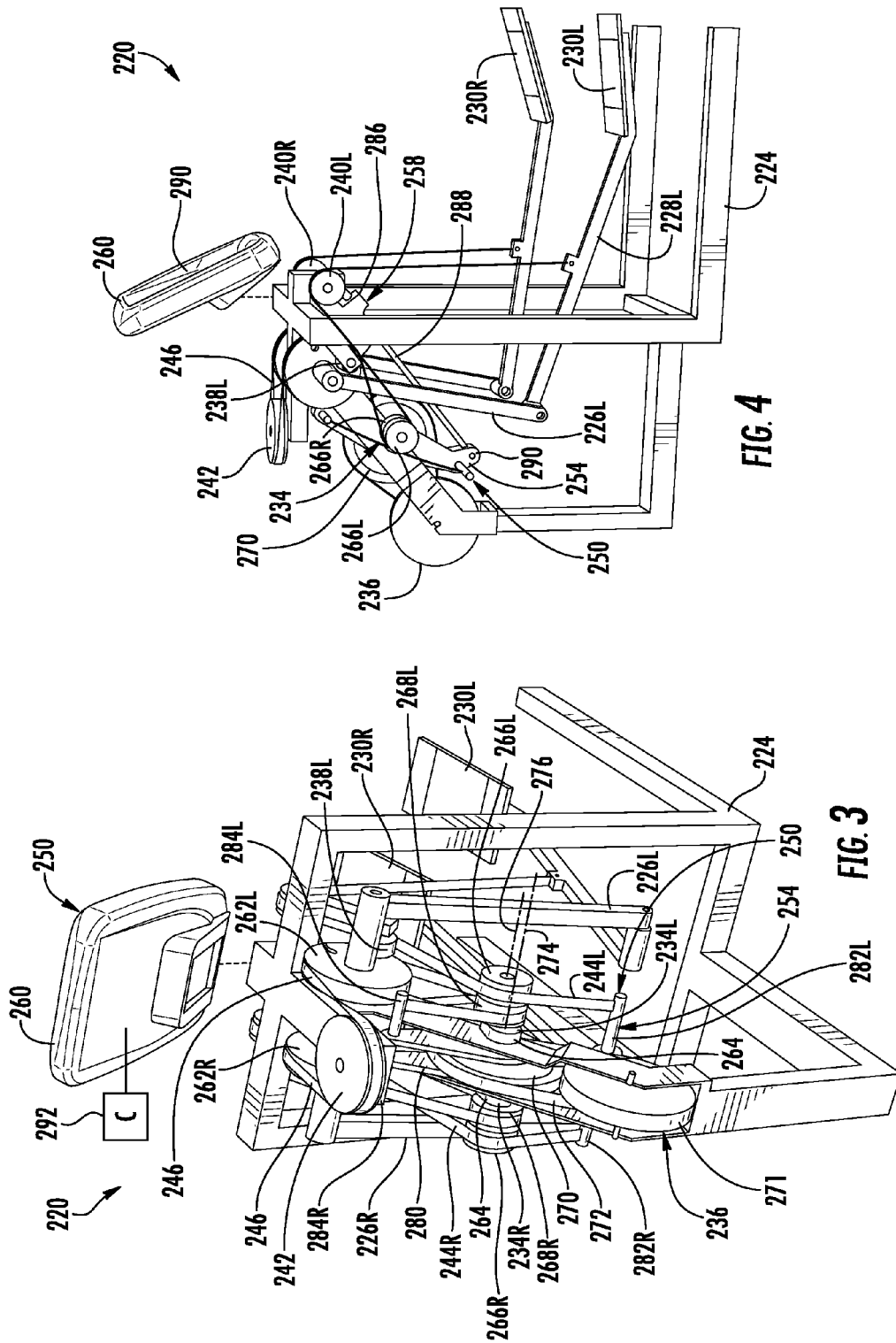
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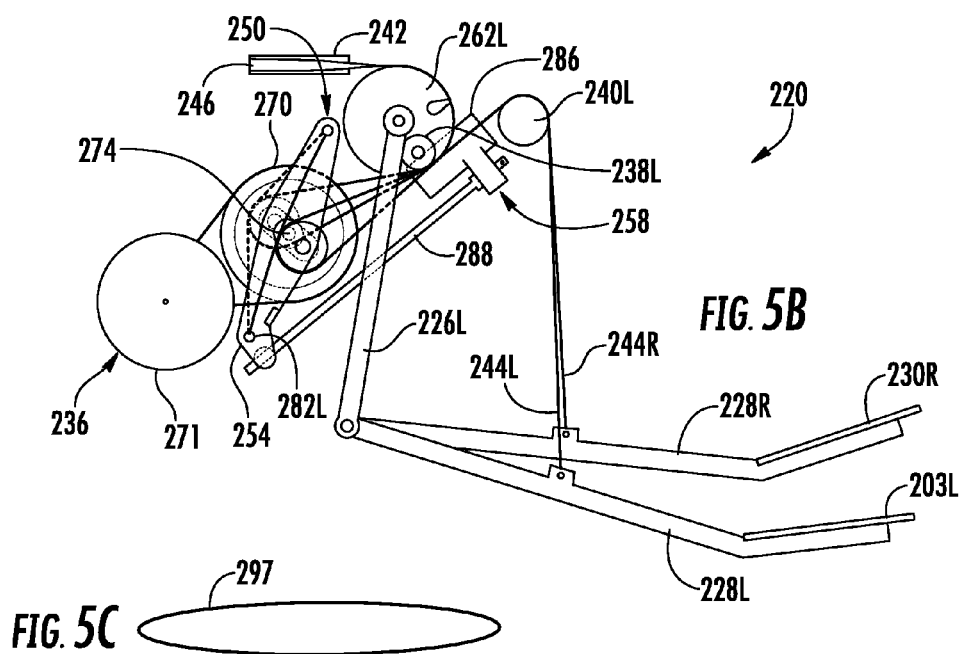
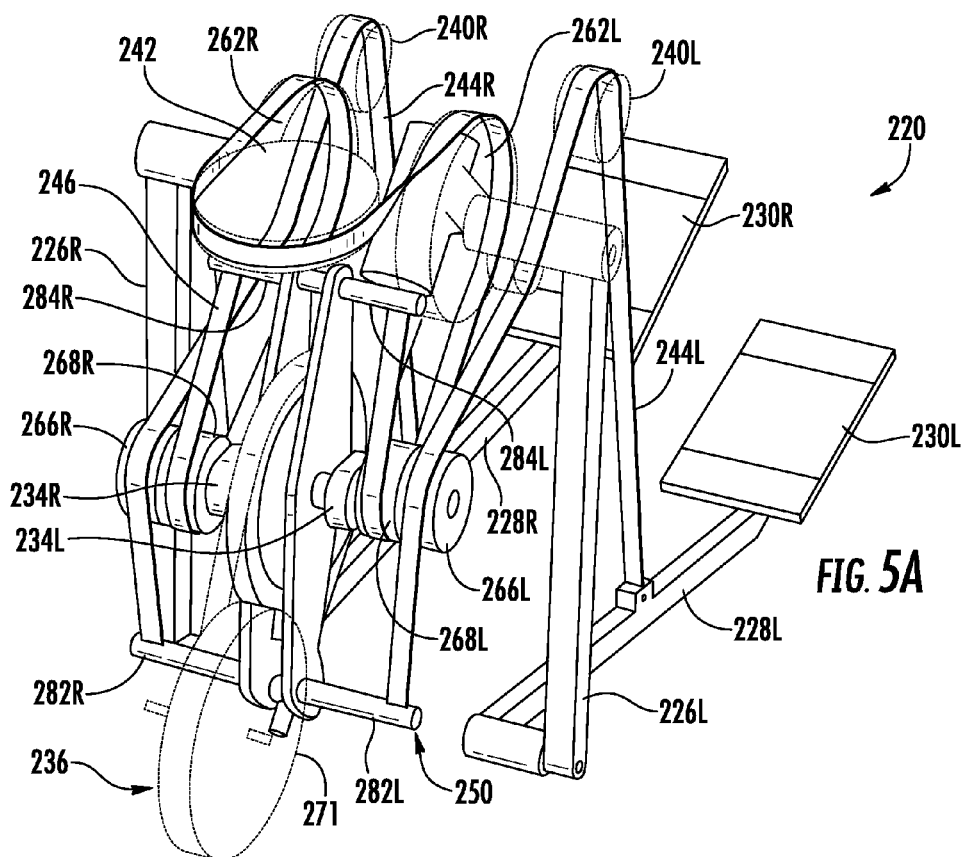
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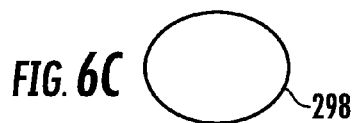
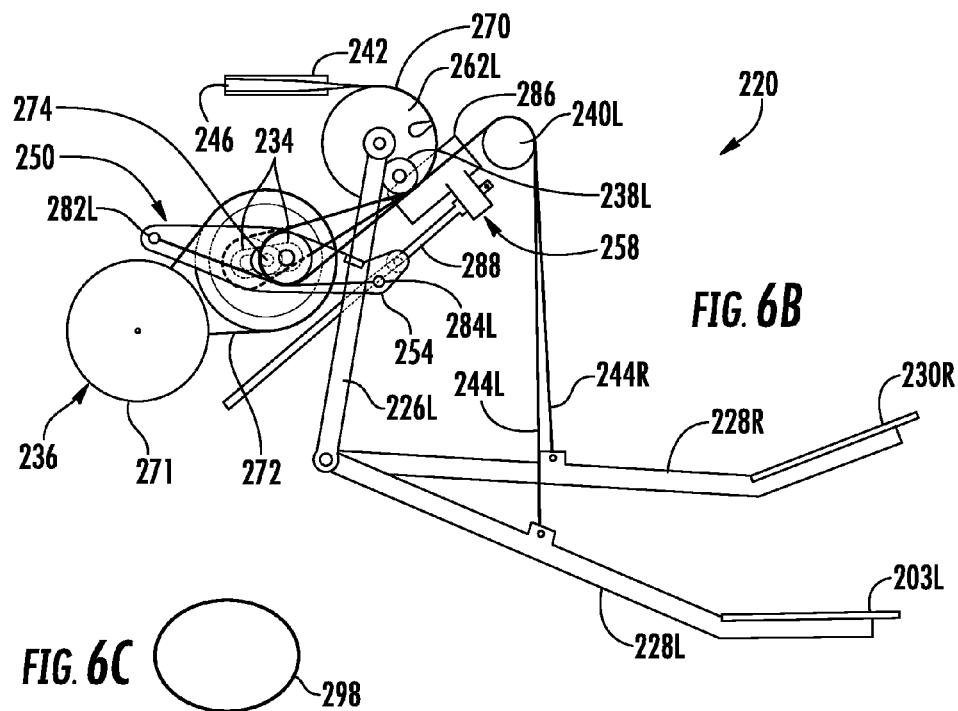
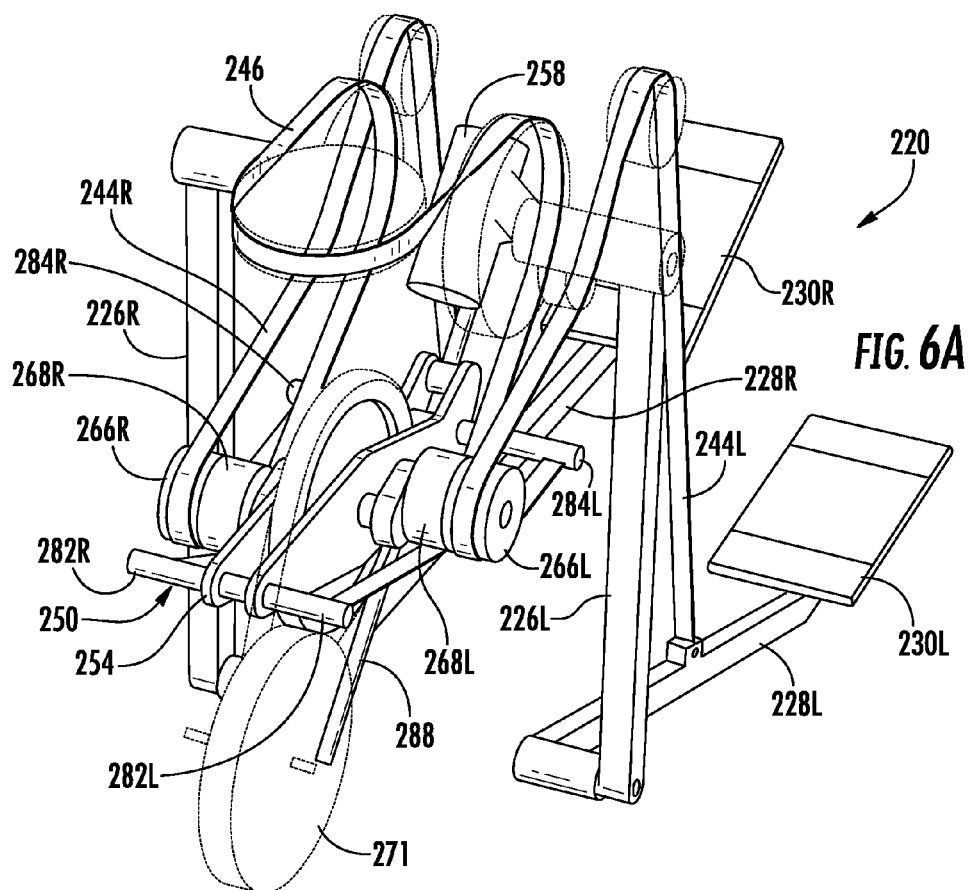
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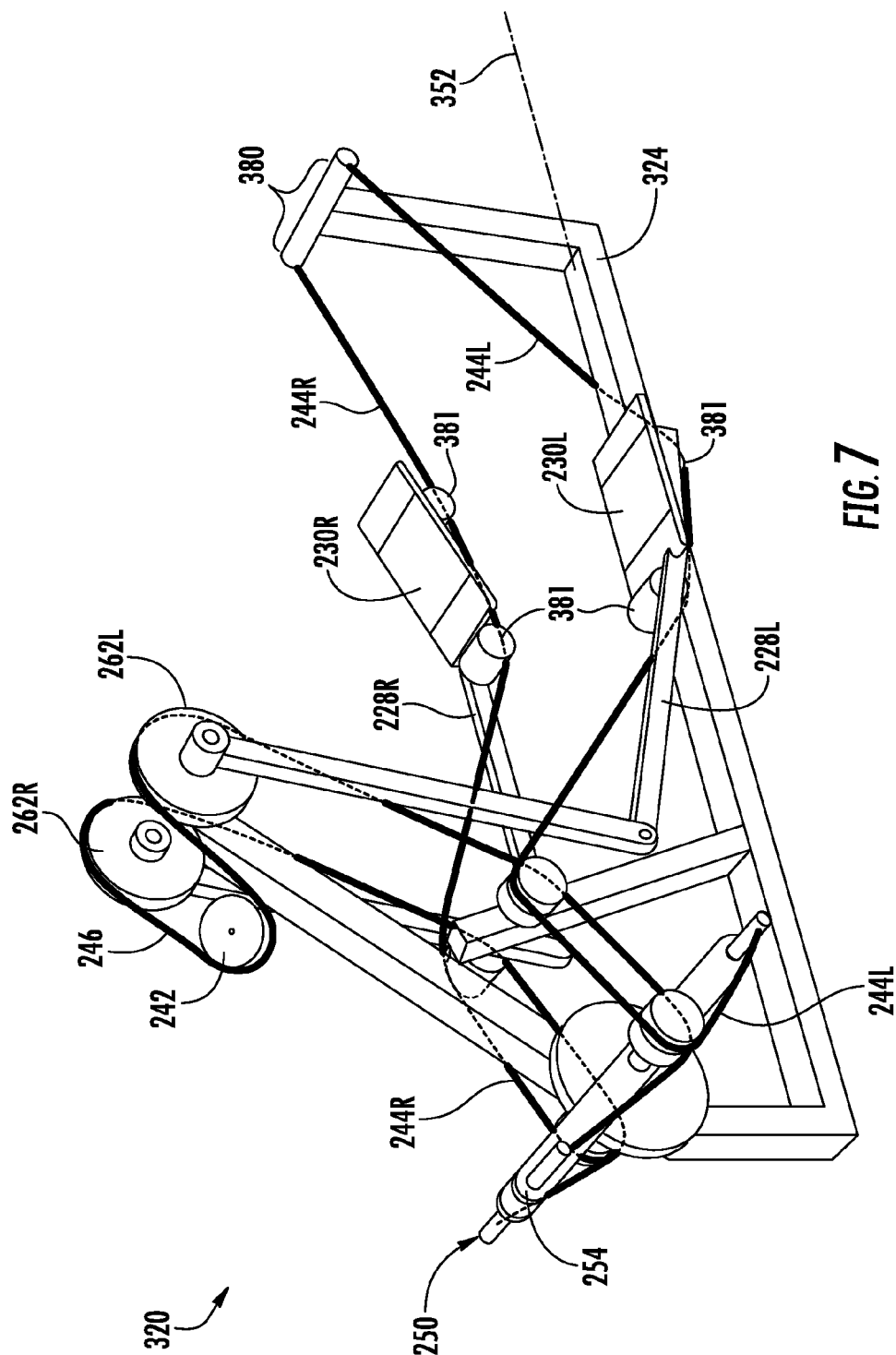
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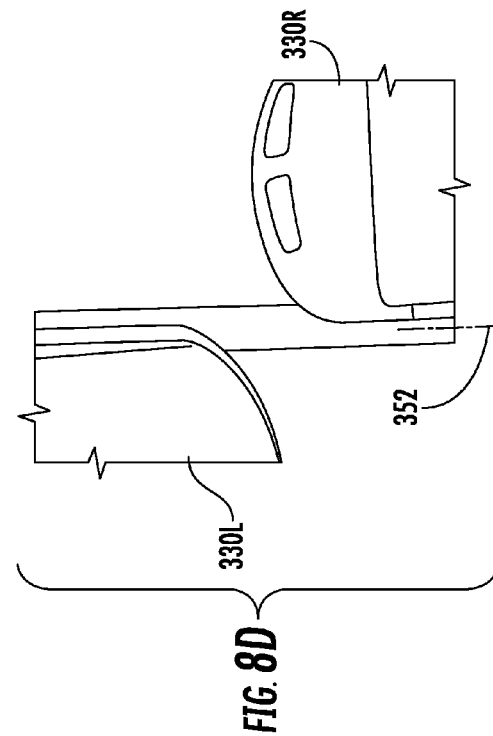
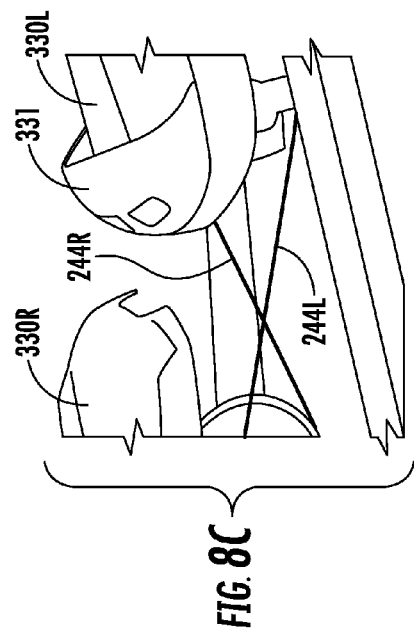
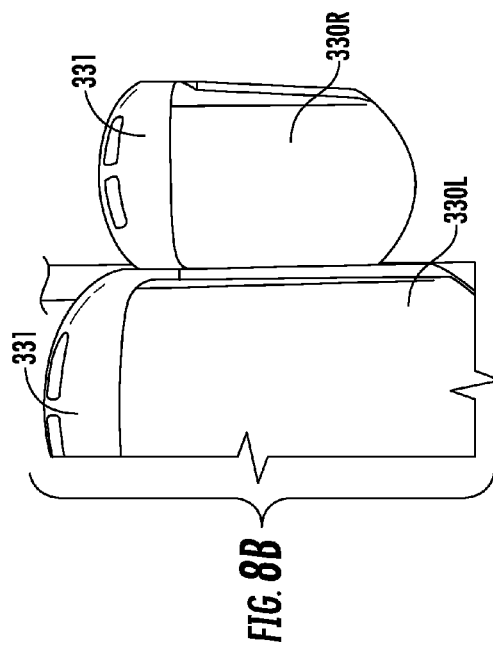
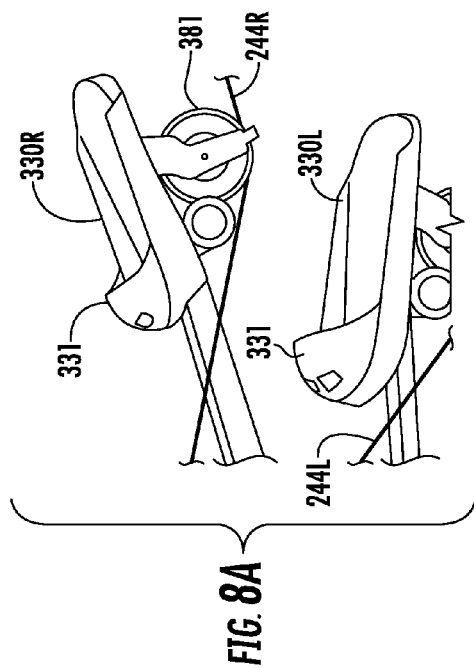














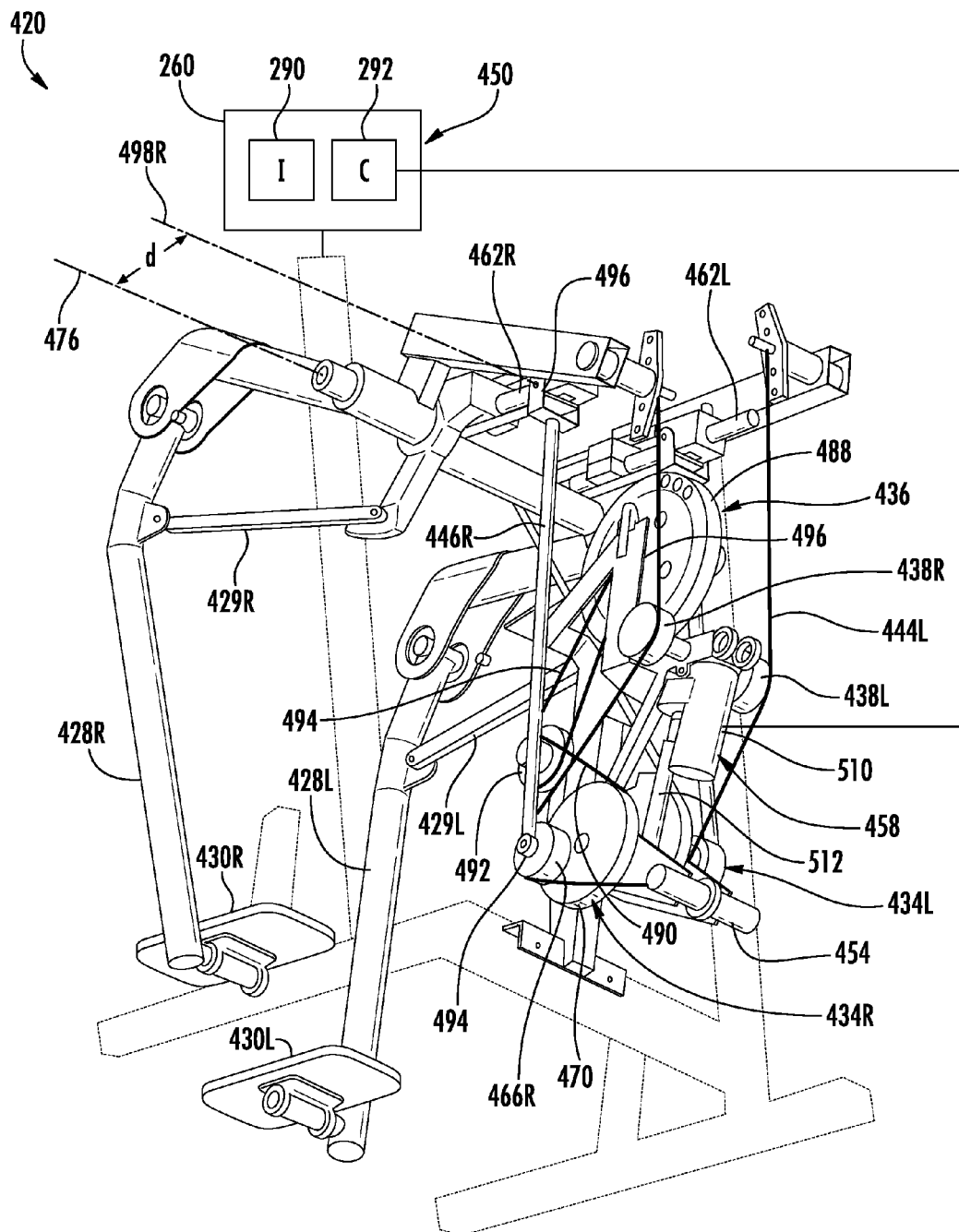


FIG. 9

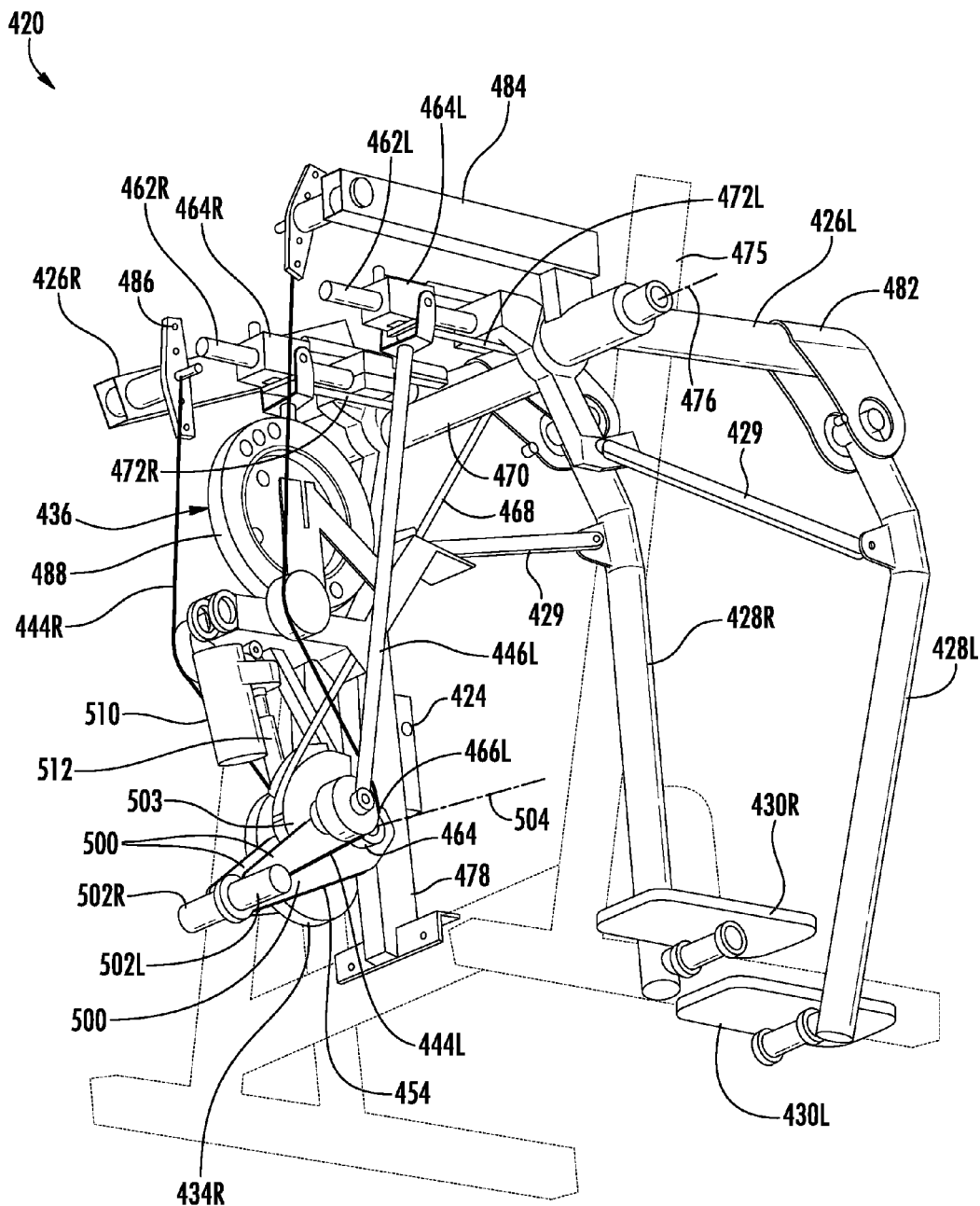
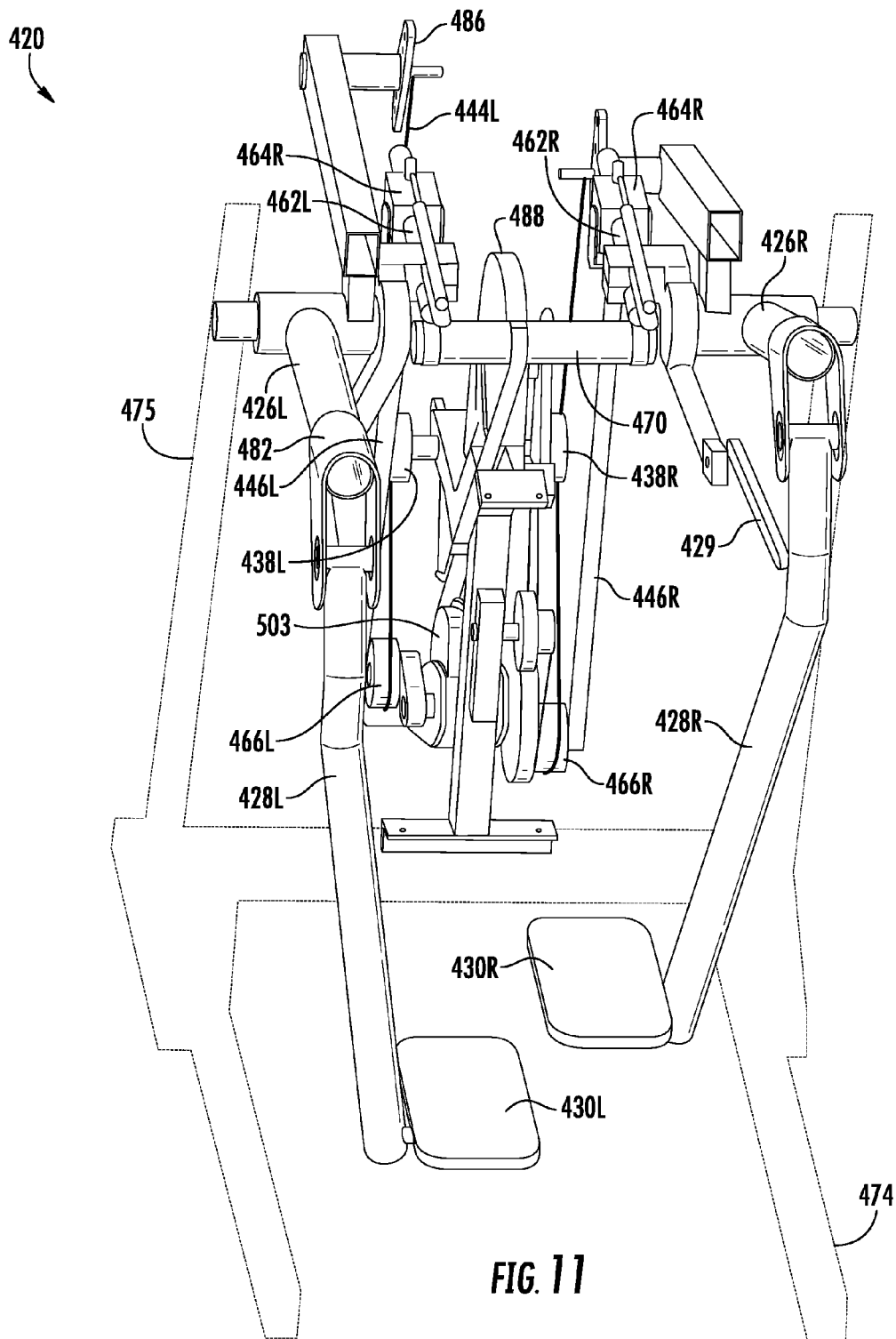


FIG. 10



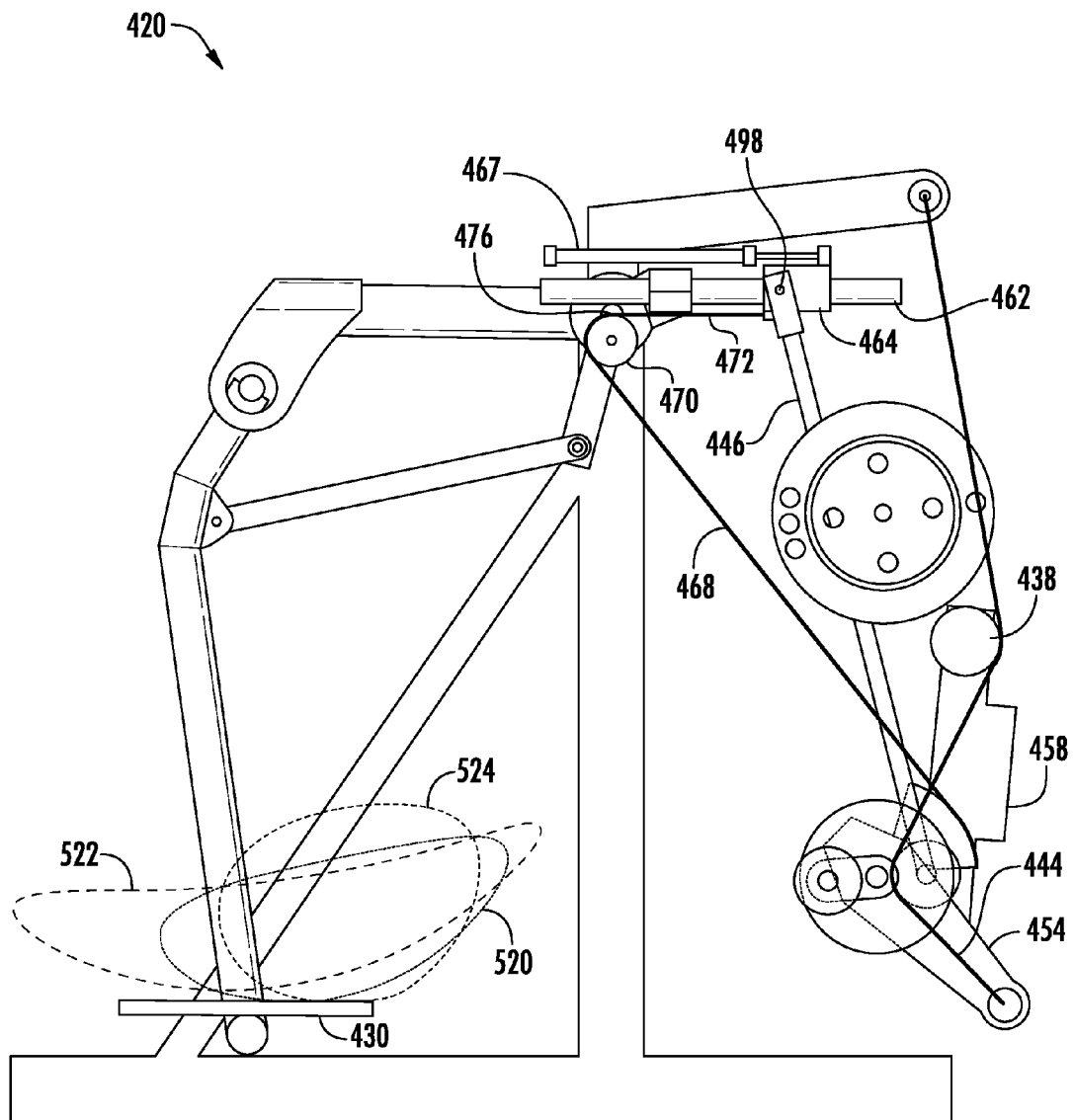


FIG. 12

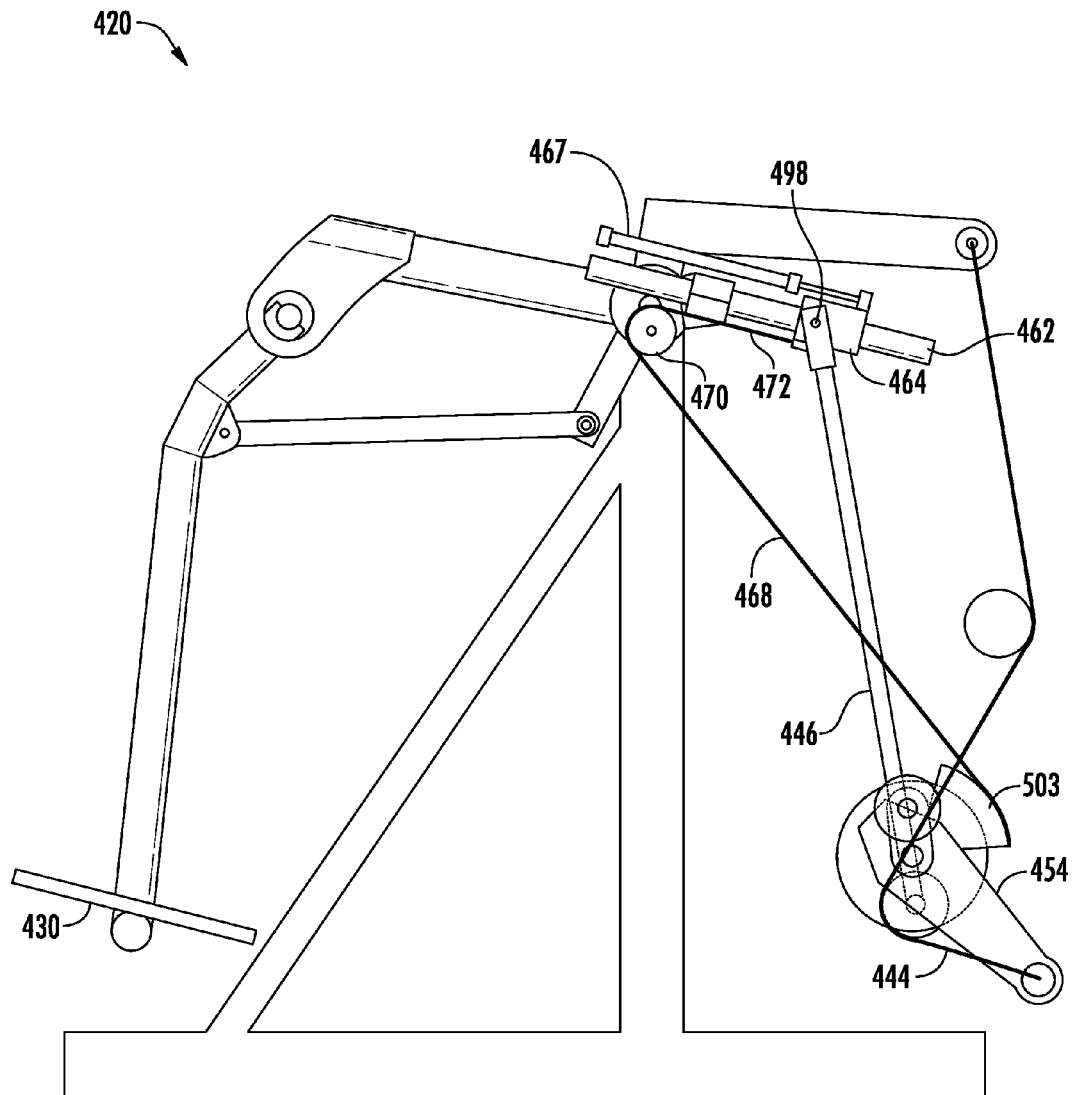


FIG. 13

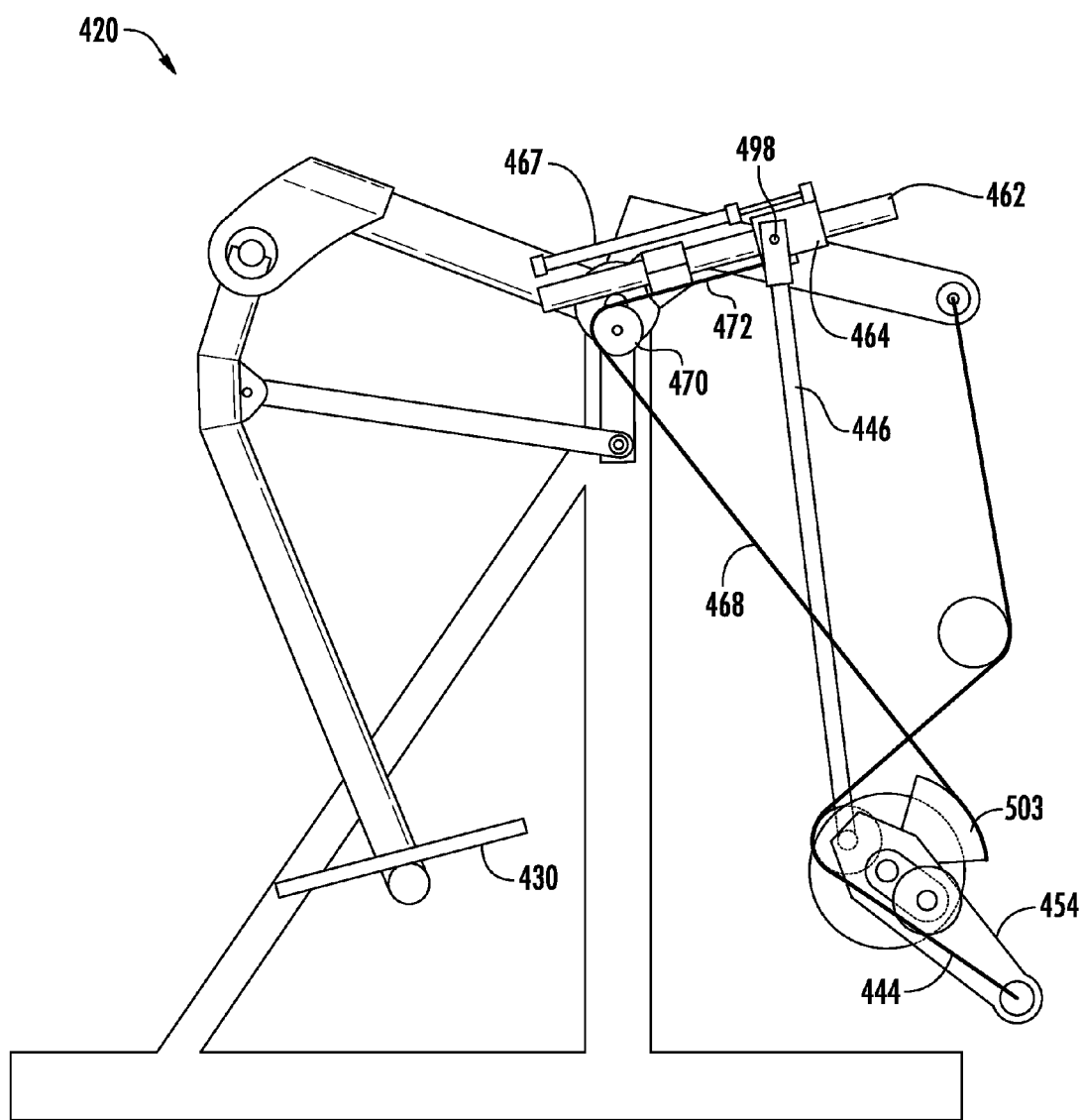


FIG. 14

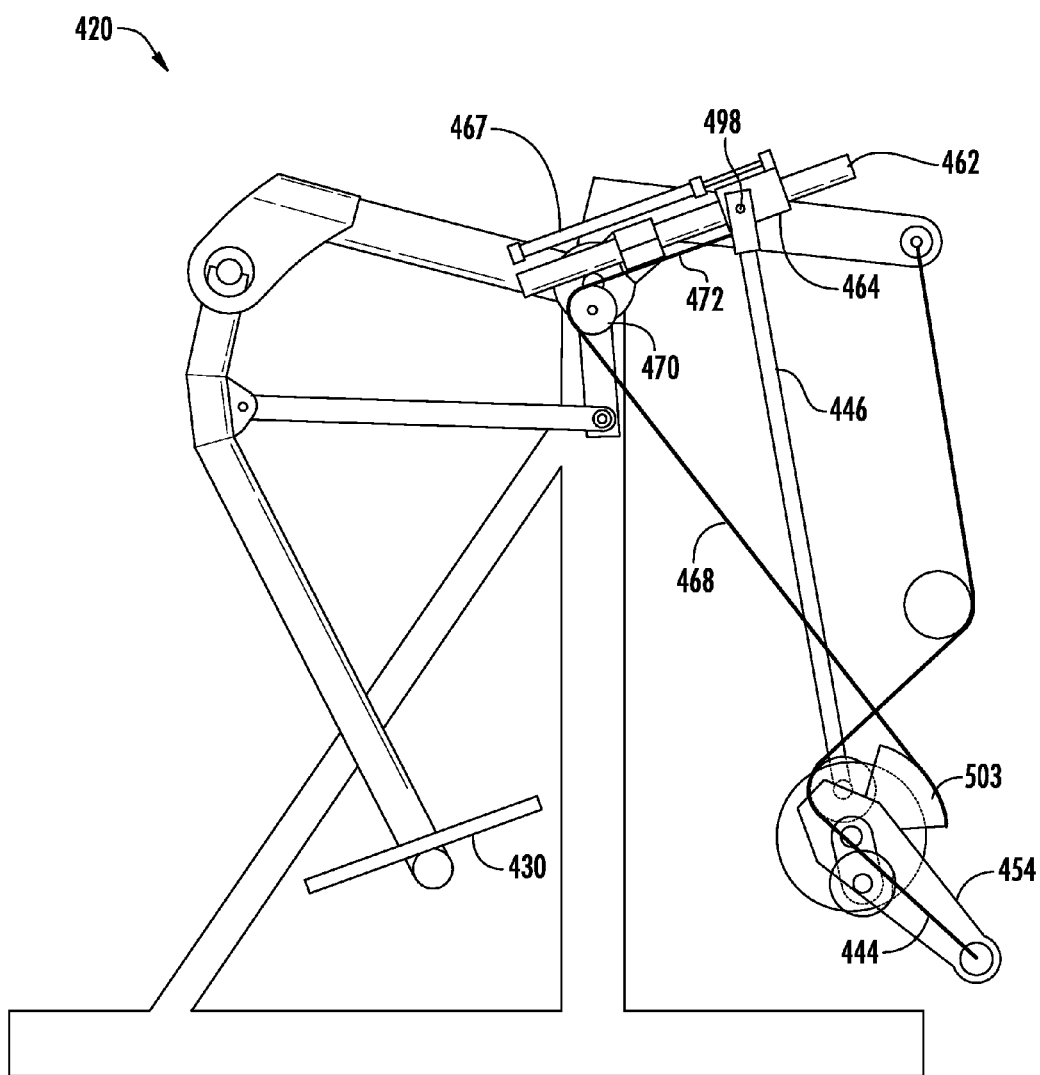


FIG. 15

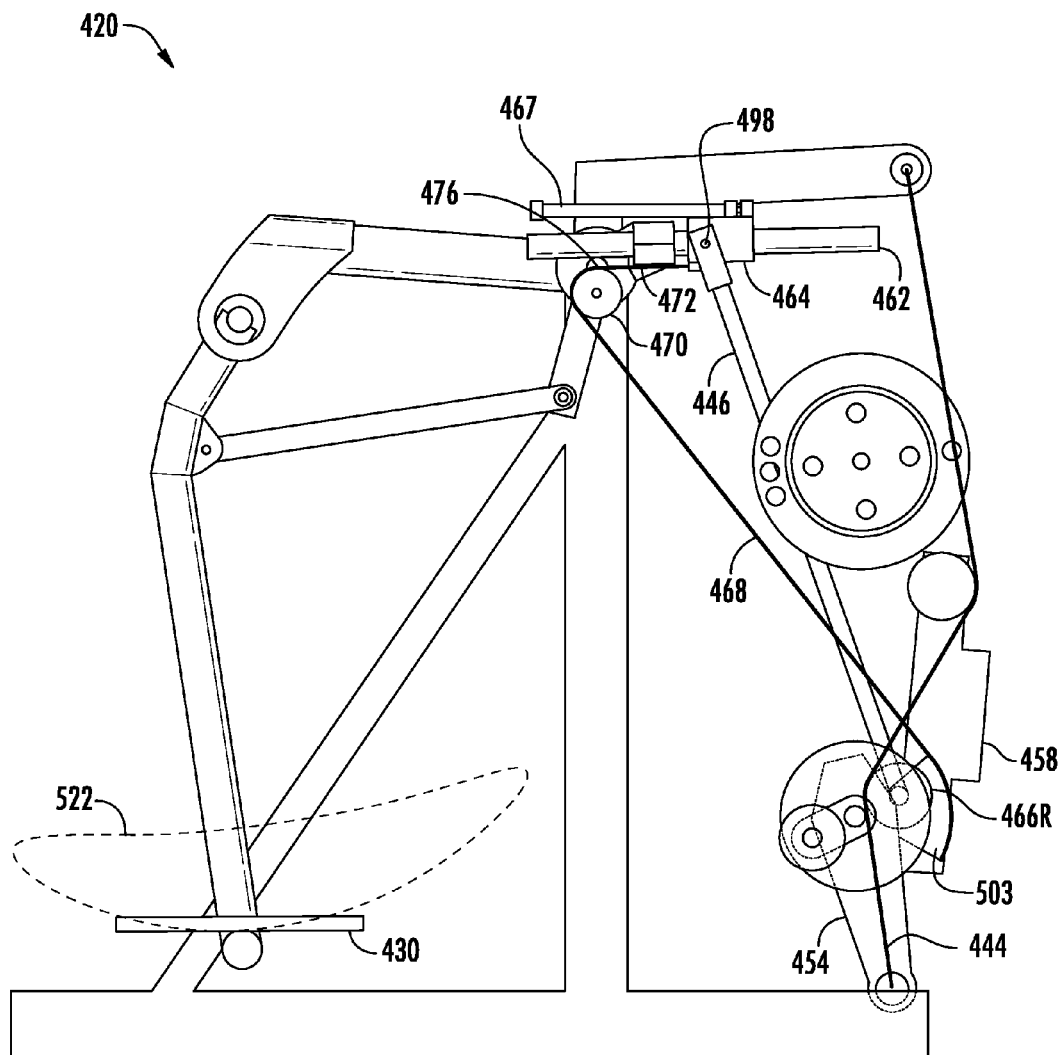


FIG. 16



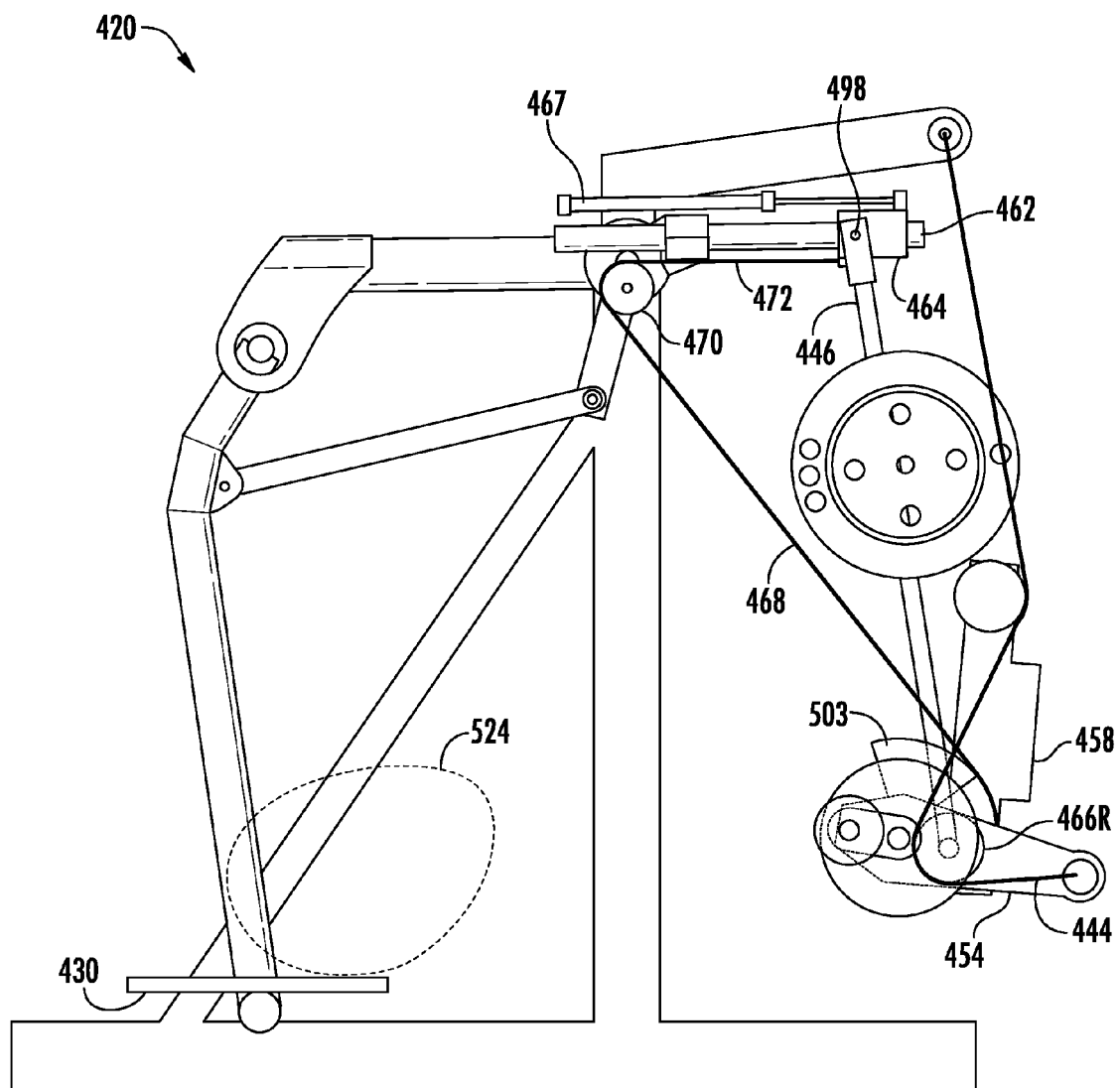
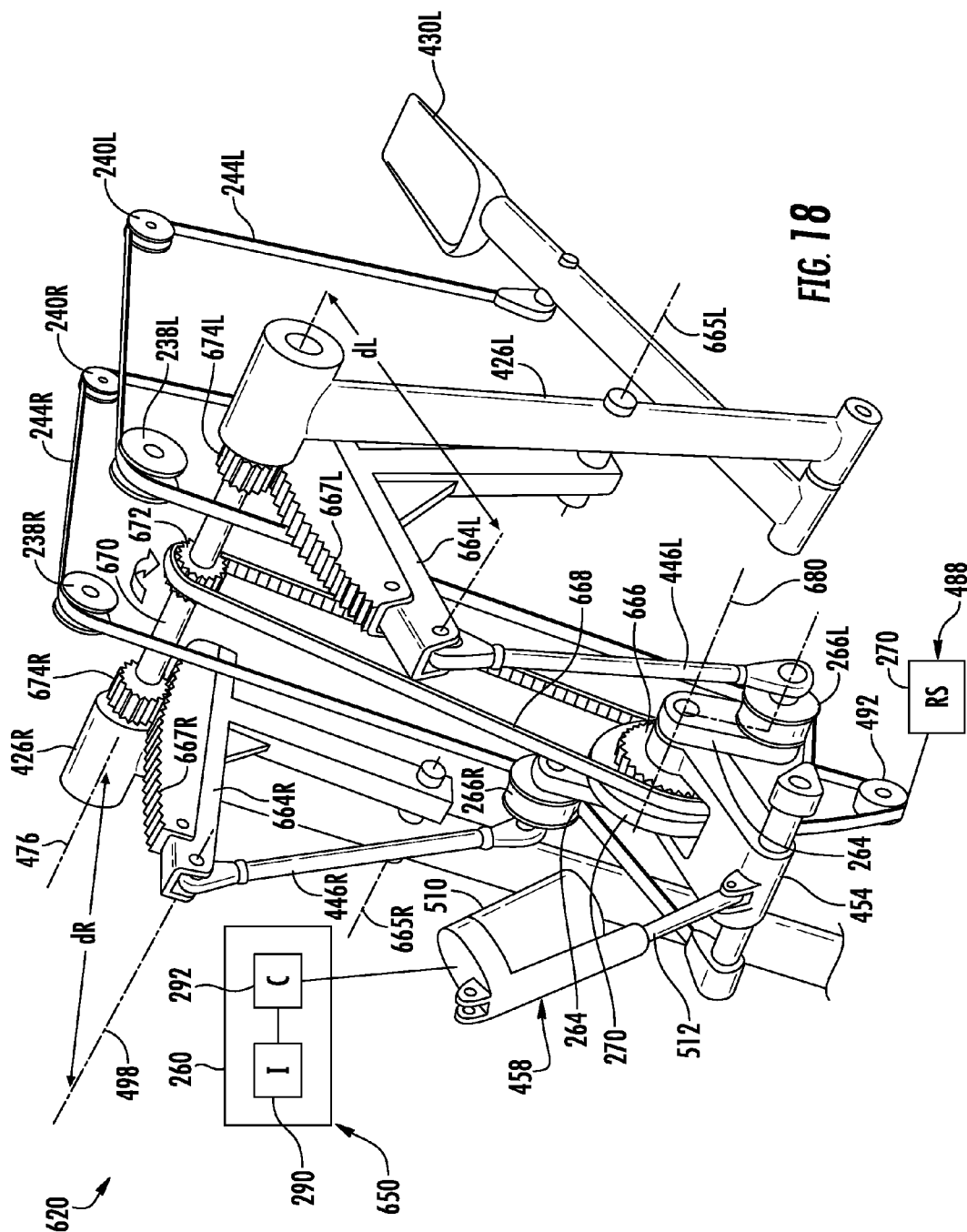
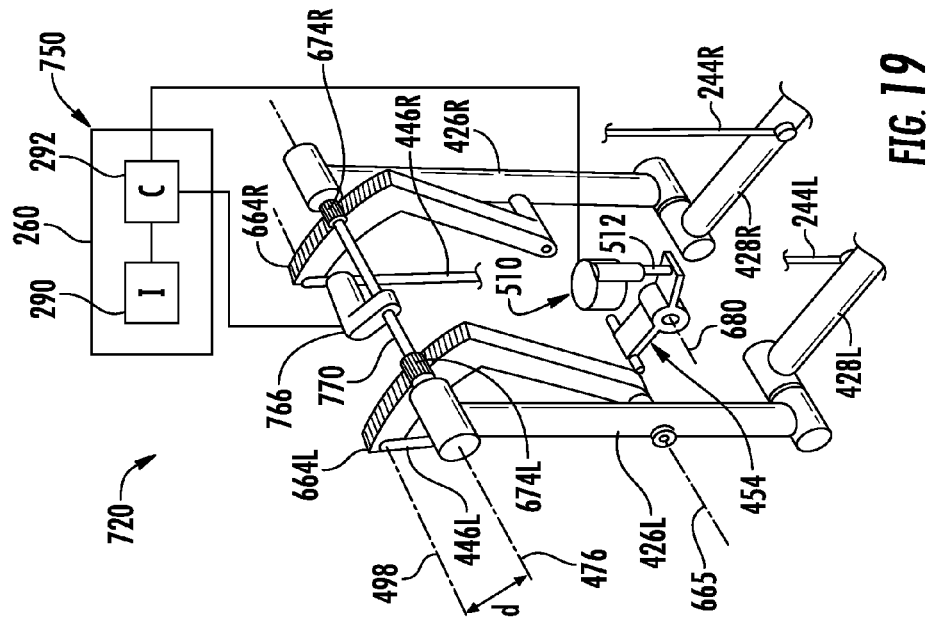
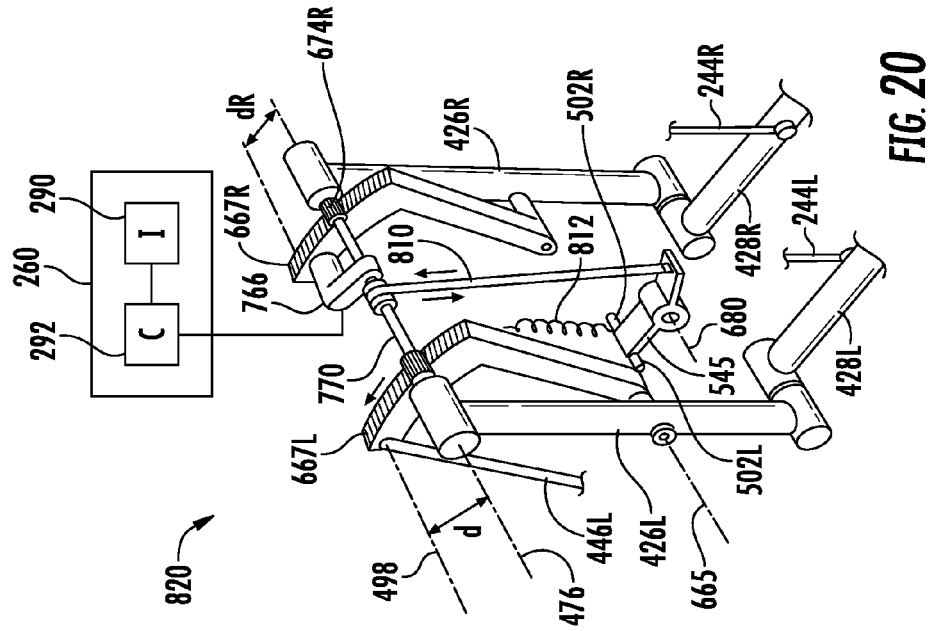


FIG. 17





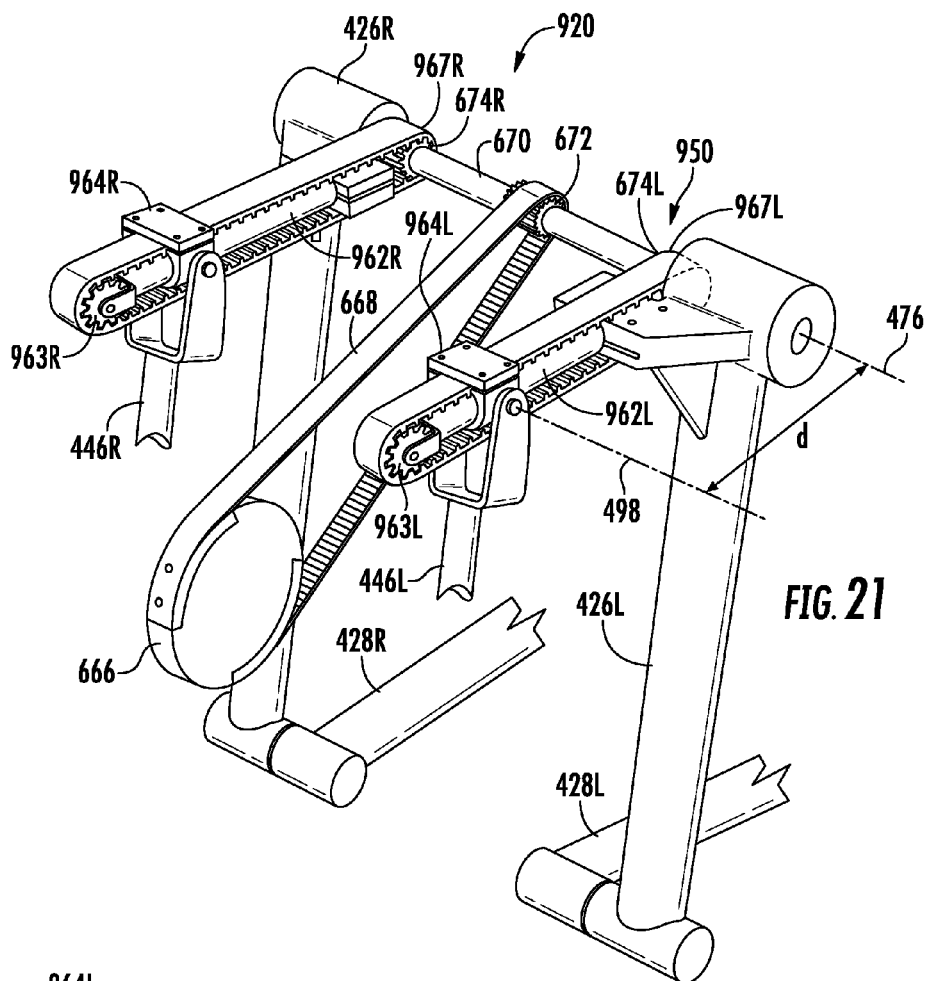


FIG. 21

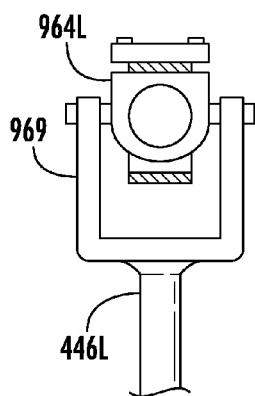
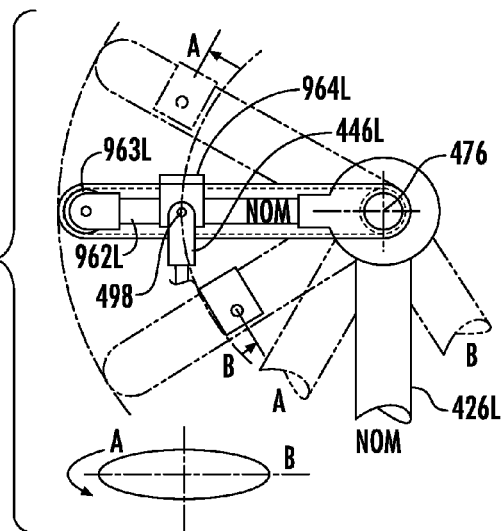
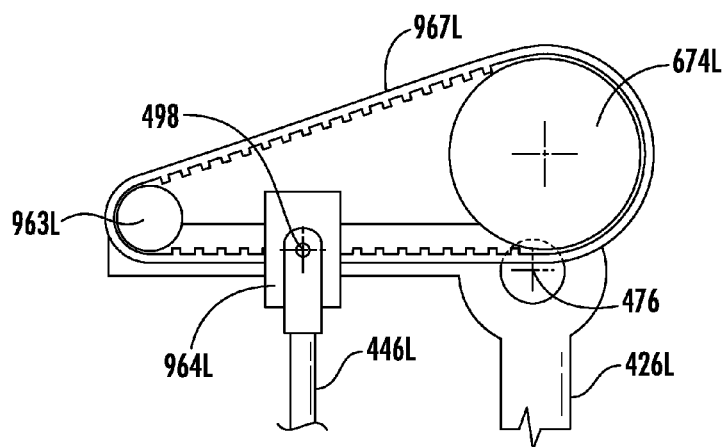
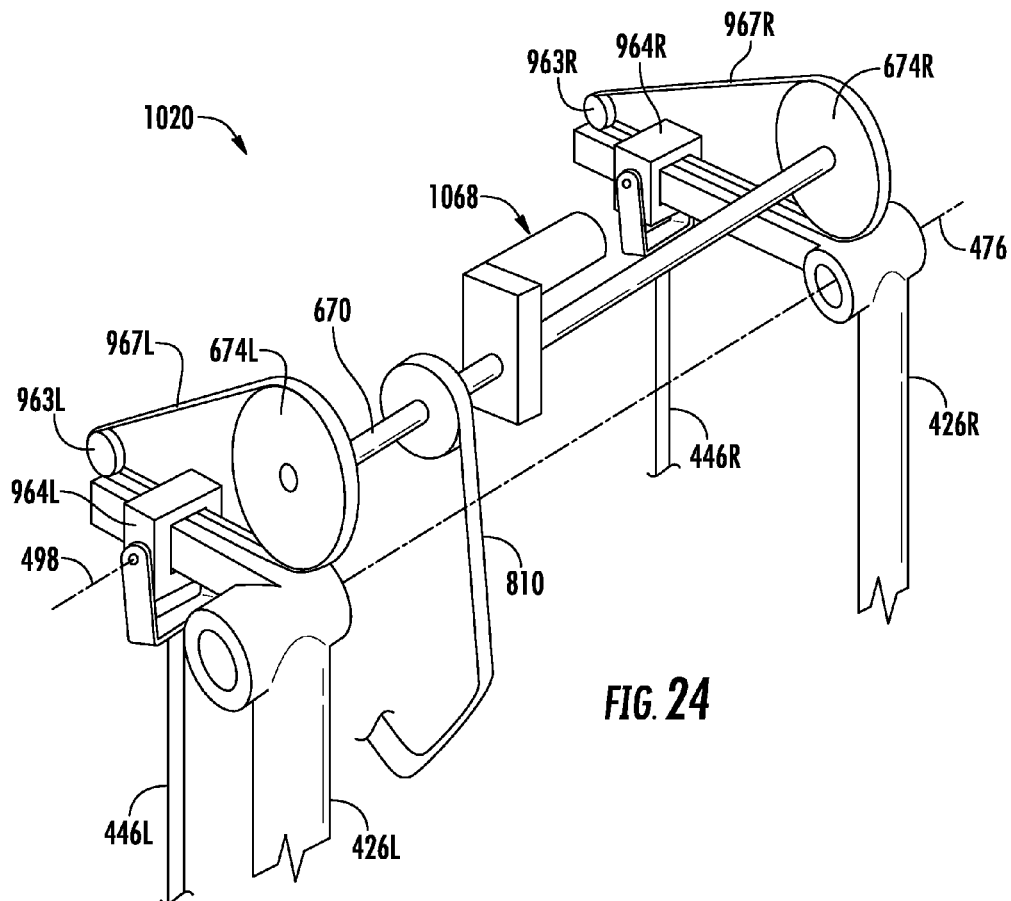


FIG. 22

FIG. 23





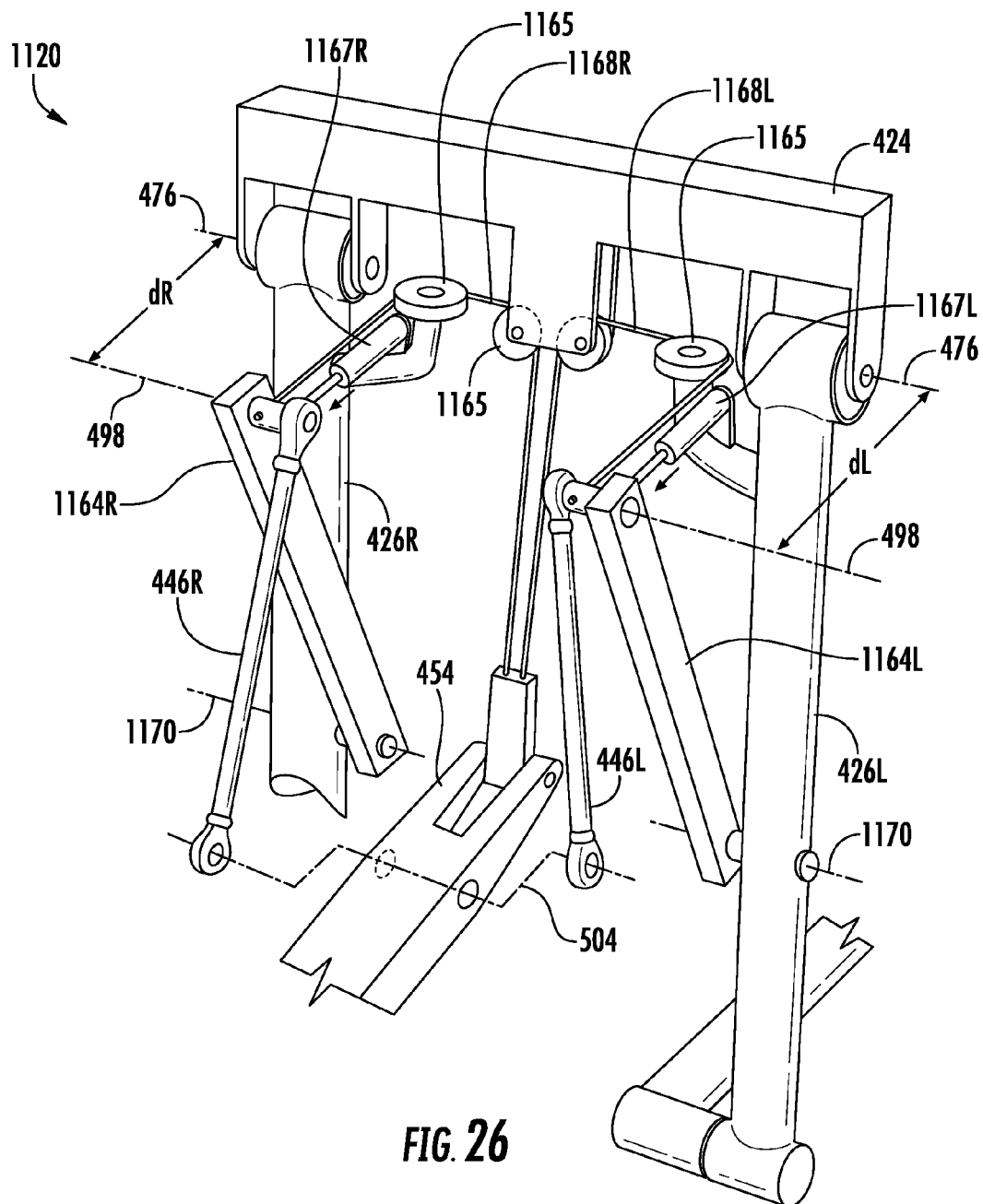


FIG. 26

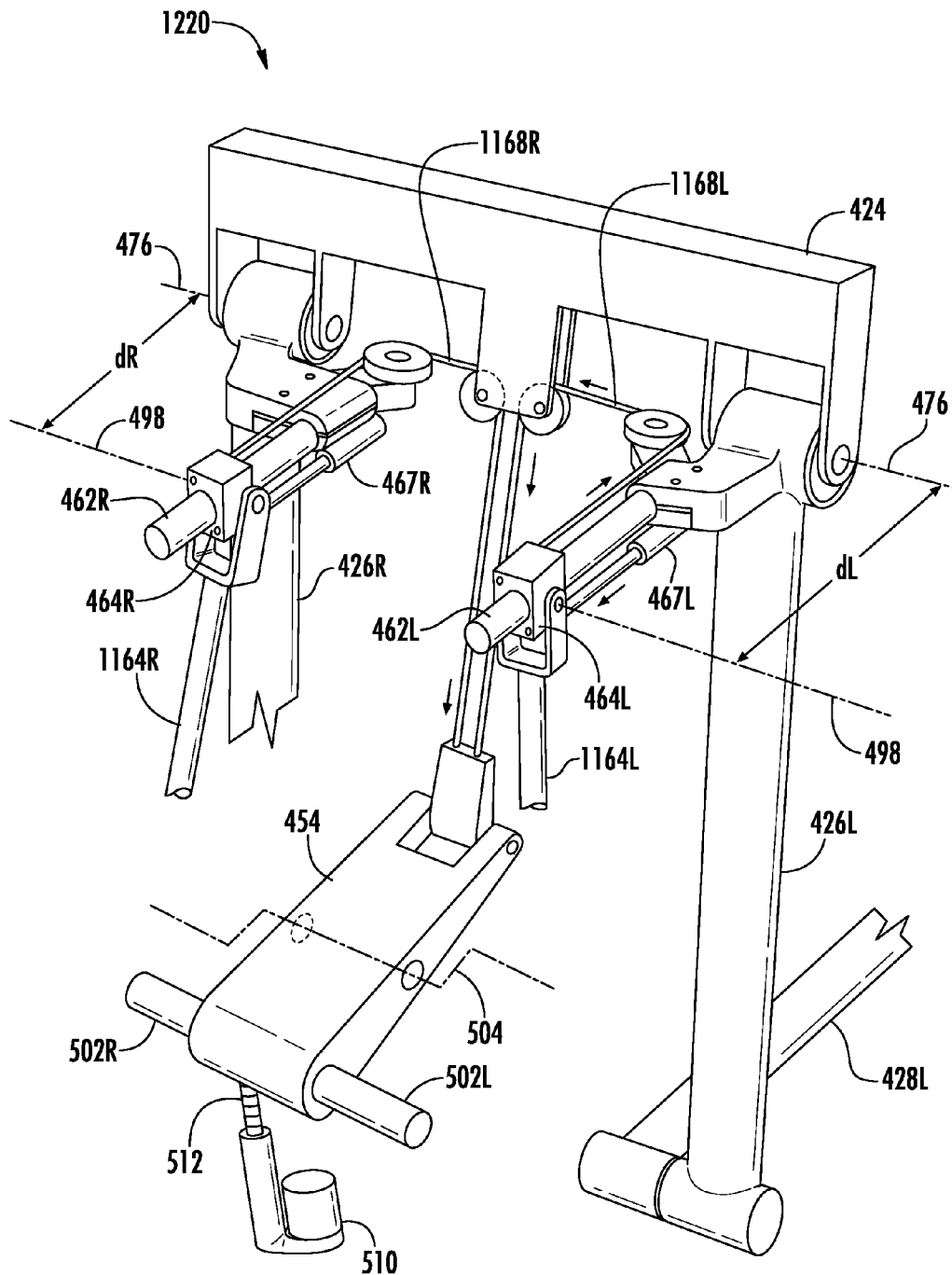
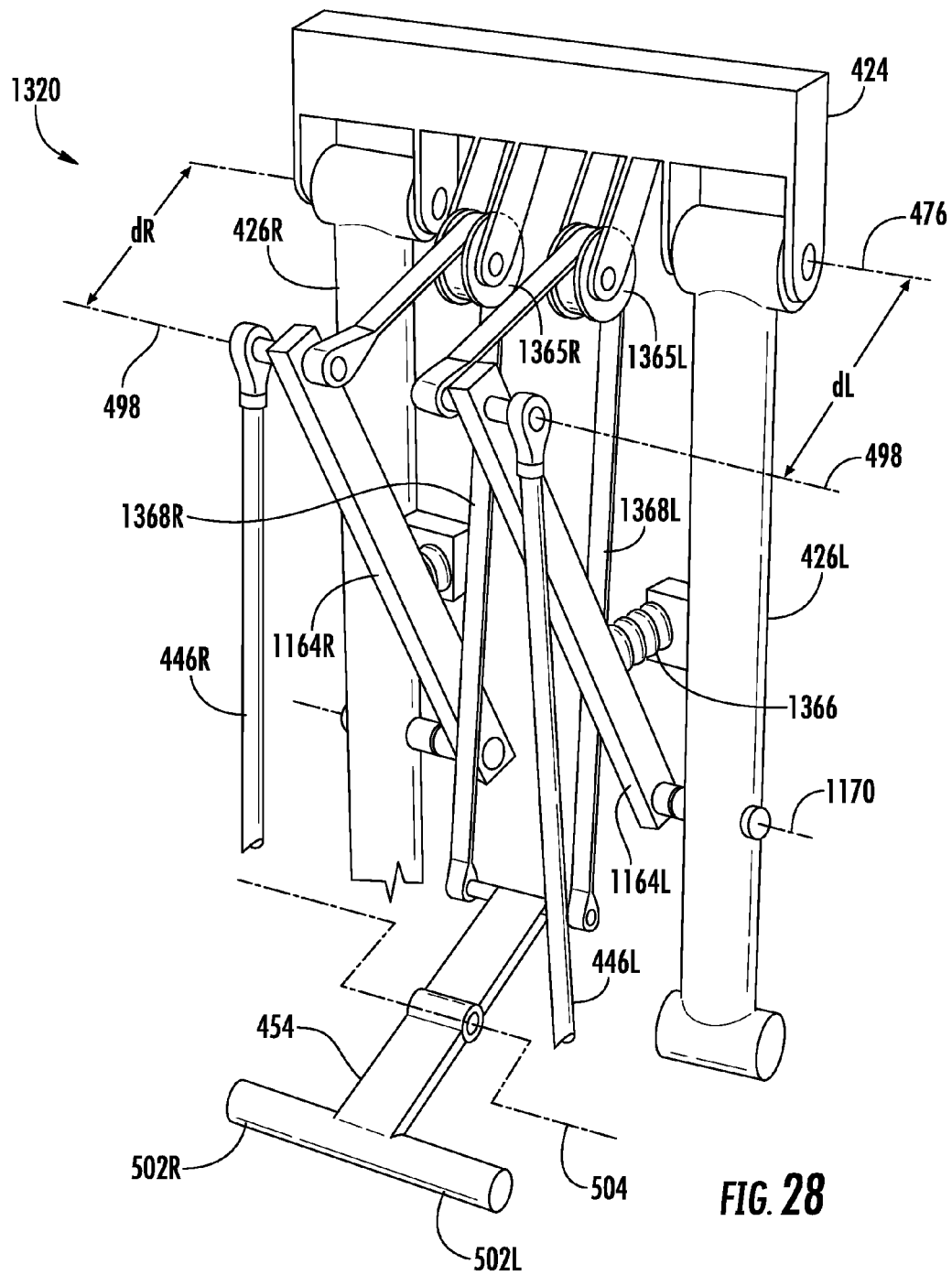
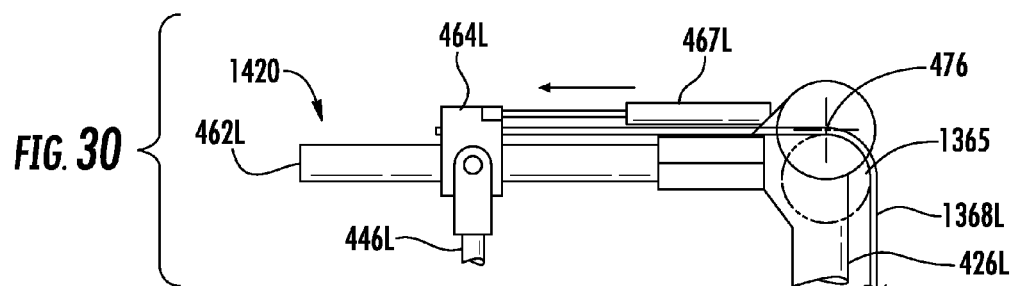
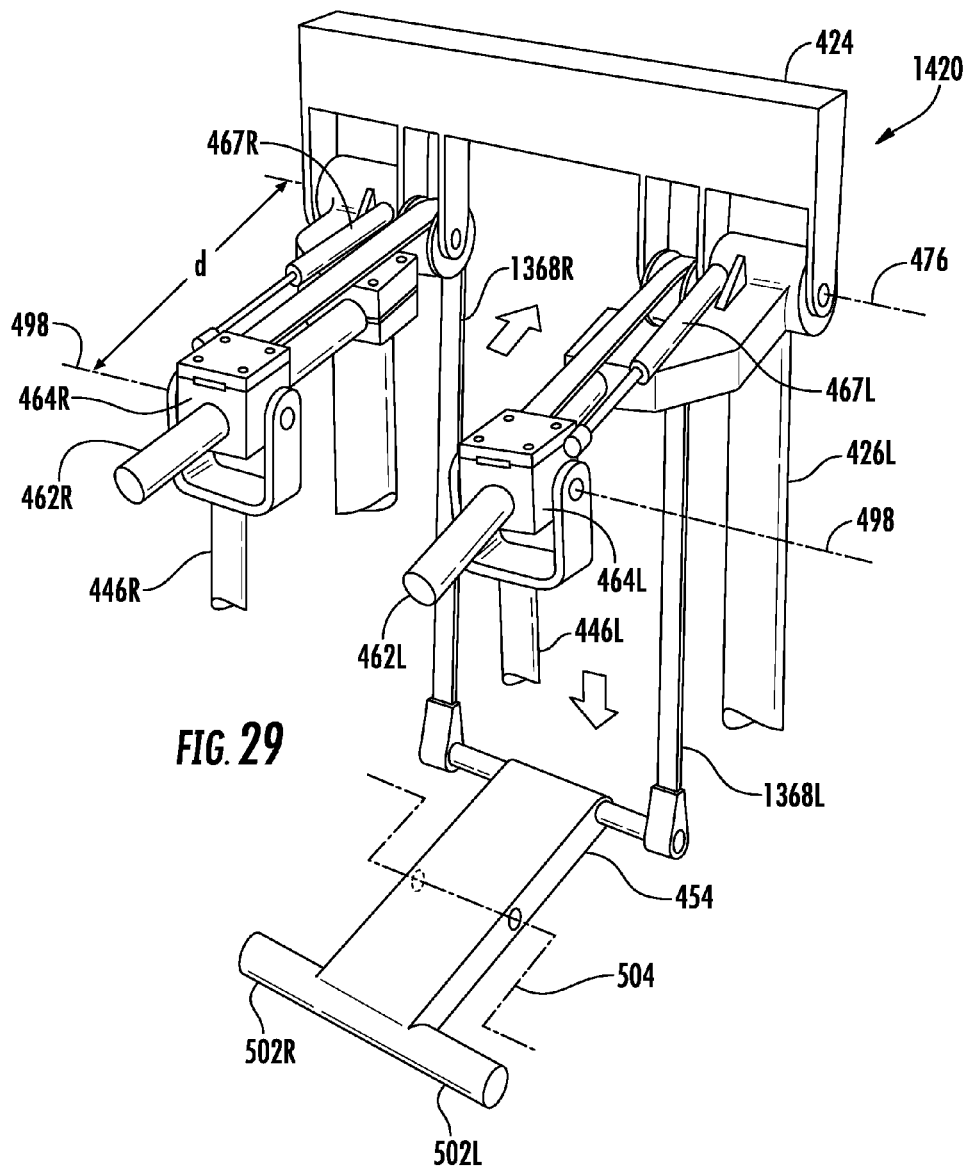
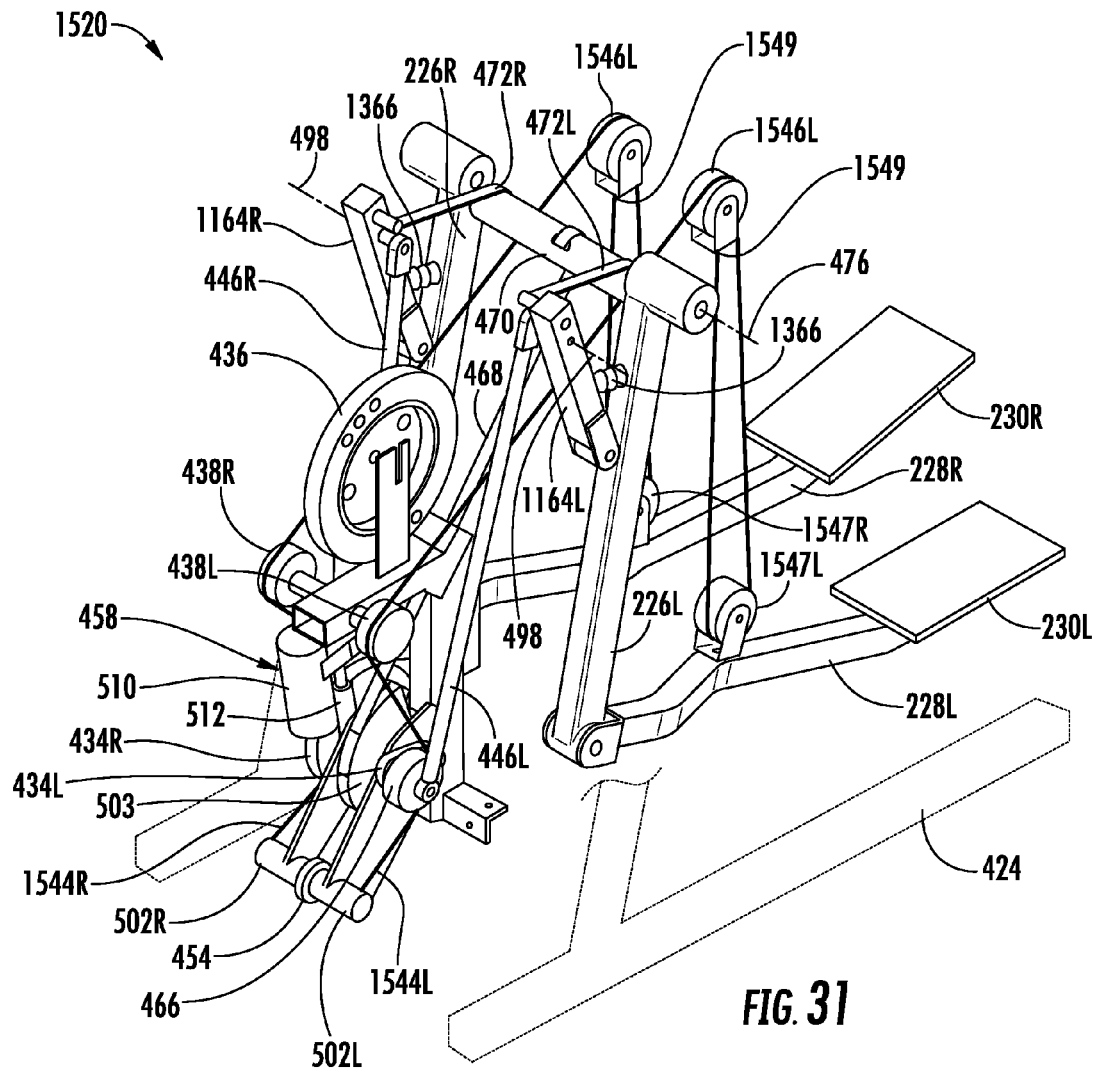


FIG. 27









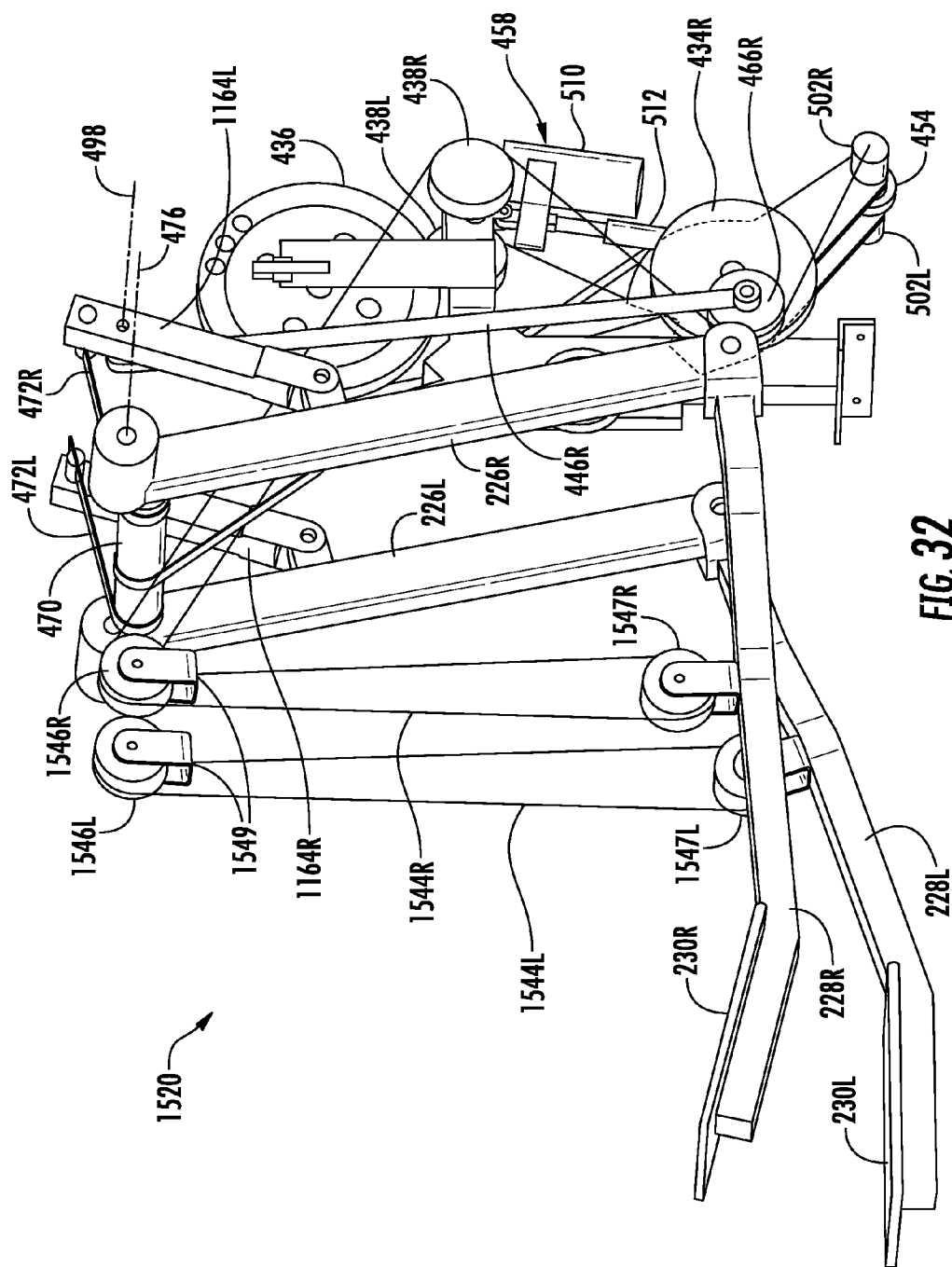


FIG. 32

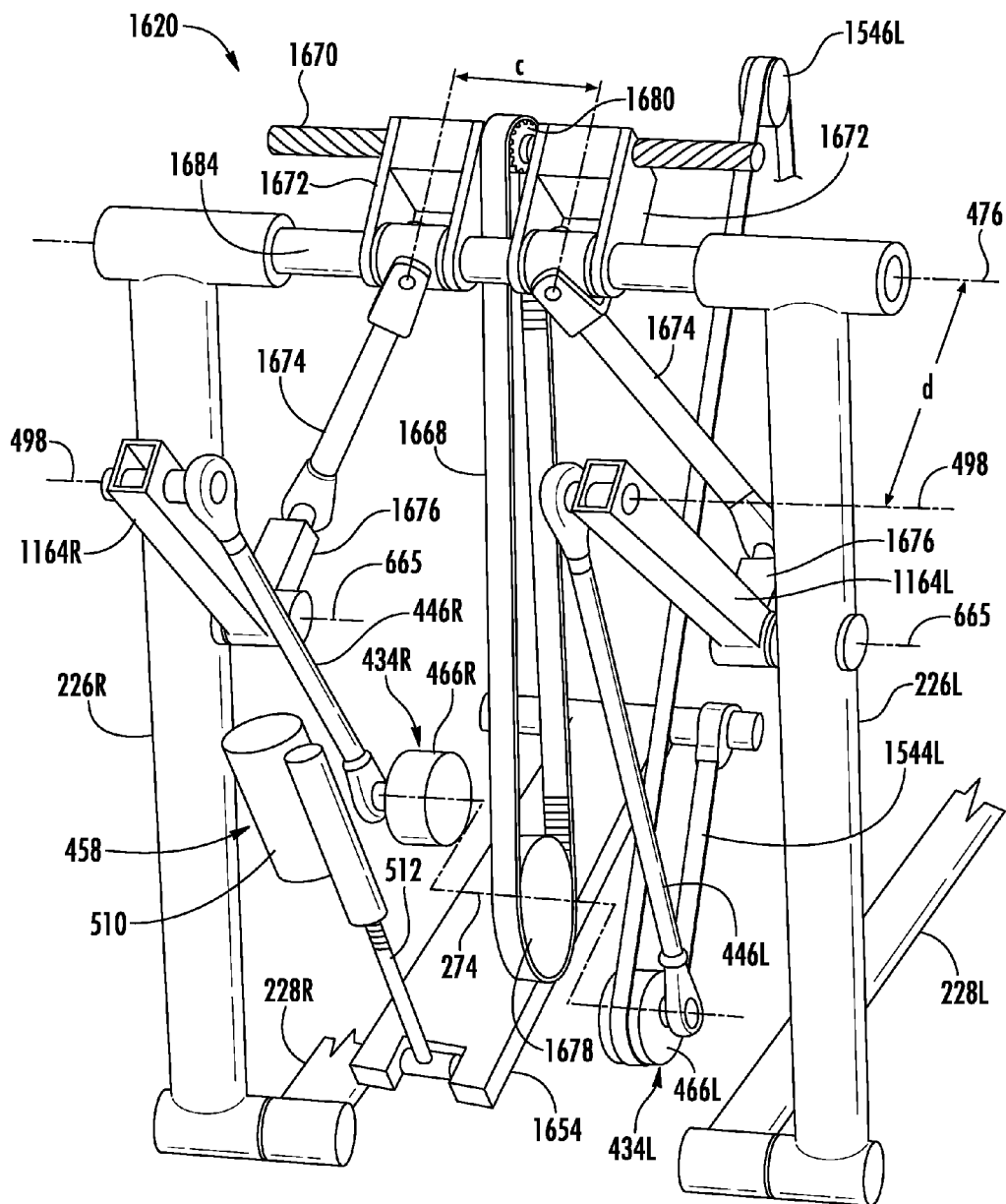


FIG. 33

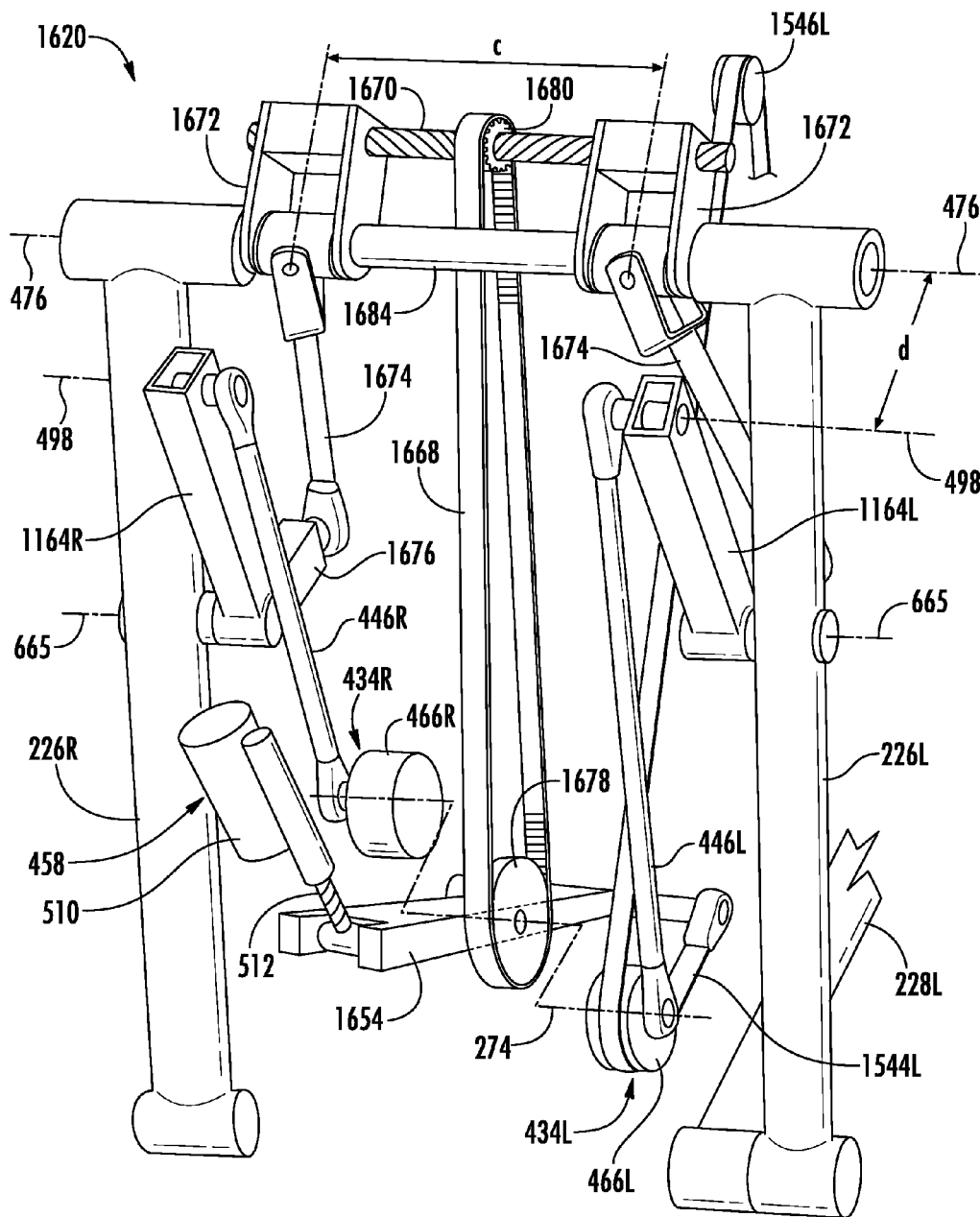


FIG. 34

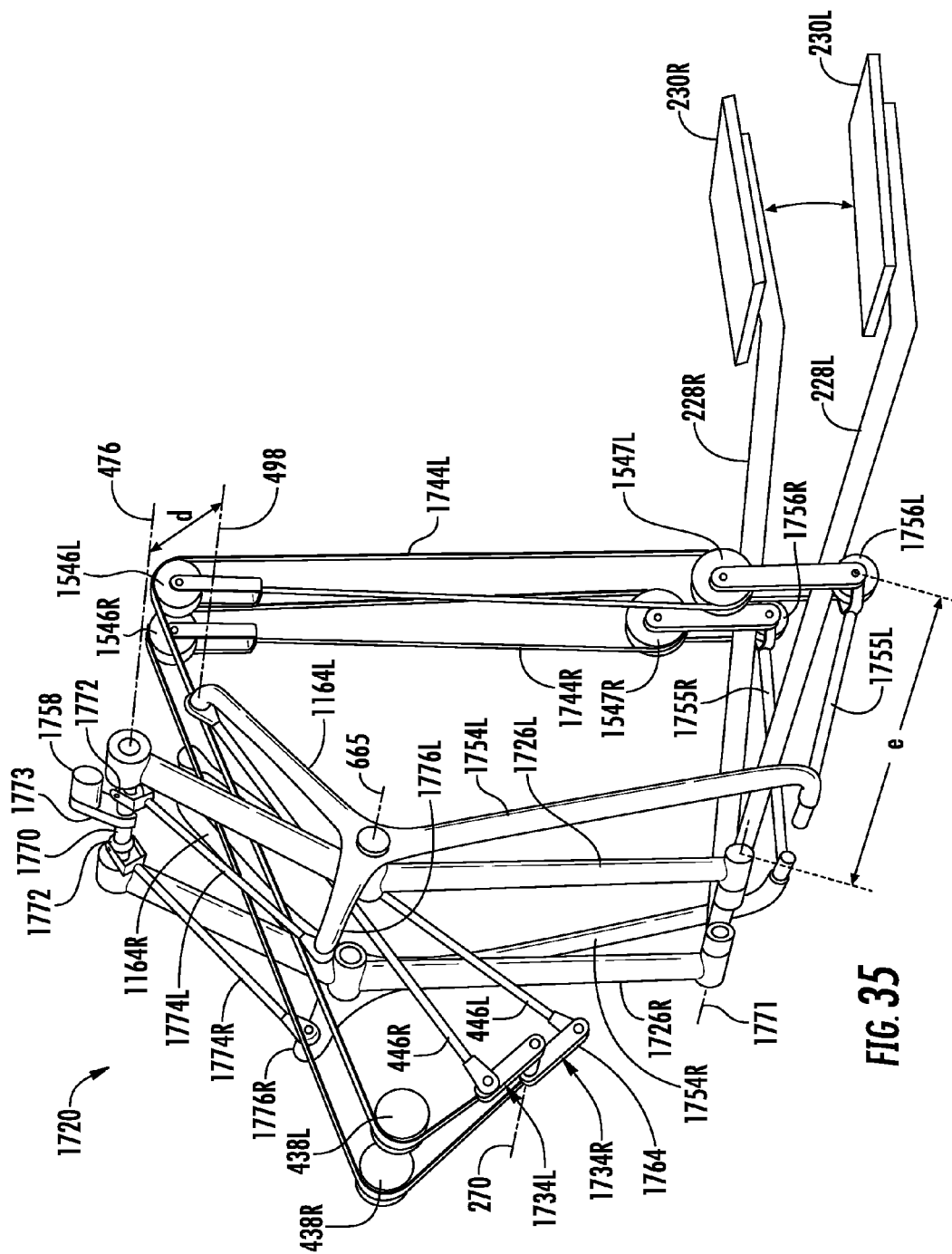
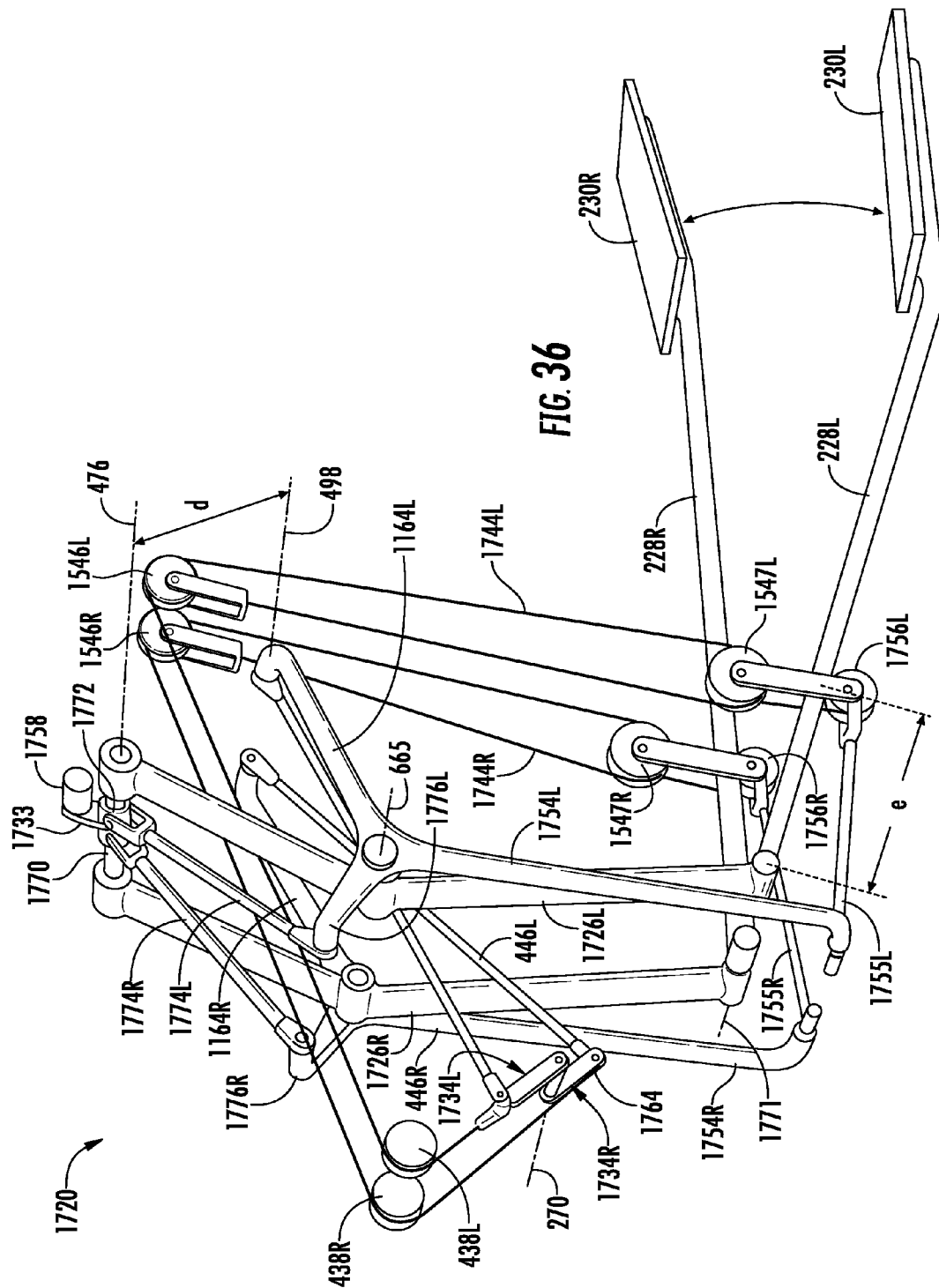


FIG. 35



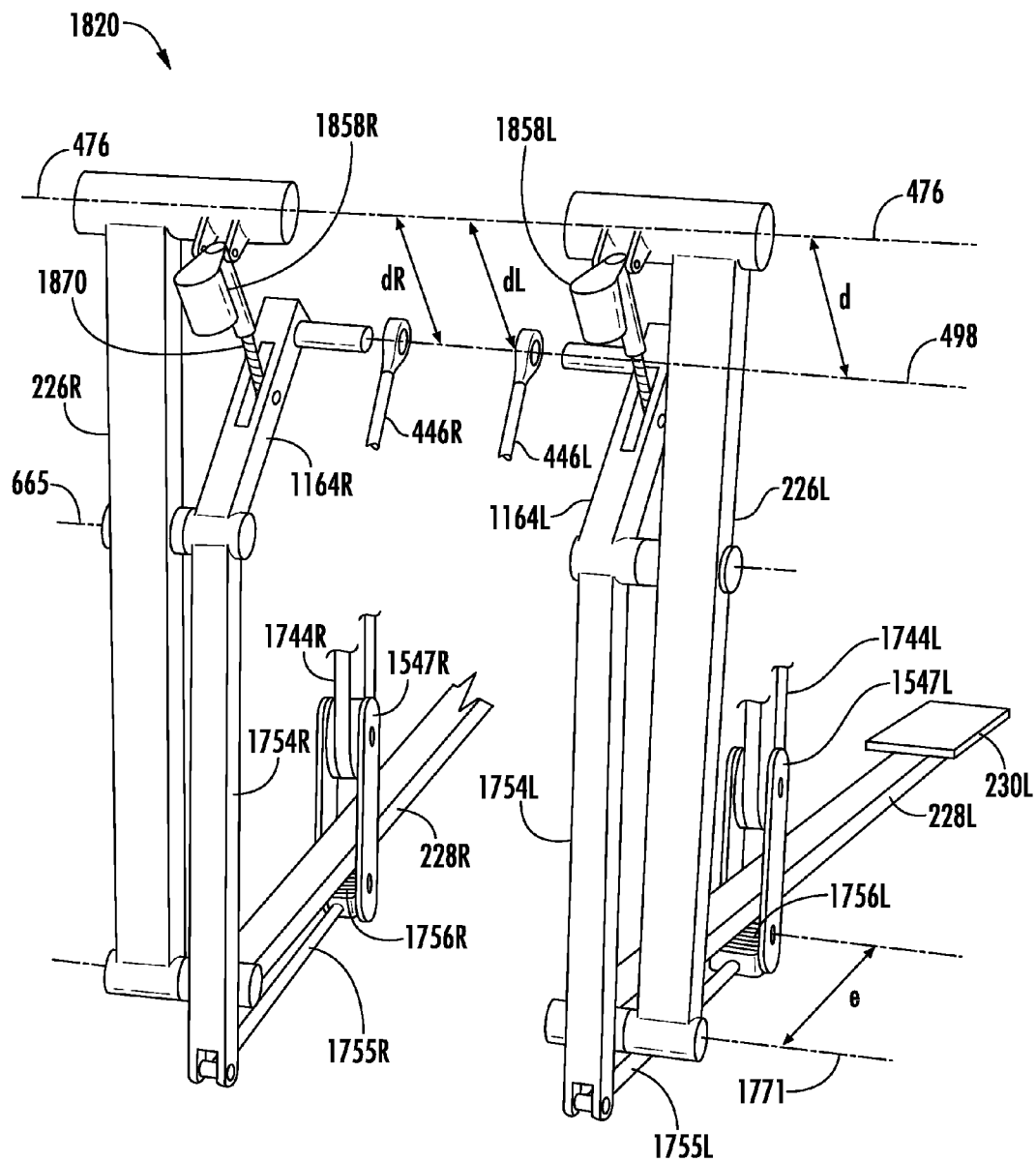


FIG. 37



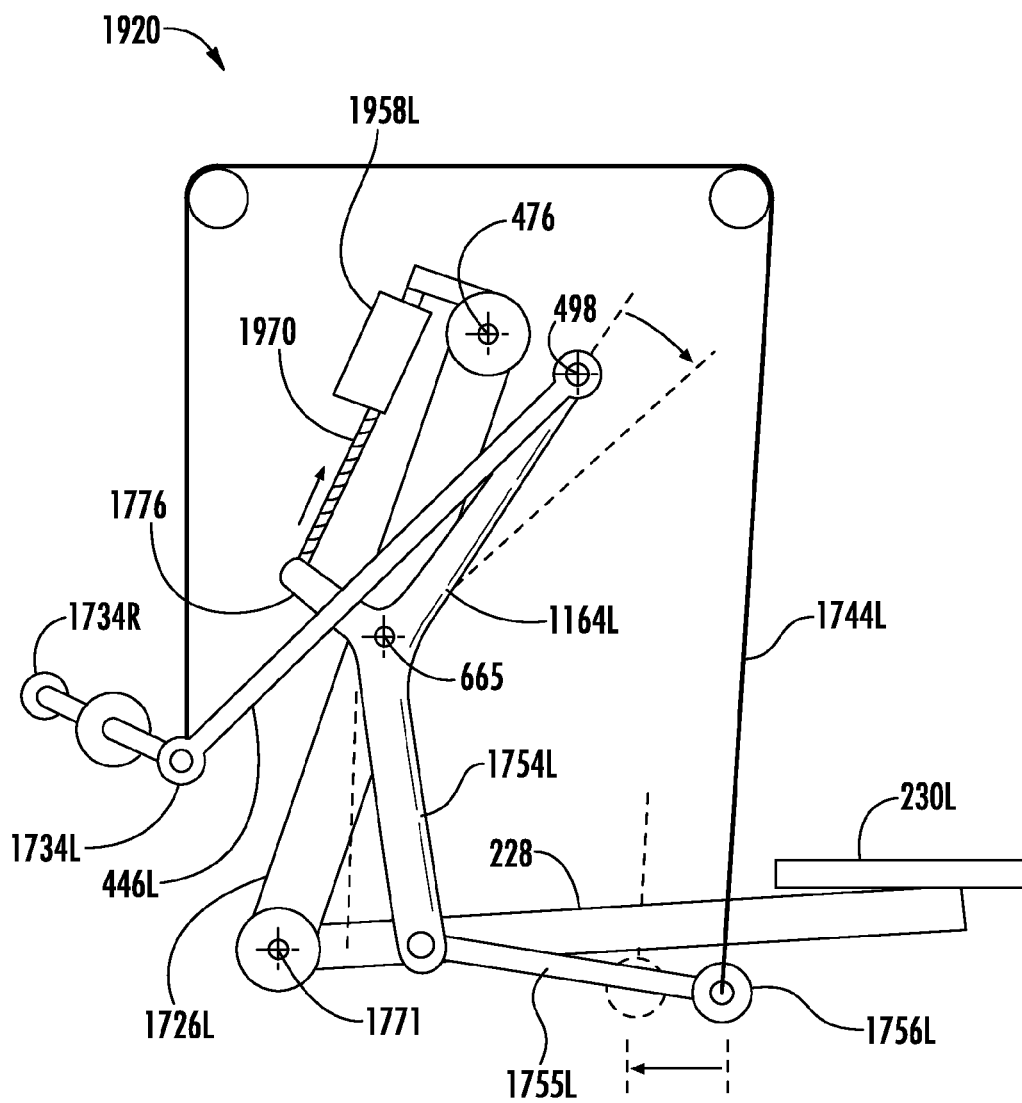


FIG. 38

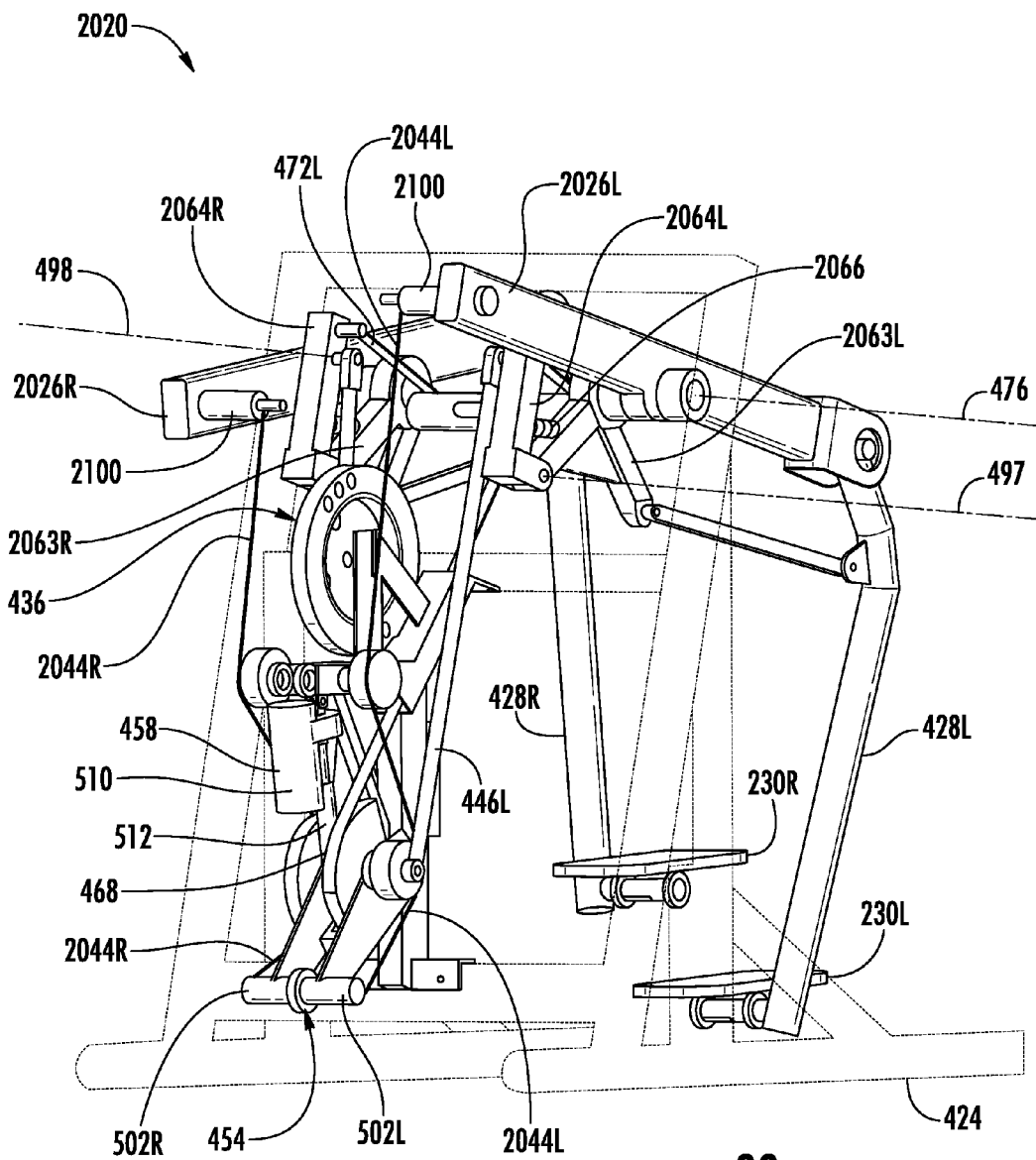
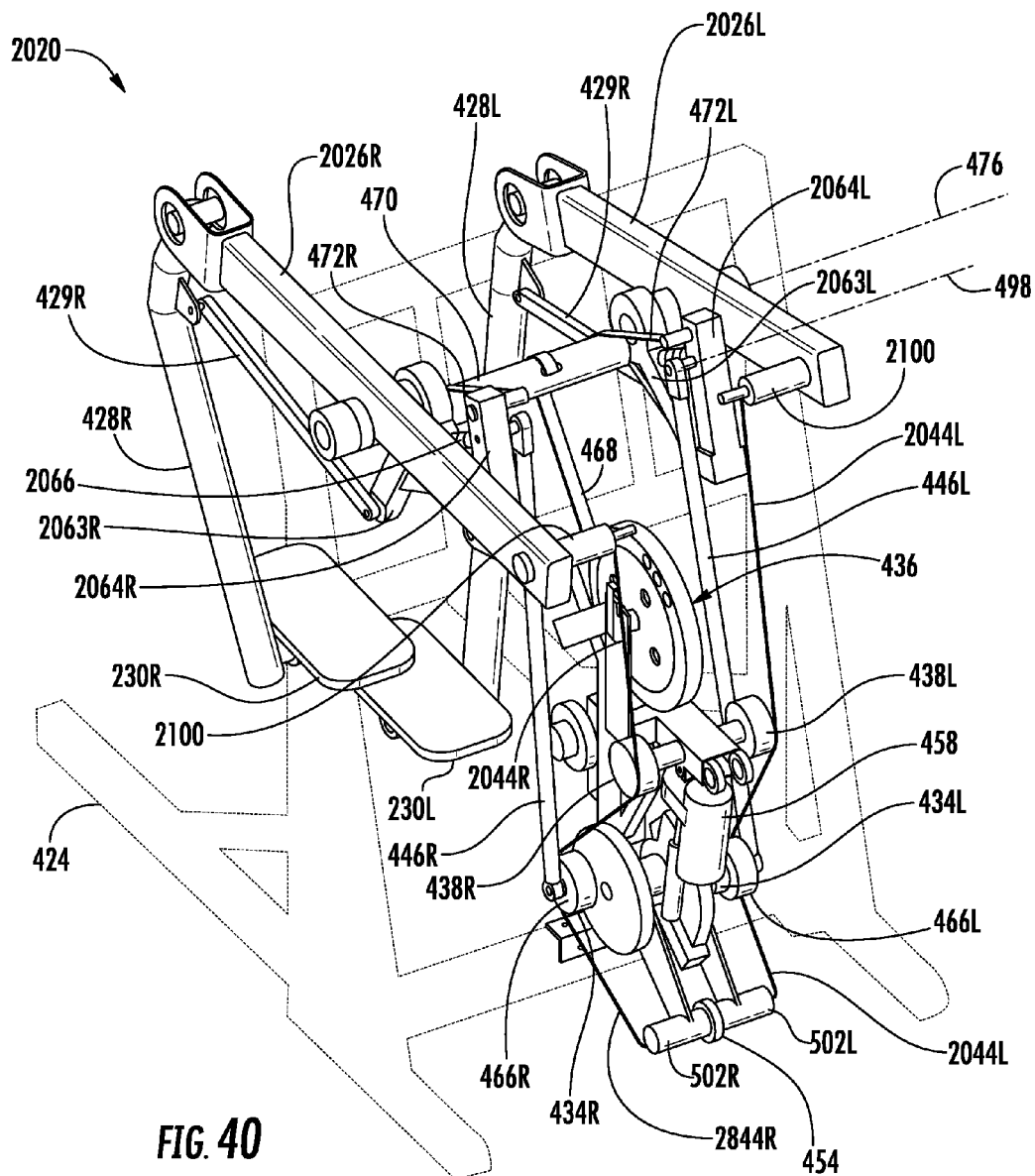
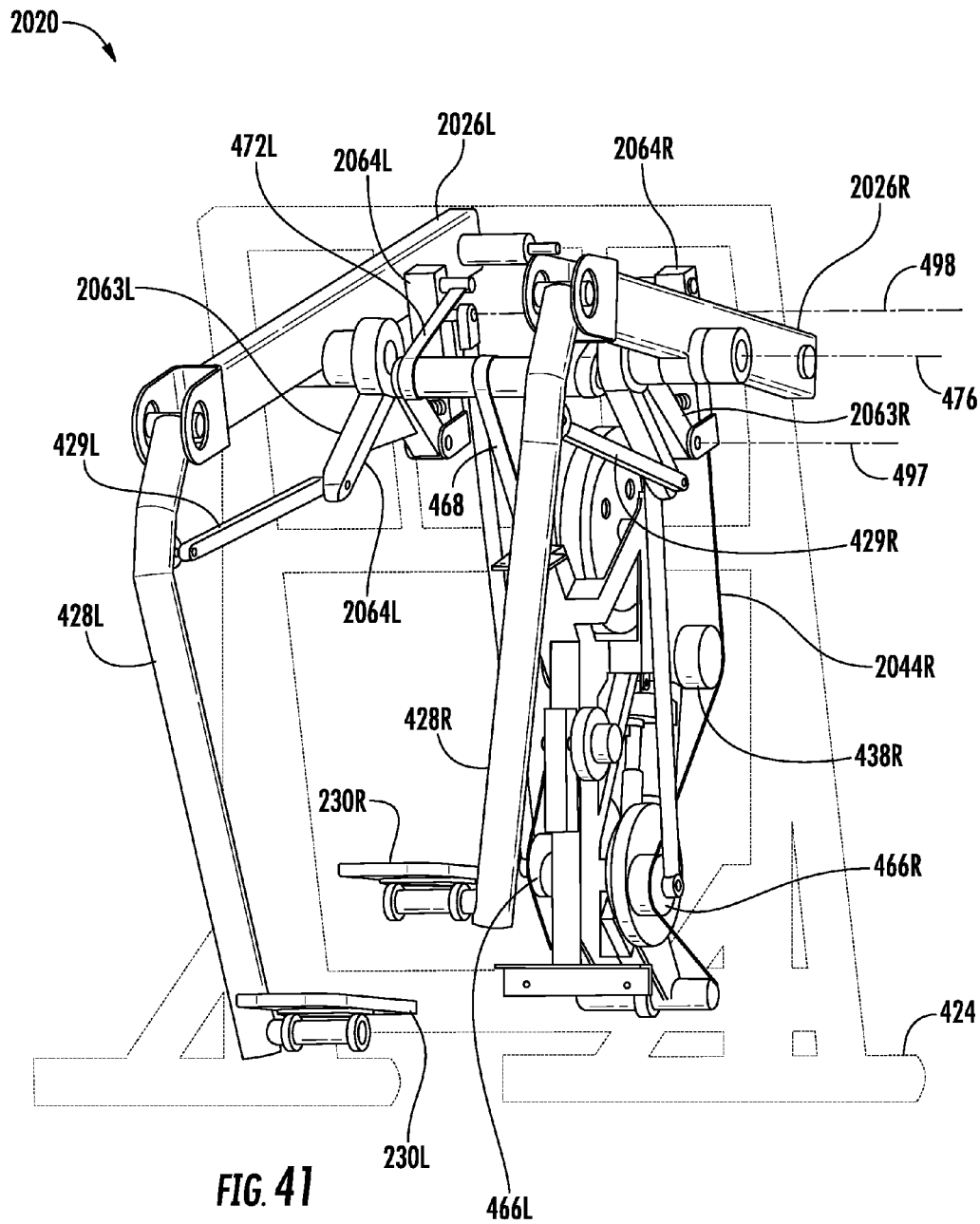
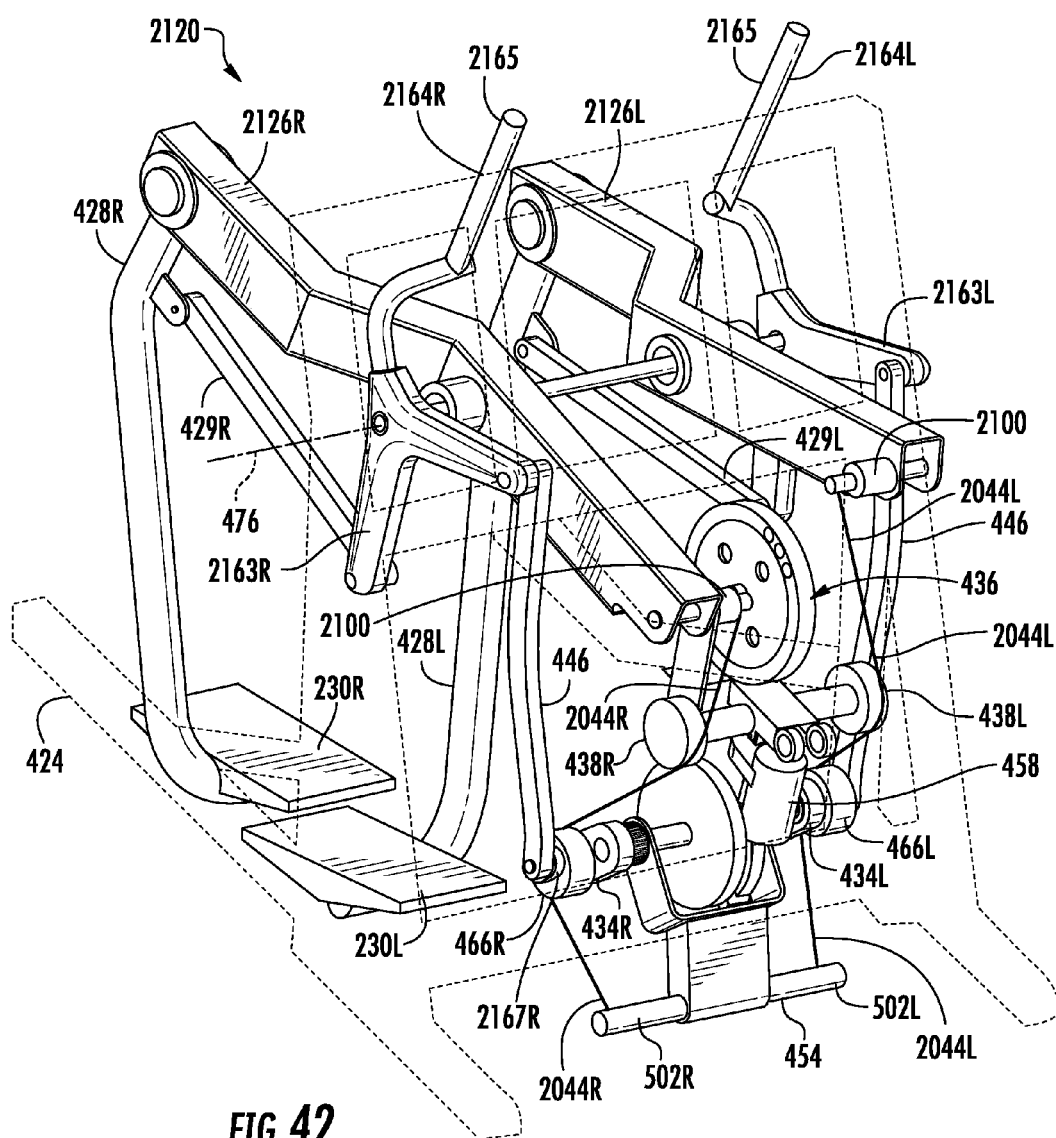
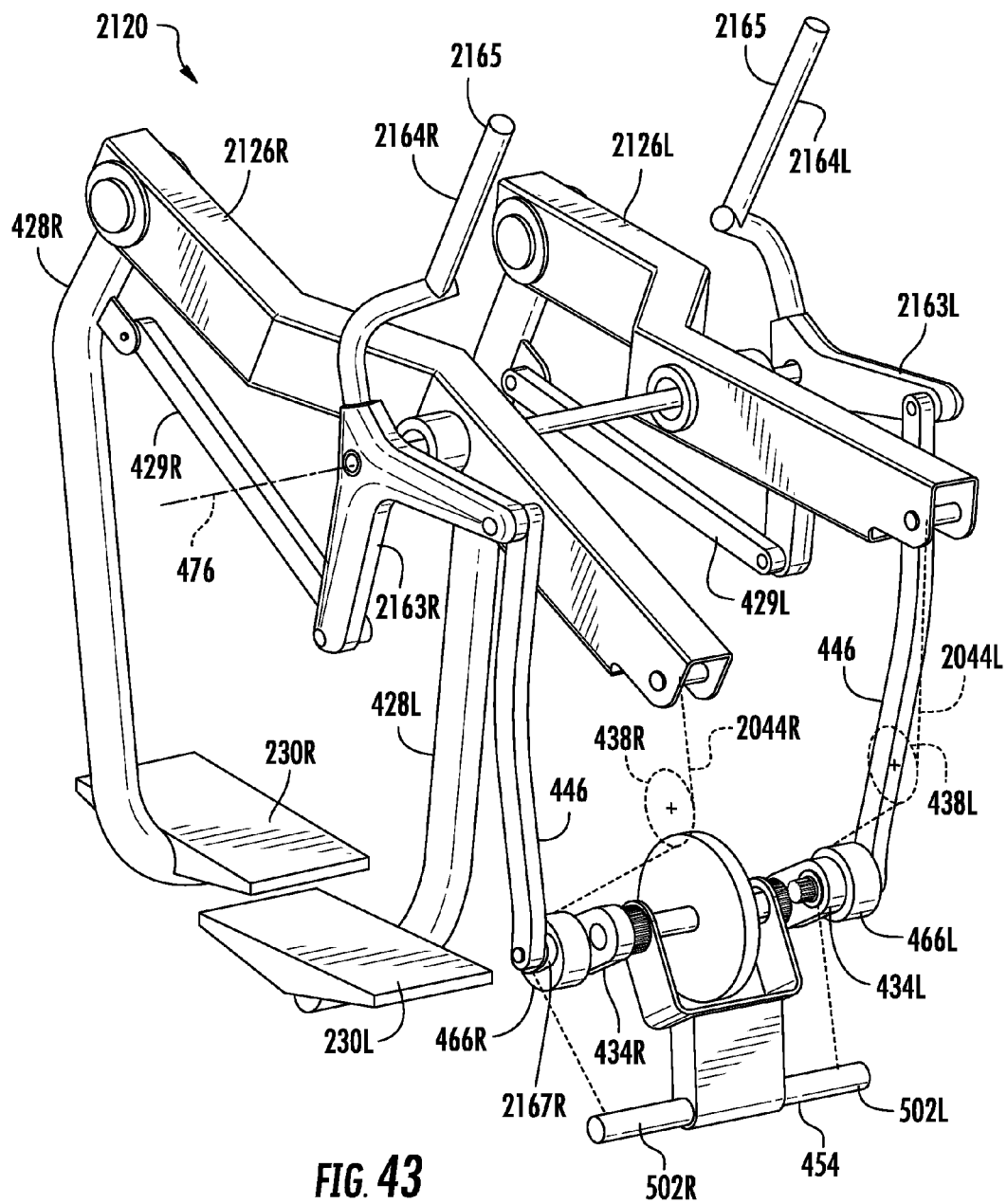


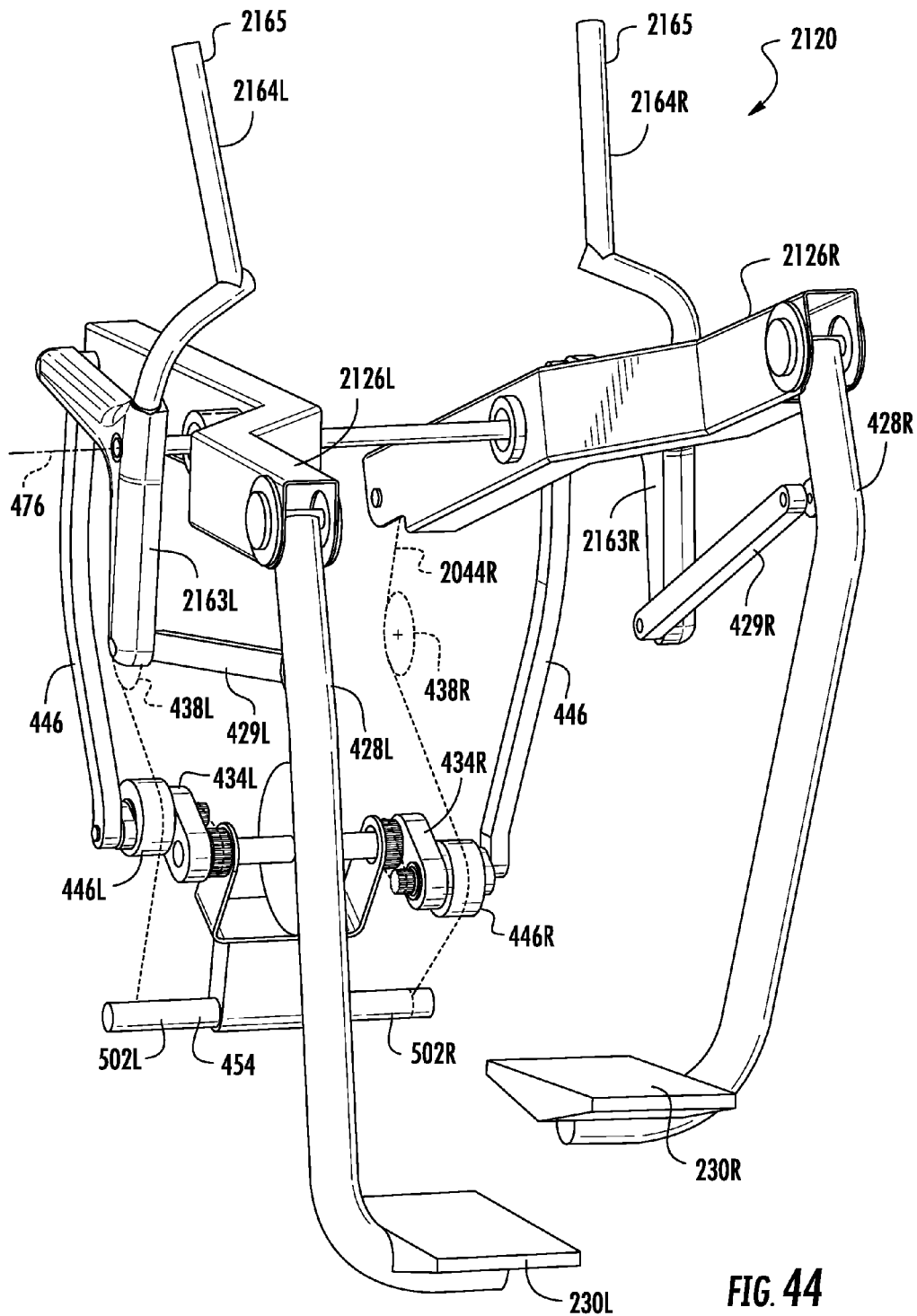
FIG. 39











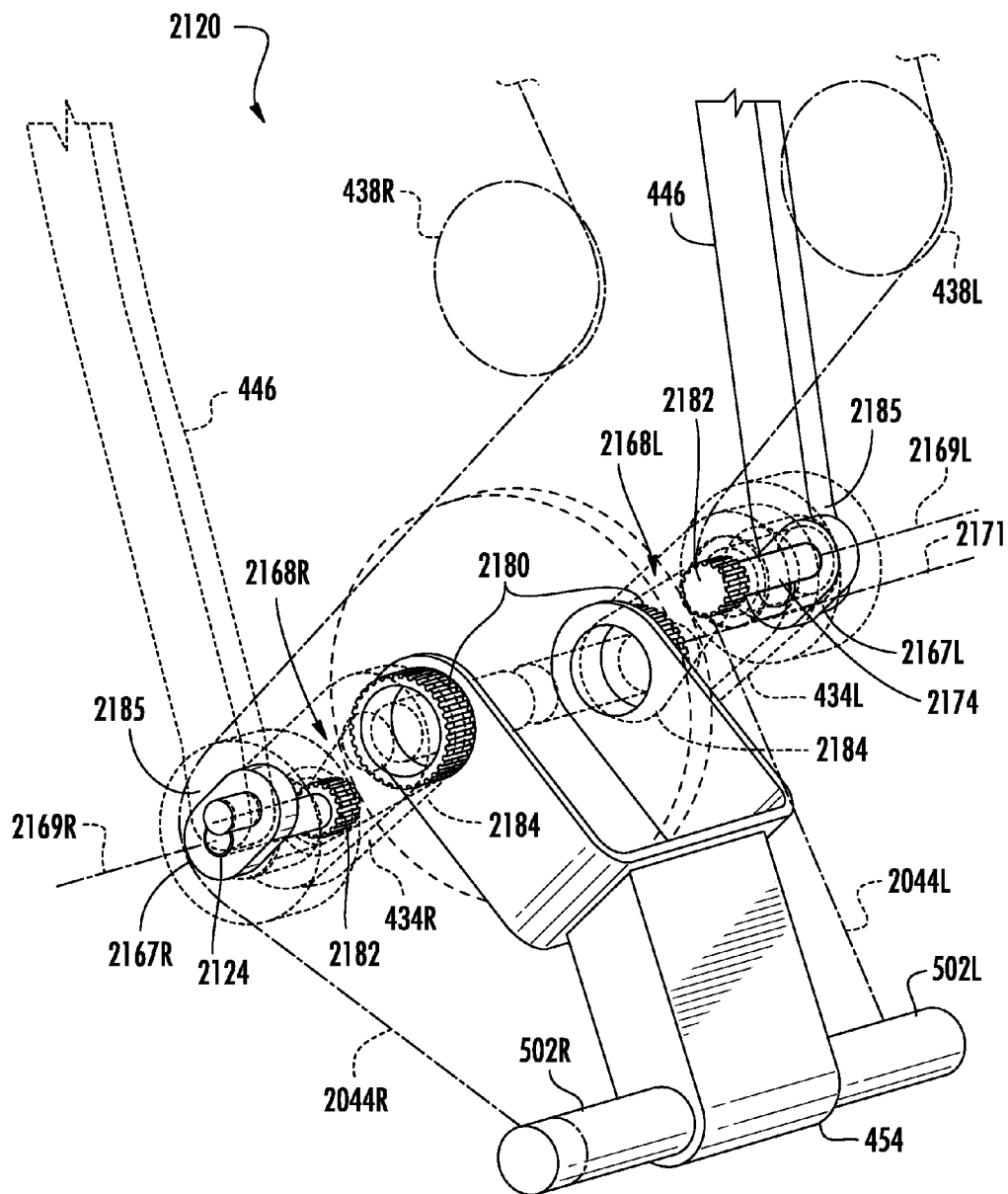
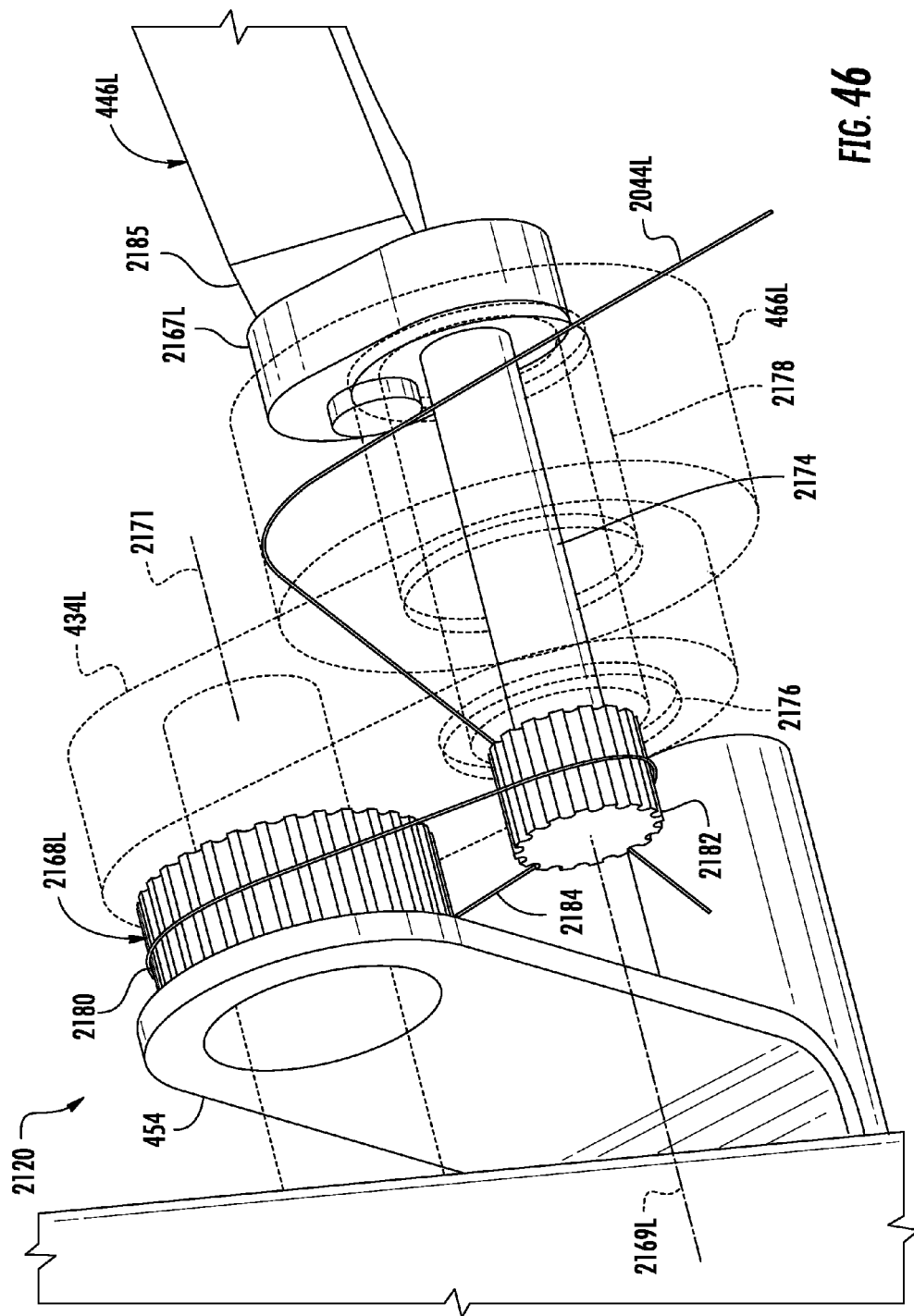


FIG. 45





**FIG. 46**

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**SELECTABLE STRIDE ELLIPTICAL****CROSS-REFERENCE TO RELATED PATENT APPLICATIONS**

The present application claims priority under 35 U.S.C. §120 from co-pending U.S. patent application Ser. No. 14/686,390 filed on Apr. 14, 2015 by Peter J. Arnold and James S. Birrell and entitled SELECTABLE STRIDE ELLIPTICAL, which claimed priority under 35 U.S.C. §119 from U.S. Provisional Patent Application Ser. No. 61/984,727 filed on Apr. 25, 2014, and also claimed priority under 35 U.S.C. §119 from U.S. Provisional Patent Application Ser. No. 62/080,299 filed on Nov. 15, 2014, the full disclosure of which are hereby incorporate by reference.

**BACKGROUND**

Elliptical exercise machines typically comprise foot pedals that are movable along an elliptical path. Such elliptical exercise machines have become a very popular piece of exercise equipment at both health clubs and in homes. Such elliptical exercise machines may at sometimes be confusing to operate or may not provide a comfortable elliptical path. Adaptive motion exercise machines also provide foot pedals that are movable in a variety of elliptical paths or other reciprocal paths, based upon the desired motion of the user. Some users find such foot motion flexibility of such adaptive motion machines to be distracting and confusing to operate.

**BRIEF DESCRIPTION OF THE DRAWINGS**

FIG. 1 is a schematic diagram of an example exercise apparatus.

FIG. 2 is a flow diagram of an example method for operating an exercise apparatus.

FIG. 3 is a front perspective view of an example implementation of the exercise apparatus of FIG. 1.

FIG. 4 is a side perspective view of the exercise apparatus of FIG. 3.

FIG. 5A is a front perspective view of the exercise apparatus of FIG. 3 in a first state and with portions omitted for purposes of illustration.

FIG. 5B is a side view of the exercise apparatus of FIG. 5A.

FIG. 5C is a diagram of a shape of an elliptical path through which footpads of the exercise apparatus of FIG. 5A move when the exercise apparatus is in the first illustrated state.

FIG. 6A is a front perspective view of the exercise apparatus of FIG. 3 in a second state and with portions omitted for purposes of illustration.

FIG. 6B is a side view of the exercise apparatus of FIG. 6A.

FIG. 6C is a diagram of a shape of an elliptical path through which footpads of the exercise apparatus of FIG. 6A move when the exercise apparatus is in the first illustrated state.

FIG. 7 is a side perspective view of another example implementation of the exercise apparatus of FIG. 1.

FIG. 8A is an enlarged side view of footpads of the exercise apparatus of FIG. 7 at a maximum step height and at midstride.

FIG. 8B is an enlarged top view of the footpads shown in FIG. 8A.

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FIG. 8C is an enlarged side view of the footpads of the exercise apparatus of FIG. 7 at a maximum stride and at a mid-step.

FIG. 8D is an enlarged top view of the footpads shown in FIG. 8C.

FIG. 9 is a front right perspective view of another example implementation of the exercise apparatus of FIG. 1.

FIG. 10 is a front left perspective view of the exercise apparatus of FIG. 9.

FIG. 11 is a top rear perspective view of the exercise apparatus of FIG. 9.

FIGS. 12-15 are side views of one side of the exercise apparatus of FIG. 9 that illustrate movement of a foot pad through an elliptical path while the exercise apparatus is in a first state.

FIG. 16 is a side view of one side of the exercise apparatus of FIG. 9 in a second state, illustrating the resulting elliptical path for the foot pad.

FIG. 17 is a side view of one side of the exercise apparatus of FIG. 9 in a third state, illustrating the resulting elliptical path for the foot pad.

FIG. 18 is a front perspective view of another example implementation of the exercise apparatus of FIG. 1.

FIG. 19 is a rear perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 20 is a rear perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 21 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 22 is an enlarged fragmentary view of a portion of the exercise apparatus of FIG. 21.

FIG. 23 is a fragmentary side view of a portion of the exercise apparatus of FIG. 21 illustrating motion during reciprocation of footpads of the exercise apparatus of FIG. 21.

FIG. 24 is a rear perspective view of another example implementation of the exercise apparatus of FIG. 1.

FIG. 25 is a fragmentary side view of the exercise apparatus of FIG. 24.

FIG. 26 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 27 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 28 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 29 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 30 is a fragmentary side view of a portion of the exercise apparatus of FIG. 29.

FIG. 31 is a front perspective view of another example implementation of the exercise apparatus of FIG. 1.

FIG. 32 is a rear perspective view of the exercise apparatus of FIG. 31.

FIG. 33 is a front perspective view of a portion of another example implementation of the exercise apparatus FIG. 1 in a first state.

FIG. 34 is a front perspective view of the portion of the exercise apparatus of FIG. 33 in a second state.

FIG. 35 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1 in a first state.

FIG. 36 is a front perspective view of the portion of the exercise apparatus of FIG. 35 in a second state.

FIG. 37 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 38 is a side view of a portion of another example implementation of the exercise apparatus of FIG. 1.

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FIG. 39 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 40 is another front perspective view of the portion of the exercise apparatus of FIG. 39.

FIG. 41 is a rear perspective view of the portion of the exercise apparatus of FIG. 39.

FIG. 42 is a front perspective view of a portion of another example implementation of the exercise apparatus of FIG. 1.

FIG. 43 is a front perspective view of a portion of the exercise apparatus of FIG. 42.

FIG. 44 is a rear perspective view of a portion of the exercise apparatus of FIG. 42.

FIG. 45 is a front perspective view of a portion of the exercise apparatus of FIG. 42.

FIG. 46 is a front perspective view of a portion of the exercise apparatus of FIG. 42.

#### DETAILED DESCRIPTION OF EXAMPLES

FIG. 1 schematically illustrates an example fitness equipment unit or exercise apparatus 20. As will be described hereafter, exercise apparatus 20 provides simpler operation and, in some embodiments, may facilitate a more natural elliptical path of motion during exercise. Exercise apparatus 20 comprises frame 24, left foot link 28L, right foot link 28R (collectively referred to as foot links 28) left foot pad 30L, right foot pad 30R (collectively referred to as foot pads 30), and adjustment synchronizer 50.

Frame 24 (as schematically illustrated) comprises a foundation, base or other structure or groups of structures that support the remaining components of exercise apparatus 20. In the example illustrated, base 24 has a centerline 52 longitudinally extending in a front to rear direction.

Foot links 28 comprise structures that support foot pads 30. Foot pads 30 comprise platforms upon which a person exercising places his or her feet during exercise and against which a person applies force to move foot pads 30 along an elliptical path. As schematically illustrated by line 54, foot pads 30 are linked to one another to move in unison along the same elliptical path (paths of the same shape), wherein the paths taken by foot pads 30 are of the same elliptical shape, but are out of phase with one another. In the example illustrated, foot pads 30 move through elliptical paths of the same shape, but which are 180° out of phase with respect to one another. For example, when foot pad 30L is at the forward-most position along the shape of the elliptical path, foot pad 30R is at the rearward-most position along the shape of the elliptical path.

Adjustment synchronizer 50 comprises an adjustment mechanism that is operably coupled to foot links 28 and foot pads 30 (as schematically illustrated by lines 56) so as to synchronously adjust both a step height and a stride length of the shape of the elliptical path that is currently being taken by each of foot pads 30. For purposes of this disclosure, the term “coupled” shall mean the joining of two members directly or indirectly to one another. Such joining may be stationary in nature or movable in nature. Such joining may be achieved with the two members or the two members and any additional intermediate members being integrally formed as a single unitary body with one another or with the two members or the two members and any additional intermediate member being attached to one another. Such joining may be permanent in nature or alternatively may be removable or releasable in nature. The term “operably coupled” shall mean that two members are directly or

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indirectly joined such that motion may be transmitted from one member to the other member directly or via intermediate members.

Adjustment synchronizer 50 simultaneously or concurrently adjusts the step height and the stride length of the elliptical path being taken by foot pads 30 in a synchronous manner in response to a single adjustment request. In one implementation, a single adjustment request is in the form of an electronic control signal generated in response to the person exercising manually entering the request using an input device, such as a pushbutton, touchscreen, touchpad, portable electronic device connected to or in communication with exercise apparatus 20 or microphone with associated speech recognition hardware and software. In yet another implementation, the single adjustment request is in the form of an electronic control signal generated in response to an exercise program calling for adjustment of the elliptical path being taken by members 30 during an exercise routine or workout.

Because adjustment synchronizer 50 concurrently or synchronously adjusts both the step height and the stride length of the elliptical path of foot pad 30, exercise apparatus 20 facilitates a greater degree of control of a proportional relationship between the step height and the stride length. In other words, the proportional relationship between the step height and the stride length may be maintained within certain predefined relationships predetermined as being more natural, predetermined as being best-suited for a particular size or other characteristic of the person exercising or predetermined as being best-suited for a particular fitness objective. Because adjustment synchronizer 50 facilitates a single input to adjust the synchronizer 50, adjustment by a person exercising may be performed through a single input to the exercise apparatus 20, providing ease-of-use and allowing the person exercising to focus on the exercise being performed.

In the example illustrated in FIG. 1, the synchronous or coordinated adjustment of both the step height and the stride length of the same elliptical path being taken by each of foot pads 30 further facilitates greater control over the coordinated movement of foot pads 30 such that foot pads 30 are moved along elliptical paths in vertical planes that transversely closer to one another and closer to the centerline 52. In such an implementation, inner portions footpads 30 vertically overlap one another along the centerline 52, wherein footpad 30L (when at a 12:00 position) overlaps the underlying footpad 30R (when at a 6:00 position) and vice versa. The coordinated or synchronized adjustment of the step height and stride length helps to ensure that the actual positions of the footpads 30 do not meet at the overlapping points along the centerline 52 and do not collide.

In yet another implementation, greater control over the coordinated movement of foot pads 30 facilitates movement of the footpads in converging or diverging planes, allowing such paths of foot pads 30 to be more natural or that are more similar to a natural stride of a person jogging or running. In particular, a person's natural stride frequently results in the front foot landing below the person's center of mass, proximate a center of the path being taken by the person running. In such an alternative implementation, the movement of foot pads 30 along the elliptical path is guided or controlled such that when a foot pad is at the forward-most, lowermost point of the elliptical path being taken, the footpad is closer to the centerline 52 or crosses the centerline 52 to a greater extent as compared to the corresponding location of the other footpad 30. In other words, the forward-most footpad 30 is closer to centerline 52 as compared to the rearward-

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most footpad 30. The coordinated or synchronized adjustment of the step height and stride length helps to ensure that, although the elliptical path of each of the footpad 30 overlap, the actual positions of the footpads 30 never meet at the overlapping points along centerline 52. As a result, footpads 30 do not collide.

For purposes of this disclosure, the term “step height” refers to the vertical distance between a lowest point and the highest point of any one elliptical path. The term “stride length” refers to the distance between the forward-most point and the rearward-most point of any one elliptical path. In the example illustrated, the adjustment of the step height and the stride length results in a change in the shape of the elliptical path being taken. For purposes of this disclosure, the term “elliptical path” refers to a continuous loop in space having no ends corresponding to and resulting from rotation of a crank through one single complete 360° revolution.

FIG. 2 is a flow diagram of an example method 100 that may be carried out by exercise apparatus 20 or another similar exercise apparatus. As indicated by block 102, left and right footpad 30 are movably supported for movement through an elliptical path. Although each footpad 30 moves through its own path, each of footpads 30 move through an identically shaped elliptical path.

As indicated by block 104, the step height in the stride length of the elliptical path is synchronously adjusted. In other words, an adjustment of the step height automatically, and without additional user intervention, results in adjustment of the stride length, and vice versa. In one implementation, the synchronous adjustment is facilitated by a mechanical coupling of the footpad 30. In another implementation, the synchronous adjustment is facilitated by a controller which outputs control signals to concurrently or synchronously adjust both the step height and the stride length of the elliptical path being taken by foot pads 30.

FIGS. 3 and 4 illustrate exercise apparatus 220, an example implementation of exercise apparatus 20. Exercise apparatus 220 comprises frame 224, left side leg 226L, right side leg 226R (collectively referred to as side legs 226), left foot link 228L, right foot link 228R (collectively referred to as foot links 228), left footpad 230L, right footpad 230R (collectively referred to as foot pads 230), left crank 234L, right crank 234R (collectively referred to as cranks 234), resistance system 236, flexible member guides 238L, 238R (collectively referred to as guides 238), 240L, 240R (collectively referred to as guides to 240), flexible member guide 242, left stride height adjusting flexible member 244L, right stride height adjusting flexible member 244R (collectively referred to as flexible members 244), stride length adjusting flexible member 246, and adjustment synchronizer 250 comprising adjustment member 254, adjuster 258 and monitor 260. Frame 224 comprise a foundation or series of bars, brackets, rods or other structures joined to one another to support the remaining components of exercise apparatus 220 upon an underlying surface. Although illustrated with the particular configuration, frame 224 may have other sizes, shapes and configurations as well.

Legs 226 comprise structures pivotally suspended and supported by frame 224. In the example illustrated, leg 226L comprises a flexible member guide 262L while leg 226R comprises a flexible member guide 262R. Guides 262L and 262R (collectively referred to as guides 262) guide movement of flexible member 246 and couple the rotational or pivotal movement of legs 226 with the translation or movement of flexible member 246. In the example illustrated, each of guides 262 comprises a pulley pivotally supported by frame 224 so as to rotate with the remainder of the

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respective leg 226. In other implementations, guides 262 comprise a pie-shaped or wedge-shaped member having a surface or groove guiding and/or gripping flexible member 246. In some implementations in which flexible member 246 comprises a toothed belt, guides 262 comprise corresponding teeth or corresponding openings. Each of legs 226 has an end portion pivotally coupled to a respective one of foot links 228.

Foot links 228 extend from legs 226 and support footpads 230. Footpad 230 comprise platforms, paddles or pedals upon which a person exercising places his or her feet during exercise and against which a person applies force to move foot pads 230 along an elliptical path. Foot pads 230 may have a variety of different sizes, shapes and configurations. Foot pads 230 are linked to one another to move in unison along the same elliptical path (paths of the same shape), wherein the paths taken by foot pads 230 are of the same elliptical shape, but are out of phase with one another. In the example illustrated, foot pads 230 move through elliptical paths of the same shape, but which are 180° out of phase with respect to one another. For example, when foot pad 230L is at the uppermost position along the shape of the elliptical path, foot pad 230R is at the lowermost position along the shape of the elliptical path. Similarly, when foot pad 230L is at the forward-most position along the shape of the elliptical path, foot pad 230R is at the rearward-most position along the shape of the elliptical path.

Cranks 234 cooperate to synchronize movement of footpads 230 and to apply a resistance to such movement. Cranks 234 each comprise a crank arm 264 that rotates about an axis 274 which eccentrically support flexible member crank guides 266L, 266R (collectively referred to as crank guides 266) and 268L and 268R (collectively referred to as crank guides 268) relative to axis 274. As shown by FIG. 3, cranks 234 are connected to so as to rotate with and extend from a shared central disc 270. Each of the arms 264 forming cranks 234 or angularly offset 180° with respect to one another. As a result, footpads 230 move through paths having the same elliptical shape, but wherein the elliptical shaped paths are 180 degrees out of phase with respect to one another.

Flexible member crank guides 266 comprise members that are connected to arms 264 and carried by arms 264 so as to rotate about axis 274 and about which flexible members 244 wrap so as to transmit force to crank guides 266 and ultimately to support 264 of crank 234. In the example illustrated, flexible member crank guides 266 are pivotally or rotationally coupled to the respective arms 264 so as to rotate about or pivot about the respective axes 276 which are radially spaced from axis 274.

Flexible member crank guides 268 comprise members that are connected to and carried by arms 264 also rotate about axis 274 and about which stride length adjusting flexible member 246 wrap so as to also transmit force to crank guides 268 and ultimately to cranks 234. Flexible member crank guides 268 are pivotally or rotationally coupled to their respective arms 264 so as to rotate about or pivot the respective axes 276 which are radially spaced from axis 274. In the example illustrated, each flexible member crank guides 266 and 268 comprises a pulley. In other embodiments, each flexible member crank guide 266 and 268 may alternatively comprise a spool or disc against which a flexible member moves or slides without rotation of the flexible member crank guide 266.

Resistance system 236 applies additional resistance to the rotation of crank 234. In the particular example illustrated, resistance system 236 provides a selectively adjustable

incremental resistance to the rotation of cranks **234**. Resistance system **236** comprises resistance source **271** and belt **272**. Resistance source **271** comprises a mechanism configured to rotate against a selectively adjustable resistance. In one embodiment, resistance source **271** comprises a metal plate and one or more magnets forming an Eddy brake. In one embodiment, the one or more magnets comprise electromagnets, allowing the strength of the magnetic force to be selectively adjusted to control and vary the resistance applied against the rotation of cranks **234**. In another embodiment, resistance source **271** may comprise an electric generator. In still another embodiment, resistance source **271** may comprise two surfaces in frictional contact with one another to apply a frictional resistance against rotation of cranks **234**. In another embodiment, air brakes may be utilized. In still other embodiments, other brakes or resistance mechanisms may be utilized.

Belt **272** operably couples resistance source **271** to disk **270** and cranks **234**. In one implementation, belt **272** is entrained about a pulley which rotates with resistance source **271** and a corresponding pulley associated with disk **270**. In other implementations, chain sprocket arrangements or gear trains operably couple rotation of cranks **234** and rotation of corresponding components of resistance source **271**. In still other implementations, resistance system **271** may comprise other braking or resistance sources or may be omitted.

Flexible member guides **238** and flexible member guides **240** comprise structures having surfaces that guide movement of flexible members **244**. In one implementation, guides **238** and **240** comprise rotatable pulleys. In another implementation, guides **238**, **240** comprise curved channels, grooves or other stationary structures are surfaces against which flexible members **244** slide or move.

Stride height adjusting flexible members **244** comprise an elongated flexible or bendable members such as cables, bands, wires, ropes, belts, cords, strings, straps, chains and the like that extend between adjustment member **254** and foot links **228**. Flexible member **244L** has a first end portion secured or connected to adjustment member **254** and a second end portion secured or connected to foot link **228L**. Flexible member **244L** has central portions that wrap about an upwardly facing side of flexible member crank guide **266L**, a downwardly facing side of guide **238L** and an upwardly facing side of guide **240L**. Similarly, flexible member **244R** has a first end portion secured or connected to adjustment member **254** and a second end portion secured or connected to foot link **228R**. Flexible member **244R** has central portions that wrap about an upwardly facing side of flexible member crank guide **266R**, a downwardly facing side of guide **238R** and an upwardly facing side of guide **240R**. Stride height adjusting flexible members **244** link and control an extent to which foot links **228** and their respective footpads **230** pivot and move upwardly and downwardly.

Stride length adjusting flexible member **246** comprises an elongated flexible or bendable member such as a cable, band, wire, rope, belt, cord, string, strap, chain and the like that has a first end portion connected to adjustment member **254** on one side of crank **234** and a second end portion connected to adjustment member **254** on the other side of crank **234**. Stride length adjusting flexible member **246** has central portions that wrap partially about or against a downwardly facing surface of flexible member crank guide **268L**, a rear facing side or surface of guide **262L**, a front facing side or surface of guide **242**, a rear facing side or surface of guide **262R** and a downward facing side or surface of crank guide **268R**. Stride length adjusting flexible member **246**

links and controls an extent to which arms **226** and their respective footpads **230** pivot and move forwardly and rearwardly.

Adjustment synchronizer **250** simultaneously or concurrently adjusts the step height and the stride length of the elliptical path being taken by foot pads **30** in a synchronous manner in response to a single adjustment request. As noted above, adjustment synchronizer **250** comprises adjustment member **254**, adjuster **258** and monitor **260**. Adjustment member **254** comprises a structure forming a pair of elongate bars **280** and extensions **282L**, **282R**, **284L**, **284R**. Bars **280** are connected to one another and are pivotally supported by frame **224** so as to pivot in unison together about an axis. In the example illustrated, bars **280** sandwich support **264** and rotate about the rotational axis **274** of support **264**. In other implementations, bars **280** rotate or pivot about an axis different than that of crank **234** or support **264**.

Extensions **282L**, **282R**, **284L**, **284R** project from opposite sides of bars **280** and provide mounting points or connection points for ends or end portions of flexible members **244** and flexible member **246**. In the example illustrated, extensions **282L** and **282R** extend in opposite directions from opposite sides of bars **280** and are connected to end portions of flexible members **244L** and **244R**, respectively. Similarly, extensions **284L** and **284R** extend in opposite directions from opposite sides of bars **280** at an opposite end of bars **280** as extensions **282**, wherein extensions **284L** and **284R** are connected to end portions of flexible member **246**. Although bars **280** are illustrated as extending on opposite sides of support **264** of crank **234**, in other implementations, bars **280** comprise a single bar on one side of crank **234**. Although adjustment member **254** has a general shape of a pump of a railroad hand car, in other implementations, adjustment member **254** has other shapes and configurations, wherein adjustment member **254** provides first laterally spaced mounting points at a first end for connecting to ends of flexible members **244** and second laterally spaced mounting points at a second opposite end for the ends of flexible member **246**.

Overall, extension **282L**, flexible member **244L** and crank guide **266L** form a left stride height mechanism, wherein the stride height of the elliptical path taken by what pad **230L** is controlled by the positioning of extension **282L**, flexible member **244L** and crank guide **266L**. Extension **282R**, flexible member **244R** and crank guide **266R** form a right stride height mechanism, wherein the stride height of the elliptical path taken by foot pad **230R** is controlled by the positioning of extension **282R**, flexible member **244R** and crank guide **266R**. Extensions **284**, stride length adjusting flexible member **246** and crank guides **268** form a stride length mechanism, wherein the stride length of the elliptical path taken each of footpads **230** is controlled by the positioning of flexible member **246** and crank guides **268**.

Adjuster **258** (shown in FIG. 4) comprises a mechanism to incrementally pivot adjustment member **254** between various angular positions. Adjuster **258** concurrently adjusts the positioning of extensions **282**, **284** to concurrently adjust both the step height and the stride length. In the example illustrated, adjuster **258** comprises an electrically powered motor **286** that rotationally drives screw or worm screw **288** which passes through a threaded member or nut **290** pivotably coupled to bars **280** for pivotal movement about an axis perpendicular to the axis of worm screw **288** but secured against rotation about the axis of worm screw **288**. Rotation of worm screw **288** moves adjustment member **254** along the axis of worm screw **288** to pivot adjustment member **254** about its rotational axis.

In other implementations, adjuster **258** comprises other actuators. For example, in one implementation, adjuster **258** comprises a hydraulic or pneumatic cylinder-piston assembly, wherein one end of the cylinder piston assembly is pivotally supported by frame **224** and the other end of the assembly is pivotally connected to adjustment member **254**. In yet other implementations, adjuster **258** may comprise a motor other rotational actuator coupled between frame **224** and adjustment member **254**.

Monitor **260** serves as an input **290** and a controller **292** (schematically shown FIG. 3). In the example illustrated, input **290** comprises a touch screen having appropriate graphical user interfaces or icons to facilitate input from the person exercising. In other implementations, input **290** comprises one or more pushbuttons, slider bars, knobs, dials, a touchpad, keyboard, a microphone with associated speech recognition hardware and software or other currently available or future developed input devices. Input **290** facilitates input of a selected adjustment for the elliptical path taken by footpads **230**.

Controller **292** comprises a processor and associated non-transitory computer-readable medium which outputs control signals for adjuster **258** in response to inputted or programmed adjustment selections for the elliptical path of footpads **230**. In one implementation, apparatus **220** operates in a mode in which the person exercising enters a selected elliptical path shape or a selected combination of step height and stride length for a desired elliptical path. Based on such input, controller **292** outputs control signals to motor **286** so as to selectively drive or rotate worm screw **288** to position or reposition extensions **282**, **284** and the end portions of flexible elements **244** and **246** so as to attain the selected elliptical path shape or selected combination of step height and stride length. In yet another implementation, input **290** receives a selected exercise program or routine having preprogrammed or predetermined elliptical path shapes or step heights/stride lengths which are to be implemented at particular points in time during an exercise program. At the preprogrammed or predefined times, controller **292** automatically outputs control signals to motor **286** to selectively drive or rotate worm screw **288** to a selected position so as to pivot or rotate adjustment member **254** to particular angular orientation, wherein the ends of flexible members **244** and **246** are also positioned so as to partially wrap about guides **266**, **268** by predetermined extents to achieve the selected elliptical path shape or step height/stride length at the appropriate times.

FIGS. 5-6 illustrate operation of adjustment synchronizer **250**. FIGS. 5A-5C illustrate adjustment synchronizer **250** actuated to a first state in which the step height is minimized and the stride length is maximized. In response to signals generated by controller **292** as a result of either receiving a command or selection through input **290** or being directed by an exercise program stored in a non-transitory memory associated with controller **292**, controller **292** generates control signals causing motor **286** to rotate screw **288** to rotate adjustment member **254** about axis **274** to the near vertical orientation shown in FIGS. 5A and 5B. As a result, the ends or end portions of flexible members **244** and flexible member **246** are repositioned as shown. The repositioning of the end portions of flexible members **244** and flexible member **246** which are connected to extensions **282**, **284** adjusts and controls a degree to which intermediate portions of flexible members **244** and flexible member **246** wrap about crank guides **266** and **268**, respectively, such that each of footpads **230** follows the elliptical path **297** shown in FIG. 5C.

FIGS. 6A-6C illustrate adjustment synchronizer **250** actuated to a second state in which the step height is maximized and the stride length is minimized. In response to signals generated by controller **292** as a result of either receiving a command or selection through input **290** or being directed by an exercise program stored in a non-transitory memory associated with controller **292**, controller **292** generates control signals causing motor **286** to rotate screw **288** to rotate adjustment member **254** about axis **274** to the near horizontal orientation shown in FIGS. 6A and 6B. As a result, the ends or end portions of flexible members **244** and flexible member **246** are repositioned as shown. The repositioning of the end portions of flexible members **244** and flexible member **246** which are connected to extensions **282**, **284** adjusts and controls a degree to which intermediate portions of flexible members **244** and flexible member **246** wrap about crank guides **266** and **268**, respectively, such that each of footpads **230** follows the elliptical path **298** shown in FIG. 6C.

Although FIGS. 5A-5C and 6A-6C illustrate example extreme positions of adjustment member **254**, in other implementations, adjuster member **254** is actuable to other different greater or lesser extreme positions. While FIGS. 5A-5C and 6A-6C illustrate such extreme positions, adjuster **258** is configured to also position adjustment member **254** at any one of a variety of different angular orientations between the two example extreme angular orientations or positions as illustrated. In such alternative angular positions of adjustment member **254**, the step height and the stride length of the elliptical path also have distances or values in between the maximums and minimums illustrated in FIGS. 5C and 6C.

FIG. 7 illustrates exercise apparatus **320**, another example implementation of exercise apparatus **20**. Exercise apparatus **320** is similar to exercise apparatus **220** except that footpads **230** are not cantilevered, but are positioned right above flexible members **244**. Those components of exercise apparatus **320** which correspond to components of exercise apparatus **20** are numbered similarly.

As with exercise apparatus **220**, exercise apparatus **320** comprises resistance source **236**, adjuster **258** and monitor **260** shown and described above with respect to FIGS. 4 and 5. As shown by FIG. 7, exercise apparatus **320** comprises frame **324** in place of frame **224** and includes rear mounts **380**. For ease of illustration, those portions a frame **324** that support crank **234** as well as resistance system **236** are not shown. Flexible members **244** have end portions that are connected at rear mounts **380** rather than being connected directly to foot links **228**. Footpads **230** slide or glide upon or along flexible members **244**. In the example illustrated, each of footpads **230** comprises one or more rollers or pulleys **381** to facilitate such sliding or gliding movement of footpads **230** upon a top of flexible members **244**.

As with exercise apparatus of **220**, exercise apparatus **320** automatically synchronizes the adjustment of both the step height and the stride length of the elliptical path being taken by footpads **230**. Rotation of adjustment member **254** concurrently repositions the ends of flexible members **244** and flexible member **246** to concurrently adjust step height and stride length, respectively. As a result, exercise apparatus **320** facilitates a greater degree of control of a proportional relationship between the step height and the stride length. In other words, the proportional relationship between the step height and the stride length may be maintained within certain predefined relationships predetermined as being more natural, predetermined as being best suited for a particular size or other characteristic of the person exercising or predetermined as being best suited for a particular

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fitness objective. Because adjustment synchronizer 250 facilitates a single input to adjust the synchronizer 250, adjustment by a person exercising may be performed through a single input to the exercise apparatus 320, providing ease-of-use and allowing the person exercising to focus on the exercise being performed. The user friendly single input allows even a first time user to quickly understand and operate the exercise apparatus 320 without confusion or trial and error.

FIGS. 8A-8D illustrate how exercise apparatus 320 utilizes the controlled and synchronized adjustment of the step height and stride length to facilitate a closer level of footpad spacing for footpads 330. Footpads 330 are similar to footpads 230 except that footpads 330 comprise a particular example implementation of footpads 230 in which footpads 330 comprise toe caps 331. As shown by FIGS. 8A and 8B, footpads 330 move through parallel elliptical paths, wherein the parallel elliptical paths move in vertical planes that are closer to one another and closer to the centerline 352 than conventional elliptical exercise devices. In such an implementation, inner portions footpads 330 vertically overlap one another along the centerline 352, wherein footpad 330L overlaps the underlying footpad 330R and vice versa. The coordinated or synchronized adjustment of the step height and stride length helps to ensure that the actual positions of the footpads 330 do not meet at the overlapping points along the centerline 352 and do not collide.

FIGS. 9-11 illustrate exercise apparatus 420, another example implementation of exercise apparatus 20. As with exercise apparatus 20, exercise apparatus 420 provides concurrent or synchronized adjustment of both step height and stride length. Exercise apparatus 420 comprises frame 424 (partially shown in broken lines), arms 426L, 426R (collectively referred to as arms 426), foot links 428L, 428R (collectively referred to as foot links 428), support links 429, footpads 430L, 430R (collectively referred to as foot pads 430), left crank 434L and right crank 434R (collectively referred to as cranks 434), resistance system 436, flexible member guides 438L, 438R (collectively referred to as flexible member guides 438), flexible members 444L, 444R (collectively referred to as flexible members 444), stride length adjusting links 446L, 446R (collectively referred to as links 446), and adjustment synchronizer 450 comprising adjustment member 454, adjuster 458, link support guides 462L, 462R (collectively referred to as guides 462), link supports 464L, 464R (collectively referred to as supports 464), support biases 467L, 467R (collectively referred to as biases 467), synchronization coupler 468, spool 470, and support adjustment flexible members 472L, 472R (collectively referred to as flexible members 472).

Frame 424 supports the remaining components or elements of exercise apparatus 420 upon an underlying terrain or support surface. Frame 424 comprises base 474, uprights 475 and front center post 478. Base 474 extends along the floor or other underlying supporting surface. Uprights 475 extend upwardly from base 474 and pivotably support arms 426. Uprights 475 further pivotably support guides 462. Center post 478 extends upwardly from base 424 and supports crank 434, resistance system 436, guides 438, adjustment member 454 and adjuster 458. In other implementations, frame 424 may have other configurations.

Arms 426 comprise structures pivotably supported by uprights 475 for rotation about axis 476. Each of arms 426 has a rearward extending portion 482 and a forwardly extending portion 484. Rearward extending portion 482 extends rearward from axis 476 and is pivotably coupled to a respective one of foot links 428. Forward extending

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portion 484 extends forward from axis 476 and has an end connected to a respective one of flexible members 444.

Foot links 428 extend between arms 426 and footpads 430. Each of foot links 428 has an upper end pivotally connected to rearward extending portion 482 of the respective arm 426 and a lower end supporting a respective one of footpads 430. Each of foot links 428 is further controlled by link 429 which has a first end pivotally secured to the respective one of foot links 428 and a second end pivotally secured to guide 462. Links 429 connect foot links 428 to guide 462 via the pivoting member that holds guide 462.

In one implementation, each of links 429 is releasably connectable to the associated link 428 at one of plurality of available vertically spaced mounting locations. For example, in one implementation, each foot link 428 comprises a forwardly extending plate or year having column of vertically spaced apertures by which the end portion of link 429 may be pinned or otherwise mounted. Selectively repositioning the end of link 429 in one of the various vertically spaced attachment or mounting points on the associated foot links 428 allows a person to adjust the range of stride length such that the minimum or maximum of the stride length would be uniformly larger or smaller. In one implementation, each of links 429 may alternatively have a resiliently extendable/compressible length to provide cushioning. For example, one implementation, each of links 429 may comprise a shock-absorber like hydraulic or pneumatic cylinder-piston shock assembly. In another implementation, each of links 429 may comprise a resiliently compressible leaf spring, an elastomeric rubber-like link or other elongated member having a resiliently adjustable length.

Footpads 430 are supported at lower end of foot links 428. Footpads 430 comprise platforms upon which a person exercising places his or her feet during exercise, and against which a person applies force to move foot pads 430 along an elliptical path. Foot pads 430 are linked to one another to move in unison along the same elliptical path (paths of the same shape), wherein the paths taken by foot pads 430 are of the same elliptical shape, but are out of phase with one another. In the example illustrated, foot pads 430 move through elliptical paths of the same shape, but which are 180° out of phase with respect to one another. For example, when foot pad 430L is at the uppermost position along the shape of the elliptical path, foot pad 430R is at the lowermost position along the shape of the elliptical path. Further, when foot pad 430L is at the forward-most position along the shape of the elliptical path, foot pad 430R is at the rearward-most position along the shape of the elliptical path. As discussed above with respect to FIGS. 8A-8D, footpads 430 are supported and guided so as to move through parallel elliptical paths within parallel vertical planes wherein each footpads 430 overlaps a longitudinal centerline of exercise apparatus 420 and/or vertically overlaps the other of the footpads at some point during its continuous looping movement (multiple continuous rotations of 360 degrees of cranks 434 about their shared or common axis). In other implementations, the paths the footpads 430 are not parallel. In one implementation, the paths of footpads 430 have several degrees of convergence at the front of the stride, wherein the footpads still overlap.

Cranks 434 share a common axle and/or rotate about a common central axis 504 (shown in FIG. 10). Left crank 434L comprises an arm 464 (shown in FIG. 10) which eccentrically and rotationally supports left flexible element crank guide 466L. Right crank 434R comprises disk 470 which eccentrically and rotationally supports right flexible element crank guide 466R. Crank guides 466 function

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similarly to crank guides 266. Similar to crank guide 266, left flexible element crank guide 466L and right flexible element crank guide 466R (collectively referred to as crank guides 466) are angularly offset from one another by 180° with respect to axis 504. As a result, footpads 430 move through paths having the same elliptical shape, but wherein the elliptical shaped paths are 180 degrees out of phase with respect to one another.

Resistance system 436 is similar to resistance system 236 described above. As shown in FIG. 9, resistance system 436 comprises a resistance source 488 which is operably coupled to cranks 434 to resist rotation of cranks 434. In the example illustrated, resistance source 488 is operably coupled to cranks 434 by flexible member 490, stacked pulleys 492, flexible member 494 and resistance source pulley 496. Flexible member 490 wraps about disc 470 of crank 434R and about a first smaller diameter pulley of stacked pulleys 492 which are supported by center post 478. Flexible member 494 wraps about the larger diameter pulley of stacked pulleys 492 and the resistance source pulley 496. In the example illustrated in which resistance system 436 utilizes pulleys, such as pulleys 492 and 496, flexible members 490 and 494 comprise belts. In other implementations, such pulleys may be replaced with sprockets, wherein flexible members 490 and 494 comprise chains. In yet other implementations, cranks 434 are operably coupled to resistance source 488 by a gear train or other transmission mechanism.

Resistance source 488 is similar to resistance source 270 described above. In one embodiment, resistance source 488 comprises a metal plate and one or more magnets forming an Eddy brake. In one embodiment, the one or more magnets comprise electromagnets, wherein the strength of the magnetic force to be selectively adjusted to control and vary the resistance applied against the rotation of cranks 434. In another embodiment, resistance source 488 may comprise an electric generator. In still another embodiment, resistance source 488 may comprise two surfaces in frictional contact with one another to apply a frictional resistance against rotation of crank 434. In another embodiment, air brakes may be utilized. In still other embodiments, other brakes or resistance mechanisms may be utilized.

Flexible member guides 438 comprise structures or members that guide movement of flexible members 444 between crank guides 466 of cranks 434 and forward extending portions 484 of arms 426. In the example illustrated, guides 438 comprise idler pulleys rotationally supported by center post 478. In other implementations, guides 438 may comprise stationary arcuate structures that guide sliding movement of flexible members 444.

Flexible members 444 comprise elongated flexible or bendable members such as cables, bands, wires, ropes, belts, cords, strings, straps, chains and the like that extend between adjustment member 454 and arms 426. Flexible member 444L has a first end portion secured or connected to adjustment member 454 and a second end portion secured or connected to forward extending portion 484 of arm 426L. Flexible member 444L has central portions that wrap about a downwardly facing side of flexible member crank guide 466L and a forwardly facing side of guide 438L. Similarly, flexible member 444R has a first end portion secured or connected to adjustment member 454 and a second end portion secured or connected to forward extending portion 484 of arm 426R. Flexible member 444R has central portions that wrap about a downwardly facing side of flexible member crank guide 466R and a forwardly facing side of guide 438R.

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Stride length adjusting links 446 comprise elongate rods, bars or linkages having a first end portion 494 pivotably attached to a respective one of crank 434 and a second end portion 496 pivotably attached to a respective one of supports 464 for pivotal movement about an associated transverse axis 498. As will be described hereafter and illustrated in FIG. 9, the longitudinal spacing or distance *d* between axes 476 and 498 of the link 446 defines the stride length of the elliptical path being taken by footpads 430. Although the pivot axis of guide 462 at the frame is illustrated as collinear/common with the pivot axes 476 of arms 426, in exercise apparatus 420, the pivot axes of arms 426 do not have to be the same as the pivot axis (at the frame) of guide 462. In each of such implementations, the pivot axis 476 of arms 426 is tangent to a circumference of spool 470. Such a configuration reduces or minimizes the extent to which supports 464 (described hereafter) move along guides 462 as footpads 430 traverse their respective paths. In other implementations, the pivot axis 476 of arms 426 is offset (non-tangent) with respect to the circumference of spool 470.

Adjustment synchronizer 450 simultaneously or concurrently adjusts the step height and the stride length of the elliptical path being taken by foot pads 430 in a synchronous manner in response to a single adjustment request. As noted above, adjustment synchronizer 450 comprises adjustment member 454, adjuster 458, link support guides 462L, 462R (collectively referred to as guides 462), link supports 464L, 464R (collectively referred to as supports 464), support biases 467L, 467R (collectively referred to as biases 467), synchronization coupler 468, spool 470, support adjustment flexible members 472L, 472R (collectively referred to as flexible members 472), and monitor 260.

Adjustment member 454 comprises a structure forming a pair of elongate bars 500, extensions 502L, 502R and cam 503. Bars 500 are connected to one another and are pivotally supported by center post 487 of frame 424 so as to pivot in unison together about axis 504. In the example illustrated, bars 500 rotate about the rotational axis 504 of cranks 434. In other implementations, bars 500 rotate or pivot about an axis different than that of cranks 434.

Adjuster 458 comprises a mechanism to rotate adjustment member 454 through a range of less than 180° so as to adjust angular positioning of extensions 502 and the end points of flexible members 444 so as to adjust the step height of the elliptical paths being taken by footpads 430. In the example illustrated, adjuster 458 is similar to adjuster 258 described above. Adjuster 458 comprises an electrically powered motor 510 that rotationally drives screw or worm screw 512 which passes through a threaded member or nut that is pivotably coupled to bars 500 for pivotal movement about an axis perpendicular to the axis of worm screw 512 but secured against rotation about the axis of worm screw 512. Rotation of worm screw 512 moves an end portion of adjustment member 454 along the axis of worm screw 512 to pivot adjustment member 454 about its axis 504.

Extensions 502L, 502R project from opposite sides of bars 500 and provide mounting points or connection points for ends or end portions of flexible members 444. In the example illustrated, extensions 502L and 502R extend in opposite directions from opposite sides of bars 500 and are connected to end portions of flexible members 444L and 444R, respectively. In other implementations, adjustment member 454 has other shapes and configurations, wherein adjustment member 454 provides laterally spaced mounting points at one end on one side of axis 504 for connecting to ends of flexible members 444.



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Cam 503 comprises a structure which rotates with bars 500 about axis 504 and provides a mounting surface and guide for synchronization coupler 468. In the example illustrated in which synchronization coupler 468 comprises a strap or belt, cam 503 comprises a pie-shaped wedge having an outer curved surface against which synchronization coupler 468 wraps or from which coupler 468 unwraps as a result of rotation of member 454. Although cam 503 is illustrated as radial or arcuate, in other implementations, cam 503 may have other shapes other than a strict radius to allow variation of the ratio of vertical to horizontal rate of change in the stride.

Overall, extension 502L, flexible member 444L and crank guide 466L form a left stride height mechanism, wherein the stride height of the elliptical path taken by foot pad 430L is controlled by the positioning of extension 502L which controls the degree to which flexible member 444L wraps about crank guide 466L. Extension 502R, flexible member 444R and crank guide 466R form a right stride height mechanism, wherein the stride height of the elliptical path taken by foot pad 430R is controlled by the positioning of extension 502R which controls the degree to which flexible member 444R wraps about crank guide 466R. Cranks 434 and stride length adjusting links 446 form a stride length mechanism, wherein the stride length of the elliptical path for each of footpads 430 is controlled by the positioning of the pivot axis 498 of each of links 446 relative to axis 476 of arms 426.

Link supports 464 movably support the upper ends of links 446 to facilitate controlled repositioning of the pivot axis 498 of such links 446 relative to axis 476. In the example illustrated, link supports 464 are slidably supported along guides 462 for linear sliding movement in fore and aft directions. Link supports 464 are resiliently biased in one direction by support biases 467. In the example illustrated, support biases 467 comprise gas cylinder-piston assemblies having one end mounted or secured to link support guides 462 and an opposite end secured to the respective one of supports 464. In the example illustrated, support biases 467 resiliently bias supports 464 in a forward direction. In other implementations, support biases 467 may comprise other biasing mechanisms such as compression springs or other types of springs depending upon the mounting arrangement.

Synchronization coupler 468, spool 470, and support adjustment flexible members 472L, 472R (collectively referred to as flexible members 472) cooperate to mechanically link the rotational adjustment of adjustment member 454 which adjusts step height to the movement of supports 464 and pivot axis 498 of links 446. In the example illustrated, synchronization coupler 468 comprises a flexible member such as a strap, web, cord, cable, band or belt having a first end portion fixed or secured to cam 503 of adjustment member 454 and a second opposite end portion fixed or secured to spool 470.

Spool 470 comprises a cylindrical member rotatably supported by frame 424 for rotation about an axis. In one implementation, spool 470 rotates about axis 480, the pivot axis of arms 426. In another implementation, spool 470 rotates about a different axis. As spool 470 is rotated, coupler 468 wraps about or unwraps from the spool 470 while flexible members 472 unwraps from or wrap about a 470, respectively.

Flexible members 472 comprise a strap, web, cord, rope, cable, band or belt having a first end portion fixed or secured to spool 470 and a second opposite end portion fixed or secured to a respective one of supports 464. In the example illustrated, flexible members 472 are secured to spool 470 so

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as to wind about spool 470 in a first rotational direction while coupler 468 is secured to spool 470 so as to wind about spool 470 in a second opposite rotational direction. For example, when coupler 468 is being wound about spool 470, flexible members 472 are being unwound from spool 470, and vice versa.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associate with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 to rotate adjustment member 454 about axis 504 which adjusts the positioning of the endpoints of flexible members 444. This repositioning of the endpoints of flexible members 444 changes the degree to which flexible members 444 wrap about crank guides 466 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 causes end portions of coupler 468 to wind or unwind relative to cam 503 and to rotate spool 470. Rotation of spool 470 winds or unwinds flexible members 472 so as to either move supports 464 and pivot axis 498 rearwardly against the bias of biases 467 or to allow the bias of biases 467 to move supports 464 and pivot axis 498 forwardly. Movement of pivot axis 498 relative to the rotational axis 476 of arms 426 adjusts the stride length of the elliptical path being taken by footpads 430.

Although the mechanical coupling of the movement for rotation of adjustment member 454 and the movement of the pivot axis of stride length adjusting links 446 is illustrated as being carried out by coupler 468 in the form of a flexible member, spool 470 and flexible members 472 which move sliding supports 464 against the bias, in other implementations, coupler 468 may comprise a gear train, mechanical link pivot connections or other force transmitting members. In yet other implementations, in lieu of sliding supports 464 to reposition pivot axis 498, the locations at which links 446 are pivotably coupled to arms 426 may alternatively be achieved by pivoting the location of the pivot axis 498 or by moving the location of the pivot axis 498 along a rack and pinion arrangement.

FIGS. 12-15 illustrate adjustment member 454 and supports 464 in a first state during which supports 464 and the associated pivot axis 498 of links 446 are at an intermediate position along guide 462 and movement of one of footpads 430 along an elliptical path corresponding to the step height and stride length dictated by the positioning of adjustment member 454 and supports 464. FIG. 16 illustrates adjustment member 454 and supports 464 in a second example state after being synchronously or concurrently repositioned, from one position to one another by adjuster 458. In the second example state illustrated, adjustment member 454 has been rotated clockwise and downward to change the degree to which flexible member 444 wraps against and about crank guide 466 so as to (reduce) adjust the step height of the elliptical path that footpads 430 move along. Rotation of adjustment member 454 further results in rotation of cam 503 which pulls coupler 468 to rotate spool 470 so as to wind flexible member 472 and move supports 464 and the associated pivot axis 498 rearwardly along guide 462 against the bias of bias 467. As a result, the stride length of elliptical path taken by foot pads 430 is concurrently changed (increased). Consequently, the shape of the elliptical path taken by footpads 430 changes from the elliptical path 520 shown in FIGS. 13-15 to the elliptical path 522 shown in FIG. 16 (and shown in broken lines in FIG. 12).

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FIG. 17 illustrates adjustment member 454 and supports 464 in a third example state after being synchronously or concurrently repositioned, from one position to one another by adjuster 458. In the third example state illustrated, adjustment member 454 has been rotated counterclockwise from the position shown in FIG. 13 and downward to change the degree to which flexible member 444 wraps against and about crank guide 466R so as to adjust (increase) the step height of elliptical path that footpads 430 move along. Rotation of adjustment member 454 further results in rotation of cam 503 which unwinds coupler 468 which unwinds flexible member 472 in response to force is applied by biases 467 and moves support 464 and the associated pivot axis 498 forwardly along guide 462. As a result, the stride length of elliptical path taken by foot pads 430 is concurrently changed (decreased). Consequently, the shape of the elliptical path taken by footpads 430 changes from the elliptical path 520 shown in FIG. 12 to elliptical path 524 shown in FIG. 17 (and shown in broken lines in FIG. 12). As will be appreciated, adjuster 458, in response to control signals from controller 292, may selectively reposition adjustment member 454 at a multitude of different angular positions between the example extreme shown in FIGS. 16 and 17 which would also result in pivot axis 498 being selectively repositioned any corresponding multitude of different positions along guide 462 between the example extreme positions shown in FIGS. 16 and 17.

FIG. 18 illustrates exercise apparatus 620, another example implementation of exercise apparatus 20. Exercise apparatus 620 is similar to exercise apparatus 420 except that exercise apparatus 620 adjusts the step height in a fashion similar to the adjustment of the step height in exercise apparatus 220 and comprises adjustment synchronizer 650 in place of adjustment synchronizer 450. Those components of exercise apparatus 620 which correspond to components of exercise apparatus 220 or 420 are numbered similarly. Synchronizer 650 comprises supports 664L, 664R (collectively referred to as supports 664), gear 666, toothed belt 668, driveshaft 670 comprising gear 672 and pinion gears 674L, 674R (collectively referred to as pinion gear 674).

Supports 664 pivotably support end portions of links 446 for pivotal movement about pivot axis 498. Each of supports 664 is pivotally supported by a respective one of arms 426 about axes 665L and 665R. Each of supports 664 further comprises a rack gear 667L, 667R having teeth in meshing engagement with the teeth of a respective one of pinion gears 674.

Gear 666 comprises a gear coupled to adjustment member 454 so as to rotate in response to pivoting of adjustment member 454. In the example illustrated, gear 666 is fixed or joined to adjustment member 454 to rotate with the rotation of adjustment member 454 at a 1:1 ratio. In other implementations, gear 666 is operably coupled to adjustment member 454 by a gear train or other transmission so as to rotate with the rotation of adjustment member 454 at a predetermined ratio greater than or less than 1:1.

Toothed belt 668 wraps about gear 666 and gear 672 with its teeth intermeshed with the teeth of gear 666 and gear 672. Belt 668 transmits torque from gear 666 to driveshaft 670. In other implementations, torque or rotation may be transmitted from adjustment member 454 and driveshaft 670 by other transmission such as a chain and sprocket arrangement, a gear train or a belt and pulley arrangement.

Drive shaft 670 comprises a shaft rotatably supported by frame 424 independent of the rotation of arms 426 about axis 476. In the example illustrated, driveshaft 670 is also

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rotatably supported about axis 476. Driveshaft 670 carries gear 672 and pinions 674. Pinions 674L, 674R have teeth intermeshing with rack gears 667L, 667R, respectively.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associate with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 to rotate adjustment member 454 about axis 680 which adjusts the positioning of the endpoints of flexible members 244. This repositioning of the endpoints of flexible members 244 changes the degree to which flexible members 244 wrap about crank guides 266 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 causes gear 666 to also rotate. Rotation of gear 666 drives rotation of driveshaft 670 via a toothed belt 668 and gear 672. Rotation of driveshaft 670 drives rack gears 667 to pivot supports 664 about axes 665 to move pivot axis 498 relative to axis 476 of arms 426. As a result, rotation of adjustment member 454 adjusts the step height of the elliptical path taken by footpads 430 and concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIG. 19 illustrates exercise apparatus 720, another example implementation of exercise apparatus 20. Exercise apparatus 720 is similar to exercise apparatus 620 except that exercise apparatus 720 comprises adjustment synchronizer 750 in place of adjustment synchronizer 650. Those remaining components of exercise apparatus 720 are shown in FIG. 19 and numbered similarly as exercise apparatus 620 or are shown in FIG. 18. Although not shown FIG. 18, exercise apparatus 720 comprises frame 424, foot links 428, footpads 430L, 430R (collectively referred to as foot pads 430), crank 434, resistance system 436, and flexible member guides 438L, 438R (collectively referred to as flexible member guides 438).

Adjustment synchronizer 750 comprises adjustment member 454, supports 664L, 664R (collectively referred to as supports 664), driveshaft 770 comprising pinion gears 674L, 674R (collectively referred to as pinion gear 674), electric powered motor 766 and monitor 260. Adjustment member 454 and supports 664 are described above. Driveshaft 770 is similar to driveshaft 670 except that driveshaft 770 omits gear 672 as it is directly driven by motor 766. Motor 766, in response to control signals from controller 292 drives driveshaft 770 to drive pinion 674 which rotate against rack gears 667 to pivot supports 664 about axis 665 which moves pivot axis 498 of links 446 relative to axis 476 of arms 426 so as to adjust the stride length of the elliptical path taken by footpads 430 (shown in FIG. 18).

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 to rotate adjustment member 454 about axis 680 which adjusts the positioning of the endpoints of flexible members 244. This repositioning of the endpoints of flexible members 244 changes the degree to which flexible members 244 wrap about crank guides 266 and adjusts the step height of the elliptical path being taken by footpads 430. At the same time, motor 766, in response to control signals from controller 292, drives driveshaft 770 to drive pinion 674 which rotate against rack gears 667 to pivot supports 664 about axis 665 which moves pivot axis

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498 of links 446 relative to axis 476 of arms 426 so as to adjust the distance  $d$  separating axes 476 and 498 and so as to adjust the stride length of the elliptical path taken by footpads 430 (shown in FIG. 18). As a result, rotation of adjustment member 454 to adjust the step height of the elliptical path taken by footpads 430 concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIG. 20 illustrates exercise apparatus 820, another example implementation of exercise apparatus 20. Exercise apparatus 820 is similar to exercise apparatus 720 except that apparatus 720 replaces the lift actuator provided by motor 510 and screw 512 with flexible member 810 and return spring 812. Those remaining components of exercise apparatus to relate 20 are numbered similarly in FIG. 21 and/or are shown in the above figures. Flexible member 810 has a first end attached to a side of adjust member 545 on an opposite side of axis 680 as extensions 502L and 502R which are attached to flexible members 244. Flexible member 810 has a second end secured to driveshaft 770 which serves as a spool about which flexible member 810 winds and unwinds in response to being rotationally driven by motor 766. Spring 812 comprises a tension spring having one end mounted to adjust member 545 on an opposite side of axis 680 as flexible member 810 and has a second end secured to frame 424. Spring 812 applies a bias force to resolve a bias adjustment member 545 about axis 680.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 766 to rotate driveshaft 770. Rotation of driveshaft 770 winds or unwinds flexible member 810 to pivot adjust member 545 about axis 680 to reposition extensions 502 and the endpoints of flexible members 244 so as to adjust the degree to which flexible members 244 wrap about crank guides 266 and so as to adjust the step height of the elliptical path being taken by footpads 430. At the same time, rotation of driveshaft 770 drives pinions 674 which rotate against rack gears 667 to pivot supports 664 about axis 665 which moves pivot axis 498 of links 446 relative to axis 476 of arms 426 so as to adjust the distance  $d$  separating axes 476 and 498 and so as to adjust the stride length of the elliptical path taken by footpads 430 (shown in FIG. 18). As a result, rotation of adjustment member 454 adjusts the step height of the elliptical path taken by footpads 430 and concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIG. 21-23 illustrates exercise apparatus 920, another example implementation of exercise apparatus 20. Exercise apparatus to 920 this similar to exercise apparatus 620 described above except that exercise apparatus 920 utilizes an alternative mechanism for adjusting the positioning of pivot axis 498 of the upper end of links 446 relative to axis 476 of arms 426. Those remaining components of exercise apparatus 920 that correspond to components of exercise apparatus 620 are numbered similarly or are shown in FIG. 19. Although not shown in FIG. 21, exercise apparatus 920 comprises frame 424, footpads 430L, 430R (collectively referred to as foot pads 430), crank 434, resistance system 436, and flexible member guides 438L, 438R (collectively referred to as flexible member guides 438). In yet other implementations, the arrangement shown FIG. 22 is pro-

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vided as part of exercise apparatus 420 as an alternative for adjusting the position of pivot axis 498 relative to axis 476.

As shown by FIG. 22, exercise apparatus 920 comprises adjustment synchronizer 950 which adjusts, in a coordinated or synchronized manner, the step height and stride length of the elliptical path being taken by footpads 430. Synchronizer 950 comprises adjustment member 454, adjuster 458, monitor 260, gear 666, tooth belt 668 and driveshaft 670, each of which are described above with respect to FIG. 19. Synchronizer 950 further comprises slide rails 962L, 962R (collectively referred to as slide rails 962), link supports 964L, 964R, pinion gears 963L, 963R (collectively referred to as gears 963) and tooth belts 967L, 967R (collectively referred to as belts 967).

Slide rails 962 comprise rods, tubes, beams or other structures fixed to arms 426. Slide rails 962 extend forwardly of axis 476 and guide movement of link supports 964 in fore and aft directions. Slide rails 962 rotationally support pinion gears 963 at their outer foremost ends. Pinion gears 963 cooperate with pinion gears 674 to support a respective one of toothed belts 674.

Link supports 964 pivotally support the upper end of links 446 for pivotal movement about a respective axis 498. As shown by FIG. 22, in the example illustrated, the upper end of each of links 446 provided with a clevis 969 that pivotably secures the upper end of each of links 446 to support 964. As further shown by FIGS. 22 and 23, each of support 964 is clamped to the associated toothed belt 967 so as to move back and forth with the movement of the respective belt 967.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 (shown in FIG. 18) to rotate adjustment member 454 about axis 680 which adjusts the positioning of the endpoints of flexible members 244. This repositioning of the endpoints of flexible members 244 changes the degree to which flexible members 244 wrap about crank guides 266 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 causes gear 666 to also rotate. Rotation of gear 666 drives rotation of driveshaft 670 via a toothed belt 668 and gear 672. Rotation of driveshaft 670 drives rack gears pinion gears 674 which drive toothed belts 967. Movement of toothed belts 967 linearly translates supports 964 along slide rails 962 to reposition the pivot axes 498 of the upper ends of links 446 relative to pivot axis 476 of arms 426. As a result, rotation of adjustment member 454 adjusts the step height of the elliptical path taken by footpads 430 and concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIG. 23 illustrates pivoting of arm 426L and slide rail line 962L, link support 964L, pinion gear 963L and toothed belt 967L about axis 476 during reciprocation of arm 426L. As shown by FIG. 23, reciprocation of arm 426L results in slight movement of link support 964L through a stroke of arm 426L. The connection of link support 964L to either the top or the bottom of tooth belt 967L determines which end of the path will have a slight acceleration.

FIGS. 24 and 25 exercise apparatus 1020, another example implementation of exercise apparatus 20. Exercise apparatus 1020 is similar to exercise apparatus 920 except that exercise apparatus 1020 drives driveshaft 670 with a motor 1068 instead of utilizing gear 666, belt 668 and gear

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672. Similar to exercise apparatus 920, exercise apparatus 1020 omits adjuster 458 and instead utilizes flexible member 810 and spring 812 (shown in FIG. 21) to actuate or pivot adjustment member 545. In other implementations, the illustrated pinion gears 674, 963 and toothed belts 967 alternatively comprise belt and pulley arrangement or chain sprocket arrangements.

FIG. 26 illustrates exercise apparatus 1120, another example implementation of exercise apparatus 20. Exercise apparatus 1120 is similar to exercise apparatus 1520 described hereafter. Many of the components or elements of exercise apparatus 1120 correspond to components previously described above with respect to exercise apparatus 420. Exercise apparatus 1120 comprises link supports 1164L, 1164R, flexible member guides 1165, biases 1167L, 1166R (collectively referred to as biases 1167), and stride length adjusting flexible members 1168L, 1168R (collectively referred to as flexible members 1168). Those components of exercise apparatus 1120 which correspond to components of exercise apparatus 420 are numbered similarly in FIG. 26 or are shown in FIGS. 9-11.

Link supports 1164 pivotably support link supports 446 (described above) for pivotal movement about axes 498 each link support 1164 is itself pivotally supported by a respective one of arms 426 about a respective axis 1170. Pivoting of link supports 1164 about axis 1170 repositions the respective axis 498 relative to axis 476 of arms 426 to adjust the distance d (dR the right side and dL for the left side) between the respective axes 498 and axis 476 to adjust a stride length of the elliptical paths taken by the associated footpads 430. The distances dR and dL are equally and simultaneously adjusted through movement of flexible members 1165.

Flexible member guides 1165 comprise pulleys that guide and direct movement of flexible members 1168. Biases 1167 comprise mechanisms that resiliently biases link supports 1164 in one direction about axis 1170. In the example illustrated, biases 1167 comprise gas cylinders that resiliently bias and urge link supports 1164 in a forward direction. In other implementations, biases 1167 comprise compression springs. In yet other implementations, biases 1167 comprise other spring arrangements. For example, in one implementation, a torsion spring may be coupled between a respective one of link supports 1164 and a respective one of arms 426.

Flexible members 1168 comprise cords, cables, straps, belts, ropes or other flexible members. Flexible members 1168 operably couple adjustment member 454 and link supports 1164. Flexible members 1168 extend from adjustment member 454, through guides 1165 and into connection with link supports 1164. In the example illustrated, adjustment member 454 is connected to flexible members 444 on a first side of axis 504 and is pivotally connected to flexible members 1168 on a second side of axis 504.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 (shown in FIG. 18) to rotate adjustment member 454 about axis 504 which adjusts the positioning of the endpoints of flexible members 444. This repositioning of the endpoints of flexible members 244 changes the degree to which flexible members 444 wrap about crank guides 466 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 also moves the endpoints of flexible members 1168 which results in link

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supports 1164 either being pivoted against the bias of biases 1167 about axis 1170 or pivoted under the influence of biases 1167 about axis 1170. As a result, each of the axes 498 at the end of link supports 1164 is pivoted and moved relative to axis 476 of arms 262 to adjust a stride length of the elliptical path being taken by footpads 430. Thus, rotation of adjustment member 454 to adjust the step height of the elliptical path taken by footpads 430 concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIG. 27 illustrates exercise apparatus 1220, another example implementation of exercise apparatus 20. Exercise apparatus 1220 is similar to exercise apparatus 1120 except that exercise apparatus 1220 comprises support link guides 462, link supports 464 and biases 467 (described above with respect to exercise apparatus 420). Those remaining components of exercise apparatus 1220 which correspond to components of exercise apparatus 1120 and/or 420 are numbered similarly and are shown in FIGS. 9-11 and 26.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 to rotate adjustment member 454 about axis 504 which adjusts the positioning of the endpoints of flexible members 444. This repositioning of the endpoints of flexible members 444 changes the degree to which flexible members 444 wrap about crank guides 466 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 also moves the endpoints of flexible members 1168 which results in link supports 464 either being slid along the axes of guides 462 against the force of biases 1167 or under the influence of biases 1167. As a result, each of the axes 498 at the end of link supports 464 is linearly translated and moved relative to axis 476 of arms 262 to adjust a stride length of the elliptical path being taken by footpads 430. Thus, rotation of adjustment member 454 adjusts the step height of the elliptical path taken by footpads 430 and concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIG. 28 illustrates exercise apparatus 1320, another example implementation of exercise apparatus 20. Exercise apparatus 1320 is similar to exercise apparatus 1120 except that exercise apparatus 1320 comprises flexible member guides 1365, biases 1366 and flexible members 1368 in place of guides 1165, biases 1167 and flexible members 1168. Those remaining components of exercise apparatus 1320 which correspond to components of exercise apparatus 1120 and/or 420 are numbered similarly and are shown in FIGS. 9-11 and 26.

Flexible member guides 1365 comprise pulleys supported by frame 424 so as to guide movement of flexible members 1368. Flexible members 1368L, 1368R extend through and are guided by guides 1365. In the example illustrated, as shown in FIG. 30, the perimeter/circumference of guides 1365 are tangent to the pivot axis 476. Such a configuration reduces or minimizes the extent to which supports 464 (described hereafter) move along guides 462 as the footpads traverse their respective paths. In other implementations, the pivot axis 476 of arms 426 is offset (non-tangent) with respect to the circumference of guides 1365. Flexible members 1368 each have a first portion connected to adjustment member 454 on an opposite side of axes 504 as extensions 502 and a second end portion connected to a respective one

of link supports 1164. Biases 1366 resiliently bias link supports 1164 in one direction about axis 1170. In the example illustrated, biases 1366 comprise compression springs. In other implementations, biases 1366 comprise other biasing mechanism such as gas cylinders, torsion springs, tension springs and the like operably coupled between a

respective one of arms 426 and the link support 1164. In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associate with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 (shown in FIG. 9) to rotate adjustment member 454 about axis 504 which adjusts the positioning of the endpoints of flexible members 444. This repositioning of the endpoints of flexible members 444 changes the degree to which flexible members 444 wrap about crank guides 466 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 also moves the endpoints of flexible members 1368 which results in link supports 1164 being pivoted about axis 1170 against the force of biases 1366 or under the influence of biases 1366. As a result, each of the axes 498 at the end of link supports 1164 is moved relative to axis 476 of arms 262 to adjust a stride length of the elliptical path being taken by footpads 430. Thus, rotation of adjustment member 454 to adjust the step height of the elliptical path taken by footpads 430 concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIGS. 29 and 30 illustrate exercise apparatus 1420, another example implementation of exercise apparatus 20. Exercise apparatus 1420 is similar to exercise apparatus 1320 except that exercise apparatus 1420 comprises link support guides 462 and link supports 464 (described above with respect to exercise apparatus 420) in place of link supports 1368. Those remaining components of exercise apparatus 1420 which correspond to components of exercise apparatus 1320 and exercise apparatus 420 are numbered similarly and/or are illustrated above and FIGS. 9-11 and 26.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 (shown in FIG. 9) to rotate adjustment member 454 about axis 504 which adjusts the positioning of the endpoints of flexible members 444. This repositioning of the endpoints of flexible members 444 changes the degree to which flexible members 444 wrap about crank guides 466 and adjusts the step height of the elliptical path being taken by footpads 430. Rotation of adjustment member 454 by adjuster 458 also moves the endpoints of flexible members 1368 which results in link supports 464 either being slid along the axes of guides 462 against the force of biases 467 or under the influence of biases 467. As a result, each of the axes 498 at the end of link supports 464 is linearly translated and moved relative to axis 476 of arms 426 to adjust a stride length of the elliptical path being taken by footpads 430. Thus, rotation of adjustment member 454 to adjust the step height of the elliptical path taken by footpads 430 concurrently or synchronously adjusts the position of axes 498 so as to also adjust the stride length of the elliptical path taken by footpads 430.

FIGS. 31 and 32 illustrate exercise apparatus 1520, another example implementation of exercise apparatus 20.

Exercise apparatus 1520 is similar to exercise apparatus 220, exercise apparatus 420 and exercise apparatus 1320, incorporating a combination of the horizontal stride length adjustment mechanism of exercise apparatus 420 and 1320 and the vertical step height adjustment mechanism of exercise apparatus 220. As with exercise apparatus 420, exercise apparatus 1520 provides concurrent or synchronized adjustment of both step height and stride length. Exercise apparatus 1520 comprises frame 424 (partially shown in broken lines), legs 226, foot links 228, footpads 230, left crank 434L and right crank 434R (collectively referred to as cranks 434), resistance system 436, flexible member guides 438L, 438R (collectively referred to as flexible member guides 438), flexible members 1544L, 1544R (collectively referred to as flexible members 1544), flexible member guides 1546L, 1546R (collectively referred to as flexible member guides 1546), flexible element guides 1547L, 1547R (collectively referred to as flexible element guides 1547), stride length adjusting links 1164, biases 1366, adjustment member 454, adjuster 458, synchronization coupler 468, spool 470, and support adjustment flexible members 472. Each of such components is described above with respect to other exercise apparatuses but for flexible members 1544 and flexible element guides 1546, 1547. Flexible members 1544 are similar to flexible members 244 except that flexible members 1544 extend from their respective extensions 502 of adjustment member 454, about their respective crank guides 466, their respective flexible element guides 438, about their respective flexible element guides 1546, about their respective flexible element guides 1547 to securement point 1549 of frame 424.

Overall, extension 502L, flexible member 1544L and crank guide 466L form a left stride height mechanism, wherein the stride height of the elliptical path taken by foot pad 230L is controlled by the positioning of extension 502L which controls the degree to which flexible member 1544L wraps about crank guide 466L. Extension 502R, flexible member 444R and crank guide 466R form a right stride height mechanism, wherein the stride height of the elliptical path taken by foot pad 230R is controlled by the positioning of extension 502R which controls the degree to which flexible member 1544R wraps about crank guide 466R.

Link supports 1164 and stride length adjusting links 446 form a stride length mechanism, wherein the stride length of the elliptical path taken each of footpads 230 is controlled by the positioning of the pivot axis 498 of each of links 446 relative to axis 476 of arms 426.

In operation, in response to signals generated by controller 292 as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing motor 510 to rotate screw 512 to rotate adjustment member 454 about axis 504 which adjusts the positioning of the endpoints of flexible members 1544. This repositioning of the endpoints of flexible members 1544 changes the degree to which flexible members 1544 wrap about crank guides 466 and adjusts the step height of the elliptical path being taken by footpads 230. Rotation of adjustment member 454 by adjuster 458 causes end portions of coupler 468 to wind or unwind relative to cam 503 and to rotate spool 470. Rotation of spool 470 winds or unwinds flexible members 472 so as to either pivot supports 1164 about pivot axis 498 rearwardly against the bias of biases 1366 or to allow the bias of biases 1366 to pivot supports 1164 and pivot axis 498 forwardly.

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Movement of pivot axis **498** relative to the rotational axis **476** of arms **426** adjusts the stride length of the elliptical path being taken by footpads **230**.

Although the mechanical coupling of the movement for rotation of adjustment member **454** and the movement of the pivot axis of stride length adjusting links **446** is illustrated as being carried out by coupler **468** in the form of a flexible member, spool **470** and flexible members **472** which pivot supports **1164** against the bias, in other implementations, coupler **468** may comprise a gear train, mechanical link, pivot connections or other force transmitting members.

FIGS. **33** and **34** illustrate exercise apparatus **1620**, another example implementation of exercise apparatus **20**. As with exercise apparatus **420**, exercise apparatus **1620** provides concurrent or synchronized adjustment of both step height and stride length. Exercise apparatus **1620** comprises frame **424** (shown in FIG. **31**), legs **226**, foot links **228**, footpads **230** (shown in FIG. **31**), left crank **434L** and right crank **434R** (collectively referred to as cranks **434**), resistance system **436** (shown in FIG. **31**), stride length adjusting links **1164**, adjustment member **1654**, adjuster **458**, synchronization coupler **1668**, worm drive **1670**, worm drive trolleys **1672**, coupling links **1674** and levers **1676**. Those components which are the same as our which are functionally similar to previously described components are numbered similarly.

Adjustment member **1654** comprises a two-sided lever which, in the example illustrated, pivots about axis **274** of cranks **434**. A first side of lever **1654** is pivotally connected to adjuster **458** while a second opposite side of lever **1654**, on an opposite side of axis **274**, is pivotally connected to a corresponding flexible element **1644**. Flexible element **1644** extends from adjustment member **1654**, wraps partially about a corresponding crank guide **466** and about a corresponding guide **1546** prior to being connected to a corresponding one of foot links **228**. Adjustment member **1654** further comprises a toothed gear **1678** which rotates in unison with rotation of lever **1654** about axis **274**.

Synchronization coupler **1668** comprises a looped belt wrapping about toothed gear **1678** and about worm drive **1670**. In the example illustrated, coupler **1668** comprises a toothed belt meshed with the teeth of toothed gear **1678** intermeshed with teeth of pinion gear **1680** of worm drive **1670**. Worm drive **1670** comprises a helically threaded shaft having a central pinion gear **1680**. The helical threads of worm drive **1670** engage corresponding helical threads of worm drive trolleys **1672** which are guided by and slide along shaft **1684**. Links **1674** comprise rods or bars pivotably coupled to a corresponding one of trolleys **1672** and a second and pivotably secured to lever **1676** through a universal joint. Levers **1676** extend from supports **1164** and serve as a lever arm for pivoting supports **1164** about their respective axes **665** to reposition the respective pivot axes **498** relative to axis **476**.

In operation, in response to signals generated by controller **292** (shown in FIG. **4**) as a result of either receiving a command or selection through input **290** or being directed by an exercise program stored in a non-transitory memory associated with controller **292**, controller **292** generates control signals causing motor **510** to rotate screw **512** to rotate adjustment member **1654** about axis **274** which adjusts the positioning of the endpoints of flexible members **1644**. This repositioning of the endpoints of flexible members **1644** changes the degree to which flexible members **1644** wrap about crank guides **466** and adjusts the step height of the elliptical path being taken by footpads **230**. Rotation of adjustment member **1654** by adjuster **458** also

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rotates gear **1678** to drive coupler **1668** to rotate worm drive **1670**. Rotation of worm drive **1670** moves trolleys **1672** inwards or outwards. FIG. **33** illustrates trolleys **1672** at an inner most position while FIG. **34** illustrates trolleys **1672** at an outermost position. As shown by FIGS. **33** and **34**, inward or outward movement of trolleys **1672** causes links **1674** to interact upon lever **1676** so as to pivot supports **1164** about axes **665** as to reposition the pivot axes **498** of stride length adjusting links **446**. Thus, rotation of adjustment member **1654** to adjust the step height of the elliptical path taken by footpads **230** concurrently or synchronously adjusts the position of axes **498** so as to also adjust the stride length of the elliptical path taken by footpads **230**.

FIGS. **35** and **36** illustrate exercise apparatus **1720**, another example implementation of exercise apparatus **20**. As with exercise apparatus **420**, exercise apparatus **1720** provides concurrent or synchronized adjustment of both step height and stride length. Exercise apparatus **1720** comprises frame **424** (shown in FIG. **31**), legs **1726**, foot links **228**, footpads **230**, left crank **1734L** and right crank **1734R** (collectively referred to as cranks **1734**), resistance system **436** (shown in FIG. **31**), stride length adjusting links **446**, supports **1164**, adjustment member **1754**, links **1755L**, **1755R** (collectively referred to as links **1755**), foot link support **1756**, adjuster **1758**, worm drive **1770**, worm drive trolleys **1772**, coupling links **1774** and levers **1776**. Those components which are the same as our which are functionally similar to previously described components are numbered similarly.

Cranks **1734** are supported by frame **224** for rotation about axis **270**. Each of cranks **1734** comprises an arm **1764** having a first end rotating about axis **270** and a second end pivotably connected to a corresponding one of stride length adjusting links **446**. Arms **1764** are offset from one another by 180° about axis **270**.

Supports **1164**, adjustment member **1754** and lever **1776** are provided by a three legged member rotationally coupled to a corresponding one of legs **1726** so as to rotate or pivot about a corresponding axis **665**. Supports **1164** extend from axis **665** at one end and are pivotally coupled to a corresponding one of links **446** for rotation about a corresponding axis **498**. Adjustment members **1754** extend from axis **665** at one end are pivotably connected to a corresponding one of links **1755**. Each of links **1755** extends from its corresponding adjustment member **1754** to foot link support **1756**. Foot link supports **1756** a corresponding one of foot links **228**. In the example illustrated, each of foot link support **1756** comprises a roller rotationally supported by link **1755** and having a circumferential groove which receives an underside of a corresponding foot link **228** so as to roll along an underside of the corresponding foot link **228**.

Levers **1776** extend from axis **665** at one end and are pivotally connected to a corresponding one of coupling links **1774** at the other end. Worm drive **1770** comprises a helically threaded shaft rotatably supported by frame **224** for being selectively rotated by adjuster **1758**. The helical threads of worm drive **1770** engage corresponding helical threads of worm drive trolleys **1772**. Links **1774** comprise rods or bars pivotably coupled to a corresponding one of trolleys **1772** and a second end pivotably secured to one of levers **1776** through a universal joint. Levers **1776** extend from supports **1164** and serve as a lever arm for pivoting supports **1164** as well as adjustment member **1754** about their respective axes **665** to reposition the respective pivot axes **498** relative to axis **476** (to adjust the stride length of footpads **230**) and to reposition supports **1756** relative to the

forward pivot axis 1771 joining each foot link 228 to its respective leg 1726 (to adjust the step height of footpads 230).

Actuator 1758 comprises a motor operably coupled to worm drive 1770 and selectively rotates worm drive 1770. In the example illustrated, actuator 1758 comprises a motor operably coupled to worm drive 1775 by a gear train arrangement 1773. In other implementations, actuator 1758 comprises a motor operably coupled to worm drive 1775 by a chain and sprocket arrangement, a toothed belt and pinion gear arrangement or a belt and pulley.

In operation, in response to signals generated by controller 292 (shown in FIG. 4) as a result of either receiving a command or selection through input 290 or being directed by an exercise program stored in a non-transitory memory associated with controller 292, controller 292 generates control signals causing actuator 1758 to rotate worm drive 1770 so as to move links 1774 inward or outward along axis 476. FIG. 35 illustrates trolleys 1772 and the ends of links 1774 at outermost positions along axis 476 while FIG. 36 illustrates trolleys 1772 and the ends of links 1774 at innermost adjacent positions along axis 476. As shown by FIGS. 33 and 34, inward or outward movement of trolleys 1772 causes links 1774 to interact upon levers 1776 so as to pivot supports 1164 about axes 665 so as to reposition the pivot axes 498 of stride length adjusting links 446 to adjust the distance d so as to adjust stride length of the continuous elliptical path of footpads 230. At the same time, such pivoting a rotation of supports 1164 about the respective axes 665 also moves foot link supports 1756 relative to the respective axis 1771 to concurrently adjust the distance e so as to adjust the step height of the continuous elliptical path of footpads 230.

FIG. 37 illustrates an exercise apparatus 1820, another example implementation of exercise apparatus 20. As with exercise apparatus 420, exercise apparatus 1820 provides concurrent or synchronized adjustment of both step height and stride length. Exercise apparatus 1820 is similar to exercise apparatus 1720 except that exercise apparatus 1820 replaces worm drive 1770, trolleys 1772, links 1774 and the single actuator 1758 with a pair of actuators 1858. Those remaining components of exercise apparatus 1820 which correspond to consummate exercise apparatus 1720 are numbered similarly. Some components identified by the same reference numerals may have slightly different configurations, but perform similarly.

Exercise apparatus 1820 comprises frame 424 (shown in FIG. 31), legs 226, foot links 228, footpads 230, left crank 1734L and right crank 1734R (collectively referred to as cranks 1734 and shown in FIG. 35), resistance system 436 (shown in FIG. 31), stride length adjusting links 446, supports 1164, adjustment member 1754, links 1755L, 1755R (collectively referred to as links 1755), foot link support 1756 and adjusters 1858L, 1858R (collectively referred to as adjusters 1858). Exercise apparatus 1820 function similarly to exercise apparatus 1720 except that exercise apparatus 1820 utilizes adjusters 1858 in lieu of actuator 1758, worm drive 1770, trolleys 1772 and the links 1774. Each adjuster 1858 comprise a motor to selectively rotate a threaded nut or other member in meshing engagement with threaded shaft 1870 to selectively extend or retract shaft 1870. Through such selective extension and retraction of shaft 1870, each actuator 1858 pivots supports 1164 about their respective axes 665 to adjust the distance d separating axes 498 and axis 476 so as to adjust the stride length of the continuous elliptical path of footpads 230. At the same time, through such selective extension and retrac-

tion of shaft 1870, each actuator 1858 pivots its associated adjuster member 1754 about axis 665 to reposition its respective support 1756 relative to pivot axis 1771 (the joint or axis joining respective foot link 228 to the respective leg 226) so as to adjust the distance e and thereby adjust the step height of the continuous elliptical path taken by footpads 230.

FIG. 38 is a schematic diagram illustrating exercise apparatus 1920, another example implementation of exercise apparatus 20. For ease of illustration, the frame 224 is omitted and only the left side of exercise apparatus 1920 is illustrated. The right side of exercise apparatus 1920 is substantially similar to the left side illustrated. Exercise apparatus 1920 is identical to exercise apparatus 1720 except that exercise apparatus 1920 utilizes adjusters 1958 in lieu of actuator 1758, worm drive 1770, trolleys 1772 and the links 1774. Each adjuster 1958 comprise a motor to selectively rotate a threaded nut or other member in meshing engagement with threaded shaft 1970 to selectively extend or retract shaft 1970. Through such selective extension and retraction of shaft 1970, each actuator 1858 pivots supports 1164 about their respective axes 665 to adjust the distance d separating axes 498 and axis 476 so as to adjust the stride length of the continuous elliptical path of footpads 230. At the same time, through such selective extension or retraction of shaft 1970, each actuator 1958 pivots its associated adjuster member 1754 about axis 665 to reposition its respective support 1754 relative to pivot axis 1771 (the joint or axis joining foot link 228 to the respective leg 226) so as to adjust the step height of the continuous elliptical path taken by footpads 230.

FIGS. 39-41 illustrate exercise apparatus 2020, another example implementation of exercise apparatus 20. Exercise apparatus 2020 incorporates features and/or functions of exercise apparatus 420 and exercise apparatus 1520. As with exercise apparatus 420 and 1520, exercise apparatus 2020 provides concurrent or synchronized adjustment of both step height and stride length. In one implementation, the paths provided by exercise apparatus 2020 for the footpads are parallel. In another implementation, the paths provided by exercise apparatus 2020 for the footpads converge. In one implementation, the paths for the footpads provided by exercise apparatus 2020 vertically overlap one another at certain points along such paths.

Exercise apparatus 2020 comprises frame 424 (partially shown in broken lines), arms 2026L, 2026R (collectively referred to as arms 2026), foot links 428, footpads 230, left crank 434L and right crank 434R (collectively referred to as cranks 434), resistance system 436, flexible member guides 438L, 438R (collectively referred to as flexible member guides 438), stride height adjusting flexible members 2044L, 2044R (collectively referred to as flexible members 2044), stride length adjusting links 446, pivot wings 2063L, 2063R (collectively referred to as pivot wings 2063), pivots supports 2064L, 2064R (collectively referred to as pivots supports 2064), biases 2066, adjustment member 454, adjuster 458, synchronization coupler 468, spool 470, and support adjustment flexible members 472. Each of such components is described above with respect to other exercise apparatuses but for arms 2026, stride length adjusting flexible elements 472, pivot wings 2063 and pivots supports 2064.

Arms 2026 comprise elongated members pivotably supported by frame 424 for rotation about axis 476. Each of arms 2026 has a first end portion pivotally connected to an associated foot link 428 and a second end portion that supports an extension 2100 which is connected to a corre-



sponding stride height adjusting flexible member **2044**. Stride height adjusting flexible members **2044** are similar to flexible members **444** except that flexible members **2044** extend from their respective extensions **502** of adjustment member **454** and about their respective crank guides **466**, flexible element guides **438** to the end extensions **2100** of arms **2026**.

Pivot wings **2063** comprise angle members pivotably coupled to the frame uprights **475** for pivotal rotation about axis **476**. In the example illustrated, pivot wings **2063** pivot independently of arms **2026**, though the axes may be col-  
linear. Each of pivot wings **2063** has a first portion pivotally secured to a corresponding one of links **429** and a second portion pivotably coupled to a corresponding one of pivoting supports **2064**. Pivoting supports **2064** each have a first portion pivotably connected to a corresponding one of pivot wings **2063** and a second portion pivotably connected to a corresponding one of links **446** which are in turn pivotally connected to a corresponding one of cranks **434**. Biases **2066** comprise compression springs captured between their corresponding pivot wings **2063** and a corresponding pivot supports **2064**. Biases **2066** resiliently bias pivot supports **2064** in a forward direction away from axis **476**.

In the example illustrated, the pivot axes **476** of arms **2026** are each tangent to a circumference of spool **470** (similar to the arrangement shown in FIG. **30** between pivot axis **476** and guides **1365**). Such a configuration reduces or minimizes the extent or range to which supports **2064** move or pivot as footpads **230** traverse their respective paths. In other implementations, the pivot axis **476** of arms **2026** is offset (non-tangent) with respect to the circumference of spool **470**.

In one implementation, each of links **429** is releasably connectable to the associated link **428** at one of plurality of available vertically spaced mounting locations. For example, in one implementation, each foot link **428** comprises a forwardly extending plate or ear having column of vertically spaced apertures by which the end portion of link **429** may be pinned or otherwise mounted. Selectively repositioning the end of link **429** in one of the various vertically spaced attachment or mounting points on the associated foot links **428** allows a person to adjust the range of stride length such that the minimum or maximum of the stride length would be uniformly larger or smaller.

Overall, extension **502L**, flexible member **2044L** and crank guide **466L** form a left stride height mechanism, wherein the stride height of the elliptical path taken by foot pad **430L** is controlled by the positioning of extension **502L** which controls the degree to which flexible member **2044L** wraps about crank guide **466L**. Extension **502R**, flexible member **2044R** and crank guide **466R** form a right stride height mechanism, wherein the stride height of the elliptical path taken by foot pad **430R** is controlled by the positioning of extension **502R** which controls the degree to which flexible member **2044R** wraps about crank guide **466R**.

Crank arms **434**, stride length adjusting links **446**, pivot wings **2063**, pivot supports **2064** and biases **2066** form a stride length mechanism, wherein the stride length of the elliptical path taken each of footpads **230** is controlled by the positioning of the pivot axis **498** of each of links **446** relative to axis **476** of arms **2026**.

In operation, in response to signals generated by controller **292** as a result of either receiving a command or selection through input **290** or being directed by an exercise program stored in a non-transitory memory associated with controller **292**, controller **292** generates control signals causing motor **510** to rotate screw **512** to rotate adjustment member **454**

about axis **504** which adjusts the positioning of the end-points of flexible members **2044**. This repositioning of the endpoints of flexible members **2044** changes the degree to which flexible members **2044** wrap about crank guides **466** and adjusts the step height of the elliptical path being taken by footpads **230**.

Rotation of adjustment member **454** by adjuster **458** causes end portions of coupler **468** to wind or unwind relative to cam **503** and to rotate spool **470**. Rotation of spool **470** winds or unwinds flexible members **472** so as to either pivot supports **2064** about pivot axis **497** rearwardly against the bias of biases **2066** or to allow the bias of biases **2066** to pivot supports **2064** and pivot axis **498** forwardly. Movement of pivot axis **498** relative to the rotational axis **476** of arms **2026** adjusts the stride length of the elliptical path being taken by footpads **230**.

Although the mechanical coupling of the movement for rotation of adjustment member **454** and the movement of the pivot axis of stride length adjusting links **446** is illustrated as being carried out by coupler **468** in the form of a flexible member, spool **470** and flexible members **472** which pivot supports **1164** against the bias, in other implementations, coupler **468** may comprise a gear train, mechanical link, pivot connections or other force transmitting members.

FIGS. **42-46** illustrate exercise apparatus **2120**, another example implementation of exercise apparatus **20**. Exercise apparatus **2120** incorporates features and/or functions of exercise apparatus **420** and exercise apparatus **1520**. As with exercise apparatus **420** and **1520**, exercise apparatus **2120** provides concurrent or synchronized adjustment of both step height and stride length. In one implementation, the paths provided by exercise apparatus **2120** for the footpads are parallel. In another implementation, the paths provided by exercise apparatus **2120** for the footpads converge. In one implementation, the paths for the footpads provided by exercise apparatus **2120** vertically overlap one another at certain points along such paths.

Exercise apparatus **2120** comprises frame **424** (partially shown in broken lines), arms **2126L**, **2126R** (collectively referred to as arms **2126**), foot links **428**, footpads **230**, left crank **434L** and right crank **434R** (collectively referred to as cranks **434**) supporting left crank guide **466R** and right crank guide **466L** (collectively referred to as crank guides **466**), respectively, resistance system **436**, flexible member guides **438L**, **438R** (collectively referred to as flexible member guides **438**), stride height adjusting flexible members **2044L**, **2044R** (collectively referred to as flexible members **2044**), stride length adjusting links **446**, bell cranks **2163L**, **2163R** (collectively referred to as bell cranks **2163**), swing arms **2164L**, **2164R** (collectively referred to as swing arms **2164**), adjustment member **454**, adjuster **458** and planetary cranks **2167L**, **2167R** (collectively referred to as planetary cranks **2167**). Each of such components is described above with respect to other exercise apparatuses but for arms **2126**, bell cranks **2163**, swing arms **2164**, planetary cranks **2167L**, **2167R** (collectively referred to as planetary cranks **2167**) and synchronization couplers **2168L**, **2168R** (collectively referred to as synchronization couplers **2168**).

Arms **2126** comprise elongated members pivotably supported by frame **424** for rotation about axis **476**. Each of arms **2126** has a first end portion pivotally connected to an associated foot link **428** and a second end portion which is connected to a corresponding stride height adjusting flexible member **2044**. Stride height adjusting flexible members **2044** are similar to flexible members **444** except that flexible members **2044** extend from their respective extensions **502**



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of adjustment member 454 and about their respective crank guides 466, about flexible element guides 438 to arms 2126.

Bell cranks 2163 comprise crank members that change motion through an angle. Bell cranks 2163 are pivotably coupled to the frame uprights 475 for pivotal rotation about axis 476. Bell cranks 2163 pivot independently of arms 2126, wherein the axes may be collinear or proximate each other on separate non-collinear axes. Each of bell cranks 2163 has a first portion pivotally secured to a corresponding one of links 429 and a second portion pivotably coupled to a first end portion of corresponding one of stride length adjusting links 446.

Swing arms 2164 are connected to and extend vertically upwards from bell cranks 2163. Each of swing arms 2164 has a hand grip portion 2165 having an outer compressible or soft surface to facilitate gripping. In some implementations, the outer compressible surface may be omitted.

Planetary cranks 2167L, 2167R comprise cranks that are carried by cranks 434 and that are rotatable or pivotable about axes 2169L, 2169R (collectively referred to as axes 2169) which are each offset from axis 2171 about which cranks 434 pivot or rotate. Each of planetary cranks 2167 has an end portion, eccentrically located with respect to the corresponding axis 2169 and pivotally connected to a second end portion of corresponding one of links 446. Pivoting or rotation of planetary cranks 2167 adjusts the clocking or angular position of the second end portion of the corresponding link 446 relative to axis 2171 and relative to the corresponding axis 2169. Because planetary cranks 2167 are rotatably supported by their corresponding crank 434, planetary cranks 2167 may be angularly repositioned to reposition and adjust the angular orientation of second end portion of the corresponding link 446. As a result, the stride length of the corresponding foot link 428 may be adjusted by repositioning and adjusting the orientation of the second end portion link 446 relative to axis 2169 and axis 2171.

Synchronization couplers 2168 connect or synchronize the motion and position of adjustment member 454 and the motion and position of planetary cranks 2167 such that adjustment of the step height through the rotation of adjustment member 454 concurrently and synchronously adjusts the stride length and step height by changing the clocking or orientation of the second end portion of links 446 relative to axes 2169. FIG. 46 illustrates synchronization coupler 2168L which is substantially identical to synchronization coupler 2168R. For purposes of illustration, portions of synchronization coupler 2168L are transparently illustrated in FIG. 46. In the example illustrated, each of planetary cranks 2167 comprises a pivot shaft 2174 that extends through bearing 2176 within crank 434L. Bearing 2178 outside crank 434L supports guide 466L. Bearings 2176 and 2178 facilitate independent rotation of shaft 2174 and crank guide 466L about axis 2169L, wherein crank 434L supports both shaft 2174 and crank guide 466.

Synchronization coupler 2168L comprises a first member 2180, a second member 2182 and a torque coupler 2184. Member 2180 comprises a member that is integrally formed as part of adjustment member 454 or that is otherwise secured or joined to adjustment member 454 so as to be carried by and rotate with member 454 about axis 2171 (shown in FIG. 45). Member 2182 comprises a member that is integrally formed as part of shaft 2174 or that is otherwise secured or joined to shaft 2174 so as to rotate with shaft 2174 about axis 2169L. Torque coupler 2184 comprises one or more members interconnecting members 2180 and 2182 such that torque and rotational movement of member 2180 as a result of adjustment member 454 pivoting about axis

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2171 is transmitted to member 2182 which results in the associated planetary crank 2167L being rotated about axis 2169 to adjust the clocking of crank 2167, and reposition the end portion 2185 of link 446L to adjust the stride length of foot link 428L.

In the example illustrated, members 2180 and 2182 comprise gears while torque coupler 2184 (schematically shown) comprises a toothed belt. In other implementations, members 2180 and 2182 comprise gears while torque coupler 2184 comprises one or more intermediate idler gears, forming a gear train. In yet another implementation, members 2180 and 2182 may comprise sprockets, wherein torque coupler 2184 comprises a chain. In still other implementations, members 2180 and 2182 may comprise pulleys, wherein torque coupler 2184 comprises a cable, belt or other continuous loop to flexible member wrapped about members 2180 and 2182.

In still other implementations, adjustment member 454 may be operably coupled to planetary crank 2167L in other fashions such that rotation or movement of adjustment member 454 also results in rotation or movement of planetary crank 2167L. In yet other implementations, in lieu of the motion of adjustment member 454 being used to drive the motion of planetary gear 467 about axis 2169L, members 2180, 2182 and 2184 may be omitted wherein each of planetary cranks 2167 is rotatably driven about its corresponding axis 2169 by an independent rotary actuator. For example, in one implementation, a servo motor or stepper motor is provided within bushing 2176, within bushing 2178 or in the location of the presently illustrated member 2182, connected to shaft 2174. In such an implementation, the rotary actuator is operated under the control of a central controller which automatically rotates the corresponding planetary crank 2167 in response to rotation of adjustment member 454 about axis 2172. In one implementation, the rotation of planetary cranks 2167 about the different axes 2169 is concurrent with and proportional to the rotation of adjustment member 454 about axis 2172. In such an implementation, the rotary actuator, under the control of the controller, serves as the synchronization couplers.

During use of exercise apparatus 2120, movement of crank 434L around axis 2171 causes planetary crank 2167L, shaft 2174 and member 2182 to rotate about axis 2169 as axis 2169 revolves around axis 2171. Members 2180 and 2182 have a gear/sprocket ratio of 2 to 1. In the example illustrated, member 2180 has twice as many teeth as member 2182. As a result, planetary crank 2167 rotates twice about axis 2169L for each rotation of crank 434L about axis 2171 providing a constant elliptical path at the pivot axis at the end 2185 of link 446. When adjustment member 454 is moved by adjuster 458 to select a stride path, member 2180 (normally stationary through the stride) changes the clocking of the planetary crank 2167L and so changes the orientation of the ellipse path at end 2185 of link 446L. The orientation of long or short sides of the ellipse path relative to the primarily linear motion of links 446 adjusts the horizontal stride length.

Overall, extension 502L, flexible member 2044L and crank guide 466L form a left stride height mechanism, wherein the stride height of the elliptical path taken by foot pad 430L is controlled by the positioning of extension 502L which controls the degree to which flexible member 2044L wraps about crank guide 466L. Extension 502R, flexible member 2044R and crank guide 466R form a right stride height mechanism, wherein the stride height of the elliptical path taken by foot pad 430R is controlled by the positioning

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of extension **502R** which controls the degree to which flexible member **2044R** wraps about crank guide **466R**.

In operation, in response to signals generated by controller **292** as a result of either receiving a command or selection through input **290** or being directed by an exercise program stored in a non-transitory memory associated with controller **292**, controller **292** generates control signals causing motor **510** to rotate screw **512** to rotate adjustment member **454** about axis **504** which adjusts the positioning of the endpoints of flexible members **2044**. This repositioning of the endpoints of flexible members **2044** changes the degree to which flexible members **2044** wrap about crank guides **466** and adjusts the step height of the elliptical path being taken by footpads **230**.

Planetary crank **2167L** and synchronization coupler **2168L** form a left side stride length mechanism while planetary crank **2167R** and synchronization coupler **2168R** form a right side stride length mechanism. Rotation of adjustment member **454** by adjuster **458** causes corresponding rotation of members **2180** to be transmitted to members **2182** by torque couplers **2184**. The resulting rotation of members **2182** rotates planetary cranks **2167** which changes the clocking positions of planetary cranks **2167** to adjust the orientation of the ellipse path at ends **2185** of links **446**, wherein the resulting orientation of long or short sides of the ellipse path relative to the primary linear motion of links **446** adjusts the horizontal stride length.

Although the present disclosure has been described with reference to example embodiments, workers skilled in the art will recognize that changes may be made in form and detail without departing from the spirit and scope of the claimed subject matter. For example, although different example embodiments may have been described as including one or more features providing one or more benefits, it is contemplated that the described features may be interchanged with one another or alternatively be combined with one another in the described example embodiments or in other alternative embodiments. Because the technology of the present disclosure is relatively complex, not all changes in the technology are foreseeable. The present disclosure described with reference to the example embodiments and set forth in the following claims is manifestly intended to be as broad as possible. For example, unless specifically otherwise noted, the claims reciting a single particular member also encompass a plurality of such particular members.

What is claimed is:

1. An apparatus comprising:

- a frame;
- a left foot link supporting a left foot pad;
- a right foot link supporting a right foot pad, the right foot pad being linked to the left foot pad so as to synchronously move out of phase with the left foot pad through an elliptical path having an adjustable step height and an adjustable stride length; and
- a left flexible member having a first end operably coupled to the left foot link; and
- a right flexible member having a first end operably coupled to the right foot link;
- a left crank guide;
- a right crank guide;
- a left crank eccentrically supporting the left crank guide relative to a first rotational axis about which the left crank is rotatable;
- a right crank eccentrically supporting the right crank guide relative to the first rotational axis about which the right crank guide is rotatable;

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an adjustment synchronizer operably coupled to the left foot link and the right foot link to synchronously adjust both the step height and the stride length of the elliptical path, wherein the adjustment synchronizer comprises:

- a movable adjustment member connected to a second end of the left flexible member and a second end of the right flexible member, wherein movement of the adjustable member adjusts an extent to which the left flexible member wraps about the left crank guide and the right flexible member wraps about the right crank guide to adjust a step height synchronously with adjustment of a stride length of paths through which the left foot link and the right foot link move.

2. The apparatus of claim 1, further comprising:

- a left side arm having a first end portion pivotally connected to the left foot link and a second end portion pivotally supported by the frame about a second axis;
- a right side arm having a first end portion pivotally connected to the right foot link and a second end portion pivotally supported by the frame about the second axis;
- a left bell crank pivotally supported by the left side arm;
- a right bell crank pivotally supported by the right side arm;
- a left support link having a first portion pivotally coupled to the left foot link and a second portion pivotally coupled to the left bell crank;
- a right support link having a first portion pivotally coupled to the right foot link and a second portion pivotally coupled to a first portion of the right bell crank;
- a left linkage having a first end portion pivotally coupled to a second portion of the left bell crank; and
- a right linkage having a first end portion pivotally coupled to a second portion of the right bell crank; and
- a synchronization coupler operably coupling the adjustment member to the a second portion of the left linkage and a second portion of the right linkage such that movement of the adjustment member adjusts a relative angular orientation of the second portion of the left linkage and the second portion of the right linkage with respect to the first axis to adjust the stride length.

3. The apparatus of claim 2 further comprising:

- a left swing arm connected to the left bell crank, the left swing arm comprising a left hand grip portion; and
- a right swing arm connected to the right bell crank, the right swingarm comprising a right hand grip portion.

4. The apparatus of claim 1, wherein the adjustment member pivots about the first axis.

5. The apparatus of claim 4 further comprising:

- a left side arm having a first end portion pivotally connected to the left foot link and a second end portion pivotally supported by the frame about a second axis;
- a right side arm having a first end portion pivotally connected to the right foot link and a second end portion pivotally supported by the frame about the second axis;
- a left bell crank pivotally supported by the left side arm;
- a right bell crank pivotally supported by the right side arm;
- a left support link having a first portion pivotally coupled to the left foot link and a second portion pivotally coupled to the left bell crank;

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- a right support link having a first portion pivotally coupled to the right foot link and a second portion pivotally coupled to a first portion of the right bell crank;
- a left linkage having a first end portion pivotally coupled to a second portion of the left bell crank; and
- a right linkage having a first end portion pivotally coupled to a second portion of the right bell crank, wherein the adjustment member is operably coupled to the second portion of the left linkage and a second portion of the right linkage such that pivoting of the adjustment member about the first axis adjusts a relative angular orientation of the second portion of the left linkage and the second portion of the right linkage with respect to the first axis to adjust the stride length.
- 6. The apparatus of claim 5 further comprising:
  - a left planetary crank carried by the left crank and pivotally coupled to the second portion of the left linkage for pivotal movement about a second axis;
  - a right planetary crank carried by the right crank and pivotally coupled to the second portion of the right linkage for pivotal movement about a third axis, wherein the adjustment member is operably coupled to the left planetary crank and the right planetary crank such that pivoting of the adjustment member pivots the left planetary crank and the right planetary crank about the second axis and the third axis, respectively.
- 7. The apparatus of claim 6 further comprising:
  - a first member connected to the adjustment member to pivot with the adjustment member about the first axis;
  - a second member connected to the left planetary crank to rotate about the second axis with the left planetary crank; and
  - a torque transmitter operably coupling the first member to the second member such that rotation of the first member about the first axis rotates the second member about the second axis.
- 8. The apparatus of claim 7, the torque transmitter comprises a flexible member.
- 9. The apparatus of claim 7, wherein the first member and the second member each comprises a gear.
- 10. The apparatus of claim 9, the torque transmitter comprises a flexible member.
- 11. The apparatus of claim 10, wherein the flexible member comprises a toothed belt.
- 12. The apparatus of claim 1 further comprising:
  - a left side arm having a first end portion pivotally supported by the frame about a second axis;
  - a right side arm having a first end portion pivotally supported by the frame about the second axis, wherein the left foot link and the right foot link are pivotally suspended from a second end portions of the left side arm and the right side arm, respectively.
- 13. The apparatus of claim 1 further comprising:
  - a second left crank guide eccentrically supported by the left crank;
  - a second right crank guide eccentrically supported by the right crank;
  - a left side leg having a first end portion pivotally connected to the left foot link and a second end portion pivotally supported by the frame about a first axis;
  - a right side leg having a first end portion pivotally connected to the right foot link and a second end portion pivotally supported by the frame about the first axis;
  - a left pulley connected to the left side leg so as to pivot with the left side leg;

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- a right pulley connected to the right side leg so as to pivot with the right side leg; and
- a stride length adjusting flexible member having ends which are movable by the adjustment member to adjust an extent to which the stride length adjusting flexible member wraps about the second left crank guide and the second right crank guide to adjust a stride length of the elliptical path.
- 14. The apparatus of claim 13, wherein the adjustment member is connected to the left flexible member and the right flexible member on a first side of the first axis and connected to the stride length adjusting flexible member on a second side of the first axis.
- 15. The apparatus of claim 1 further comprising a powered actuator to move the adjustment member.
- 16. An apparatus comprising:
  - a frame;
  - a left foot link supporting a left foot pad;
  - a right foot link supporting a right foot pad, the right foot pad being linked to the left foot pad so as to synchronously move out of phase with the left foot pad through an elliptical path having an adjustable step height and an adjustable stride length; and
  - a left flexible member having a first end operably coupled to the left foot link; and
  - a right flexible member having a first end operably coupled to the right foot link;
  - a left crank guide;
  - a right crank guide;
  - a left crank eccentrically supporting the left crank guide relative to a first axis about which the left crank is rotatable;
  - a right crank eccentrically supporting the right crank guide relative to the first axis about which the right crank guide is rotatable;
  - a movable adjustment member connected to a second end of the left flexible member and a second end of the right flexible member, wherein movement of the adjustable member adjusts an extent to which the left flexible member wraps about the left crank guide and the right flexible member wraps about the right crank guide to adjust a step height of a path of the left foot link in the right foot link;
  - a left side arm having a first end portion pivotally connected to the left foot link and a second end portion pivotally supported by the frame about a second axis;
  - a right side arm having a first end portion pivotally connected to the right foot link and a second end portion pivotally supported by the frame about the second axis;
  - a left bell crank pivotally supported by the left side arm;
  - a right bell crank pivotally supported by the right side arm;
  - a left support link having a first portion pivotally coupled to the left foot link and a second portion pivotally coupled to the left bell crank;
  - a right support link having a first portion pivotally coupled to the right foot link and a second portion pivotally coupled to a first portion of the right bell crank;
  - a left linkage having a first end portion pivotally coupled to a second portion of the left bell crank; and
  - a right linkage having a first end portion pivotally coupled to a second portion of the right bell crank
  - a left planetary crank carried by the left crank and pivotable about a third axis, the left planetary crank being pivotally connected to a second end portion of

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the left linkage, wherein the left planetary crank is retainable at any of a plurality of different angular orientations about the third axis and wherein each of the different angular orientations corresponds to a different stride length for the left foot link;

- a right planetary crank carried by the right crank and pivotable about a fourth axis, the right planetary crank being pivotably connected to a second end portion of the right linkage, wherein the right planetary crank is retainable at any of a plurality of different angular orientations about the fourth axis and wherein each of the different angular orientations corresponds to a different stride length for the right foot link.

17. The apparatus of claim 16 further comprising adjustment synchronizers operably coupled to the left foot link and the right foot link to synchronously adjust both the step height and the stride length of the elliptical path.

18. The apparatus of claim 17, wherein the adjustment synchronizers comprise:

- a first member connected to the adjustment member to pivot with the adjustment member about the first axis;

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a second member connected to the left planetary crank to rotate about the third axis with the left planetary crank; and

a first torque transmitter operably coupling the first member to the second member such that rotation of the first member about the first axis rotates the second member about the third axis;

a third member connected to the adjustment member to pivot with the adjustment member about the first axis;

a fourth member connected to the right planetary crank to rotate about the fourth axis with the right planetary crank; and

a second torque transmitter operably coupling the third member to the fourth member such that rotation of the third member about the first axis rotates the fourth member about the fourth axis.

19. The apparatus of claim 18, wherein each of the first torque transmitter and the second torque transmitter comprises a flexible member.

20. The apparatus of claim 18, wherein each of the first member, the second member, the third member and the fourth member comprises a gear.

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